

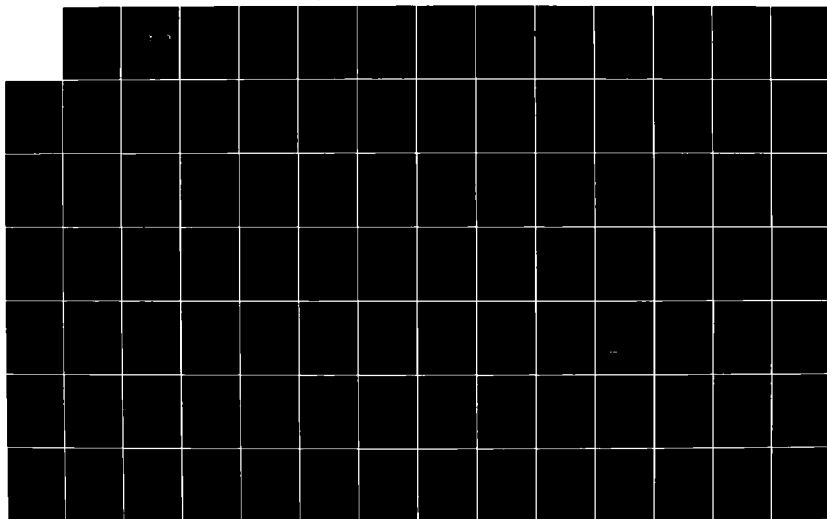
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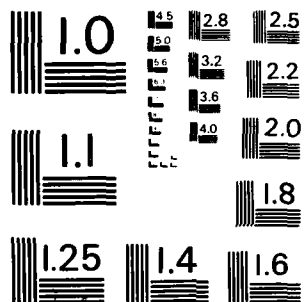
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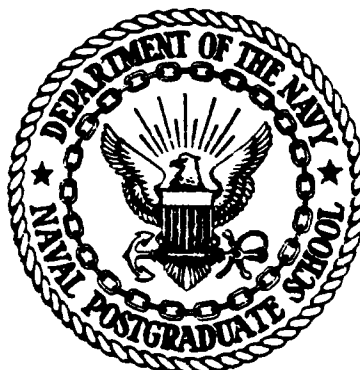




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THESIS

DUAL-PHASE NOZZLE FLOW

by

Thomas C. Nollie, Jr.

October 1982

Thesis Advisor:

J. F. Sladky

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Dual-Phase Nozzle Flow

by

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Lieutenant, United States Navy
B.S., United States Naval Academy, 1975

Submitted in partial fulfillment of the
requirements for the degree of

MASTER OF SCIENCE IN MECHANICAL ENGINEERING

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ABSTRACT

A review of the dual-phase power system was made. This study focused on the multi-component nozzle of this dual-phase system. First, an existing computer code predicting the nozzle performance was updated, and second a series of experimental tests on a variable area, two-dimensional nozzle was performed to verify the computer code.



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I. DUAL-PHASE CYCLE

The dual-phase nozzle is a key element in the dual-phase engine concept. This nozzle will be studied in detail, but first a review of the dual-phase cycle will be carried out from information obtained from References 1 and 2.

Reference 1 describes the dual-phase engine. It is a new concept which operates on a mixture of two fluids or two phases of one fluid. This cycle employs a high-torque/low rpm impulse turbine which eliminates the requirement for speed reduction needed in a conventional steam turbine. This allows for direct drive of a ship's propeller with no gear reduction. This is ideally suited for marine propulsion, reducing weight, noise, and volume.

The dual-phase concept is a modified Rankine cycle with subtle differences of extreme importance. In a normal steam turbine the working fluid enters a turbine through a nozzle where the kinetic energy of the working fluid is converted to a mechanical form. The dual-phase system introduces a second fluid prior to entry into the nozzle. This fluid is of higher vapor pressure than the steam and therefore remains in the liquid state throughout the cycle. Section A will describe this two-component cycle while section B will do the same for a single-component system. The dual-phase

system can be divided into the two groups illustrated in Figure 1. Single-component flow can be further divided into three categories.

A. TWO-COMPONENT

A two-component mixture is one in which the low vapor pressure liquid and a high vapor pressure liquid are of different chemical compounds. Some fluid combinations which have been considered are steam-krytox, steam-caloria, steam-lead, bismuth eutectic, and dow-therminol. A schematic flow diagram and process representation on the T-S diagram are shown in Figures 2 and 3 for the two-phase engine cycle using a "two-component" mixture. The liquid phase is lithium carbonate and vapor phase is steam. To illustrate the overall advantages of the two-phase engine cycle a discussion on the theory of operation will be presented using a two-component mixture and Figures 2 and 3.

The major component of the dual-phase system is the nozzle. A mixer area is located prior to the nozzle inlet. A high vapor pressure liquid is placed in contact with the low vapor pressure liquid in this area. A high pressure vapor liquid mixture is formed. Since the temperature of the liquid is greater than temperature of the water, heat is transferred to the water causing it to vaporize to point 1. Figure 4 illustrates the temperature and state point from the inlet of the mixture area to the nozzle exit. This mixture is

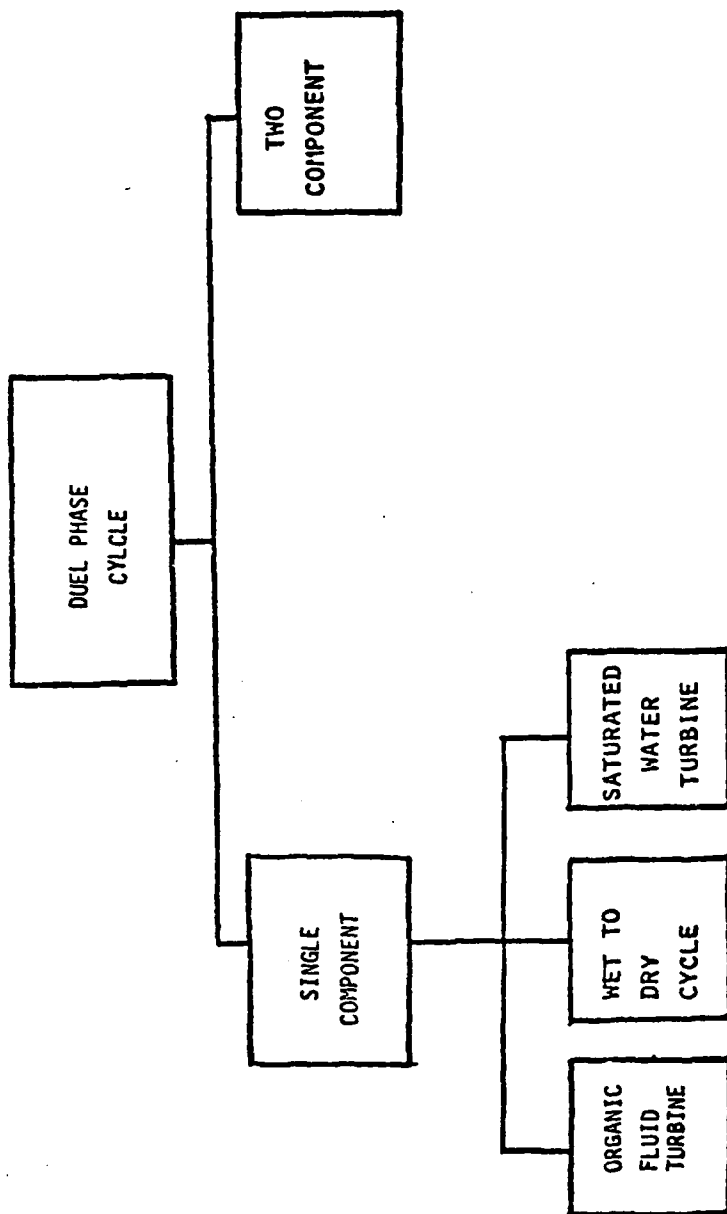


Figure 1. Division of the Dual-Phase Cycle

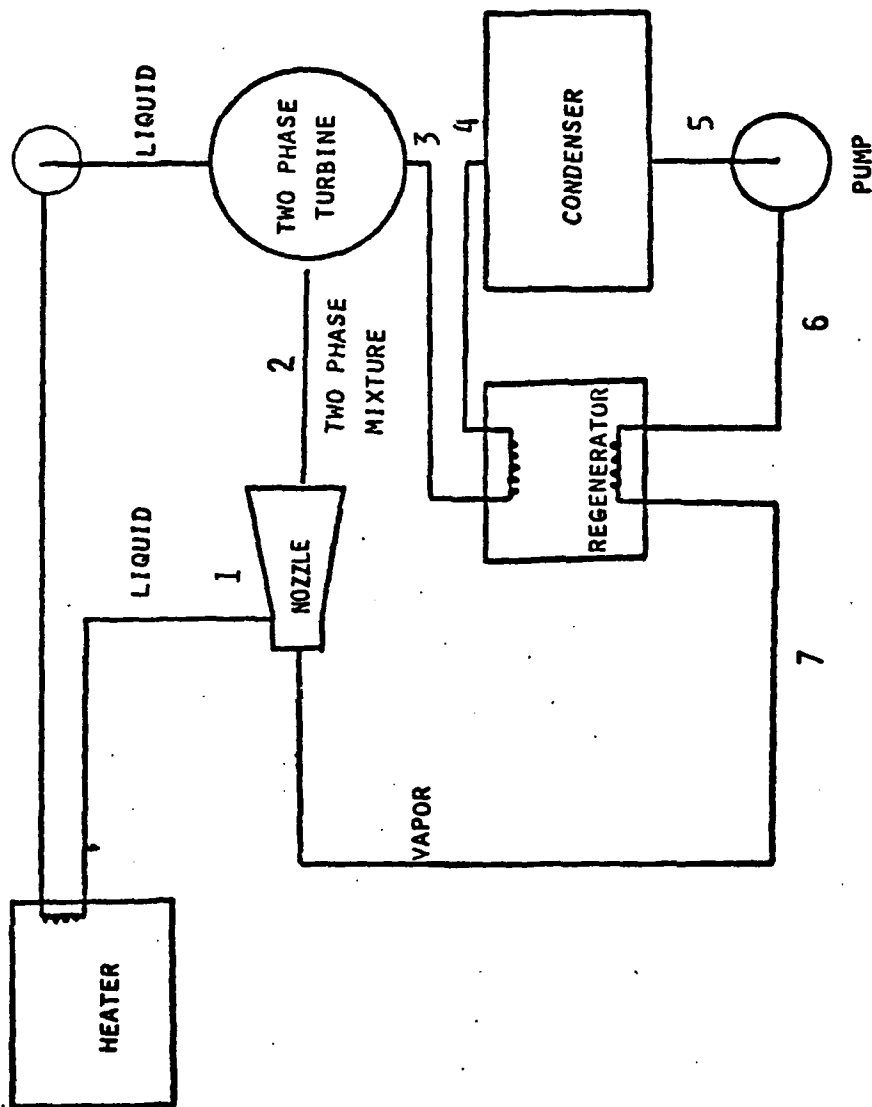


Figure 2. Dual-Phase Two-Component System

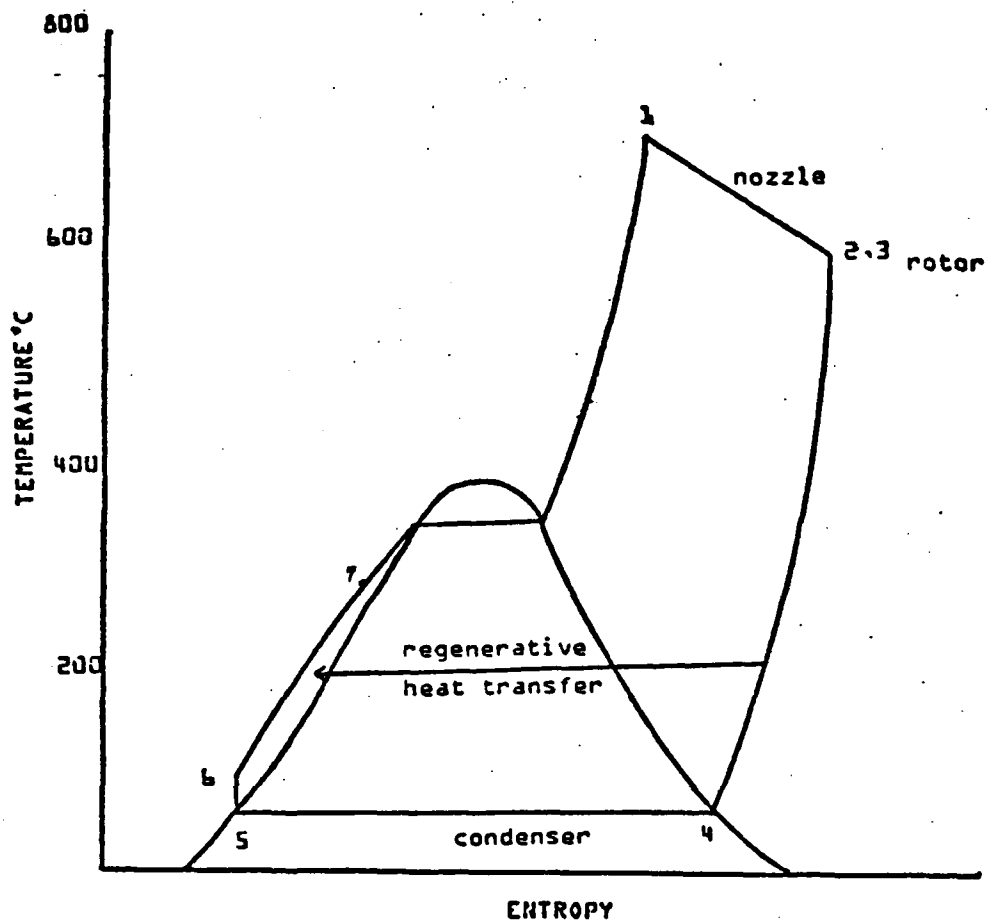


Figure 3. Dual-Phase Two-Component T-S Diagram

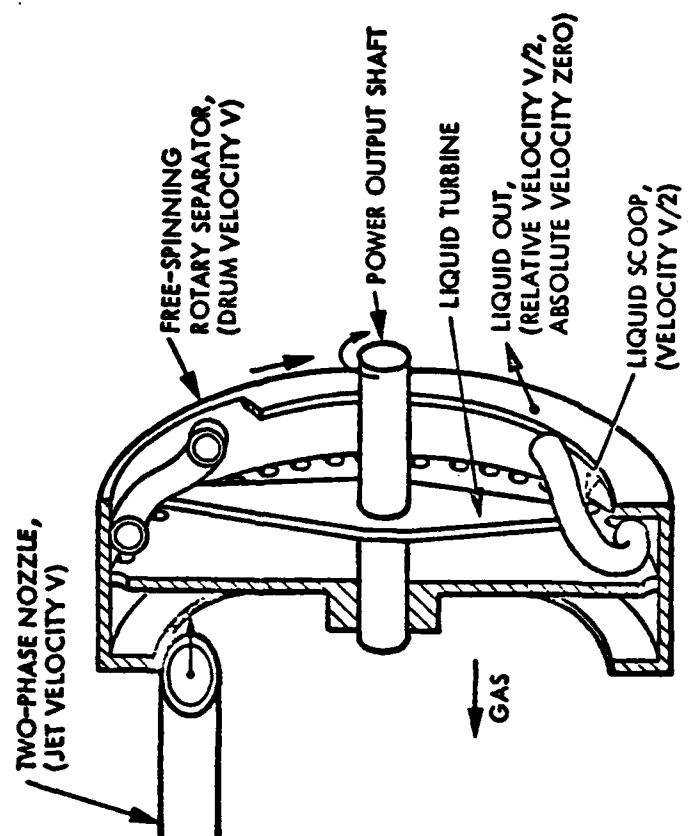
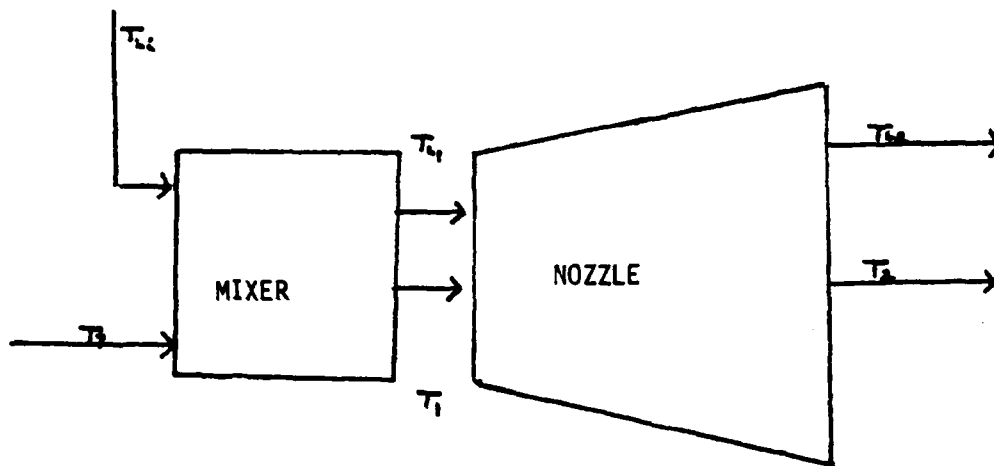


Figure 4. Liquid Impulse Turbine Schematic

expanded to a low pressure at the nozzle exit. The expanding vapor transfers momentum to the liquid droplets, while the droplets transfer heat to the vapor. This tends to approach isothermal expansion through the nozzle, point 2. After leaving the nozzle, the liquid droplets are separated from the vapor by a rotary separator located on the turbine. The rotating drum turns at close to the nozzle's exit velocity. Hence there is very little friction loss between the separated liquid and the drum wall. The centrifugal acceleration produces a very clean separation of vapor and liquid. The liquid traveling with the rim of the rotating drum transfers axially through holes in the separator disc and subsequently enters the liquid turbine proper. The kinetic energy of the liquid is converted to shaft horsepower by the liquid turbine. Figure 5 illustrates the dual-phase liquid impulse turbine assembly. The vapor remains superheated as a consequence of the heat transfer from the liquid. The vapor flows from the separator, point 3, through a regenerator where heat is transferred to the condensate. The vapor is condensed, point 5; pumped to nozzle pressure, point 6; and passed through the regenerator for heating. Heat is added between point 6 and point 1 in two methods. The regenerator adds heat to the condensate by using the steam from the turbine rotor, point 7. The remainder of heat, point 7 to 1, is added by the heated liquid mixed with the condensate in the mixer. The water is vaporized by direct-contact heat transfer.



where

- T_{L_i} = Temp of liquid entering mixer
- T_7 = Temp of water entering mixer
- $T_{L_i} > T_1$
- T_2 = Temp of steam leaving the nozzle
- T_{L_2} = Temp of liquid leaving the nozzle

Figure 5. Temperature & State Point Diagram for the Mixture & Nozzle

In the nozzle most of the thermal energy of the steam is converted to kinetic energy of the liquid droplets. This acceleration of the liquid by the vapor in the two-phase nozzle provides the kinetic energy to drive the liquid impulse turbine. The liquid velocities involved are relatively low as compared to velocity of the vapor. Thus, the output of the impulse hydraulic turbine will be high torque/low rpm. This conversion of the liquid kinetic energy to shaft power at high torque with low rpm appears to have direct application to naval propulsion.

B. ONE-COMPONENT

A one-component system is one in which the working fluid is of the same chemical compound. One of the simplest dual-phase one-component systems is illustrated in Figures 6 and 7. The working fluid is heated to saturation temperature by some type of heat source. Heat sources applicable to this case are geothermal power plants, engine exhaust, industrial waste-heat recovery, and bottoming cycles for steam and gas turbine plants. This working fluid, at saturated liquid conditions, with small amounts of vapor is expanded through a two-phase nozzle. As the expansion process takes place, the liquid partially vaporizes and accelerates the remaining liquid phase in the nozzle. The dual-phase mixture enters the rotary separator and the same process occurs as mentioned in section A. Since the liquid phase is of a much higher

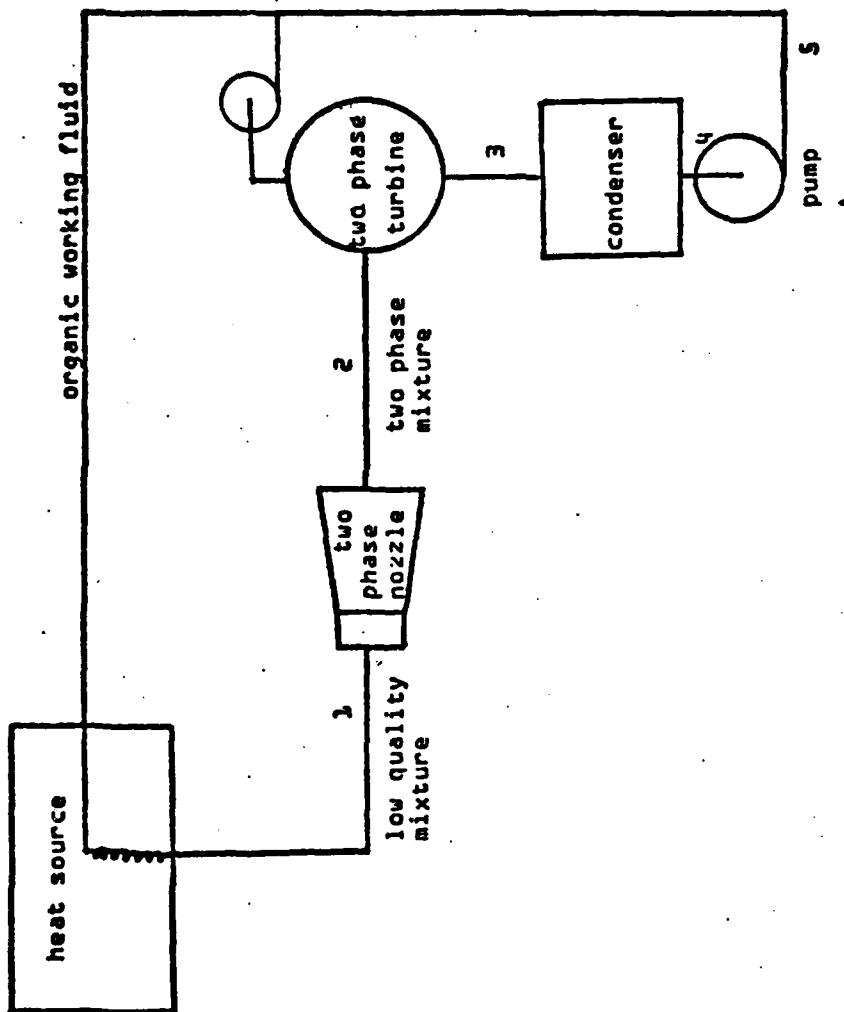


Figure 6. Dual-Phase Single-Component System

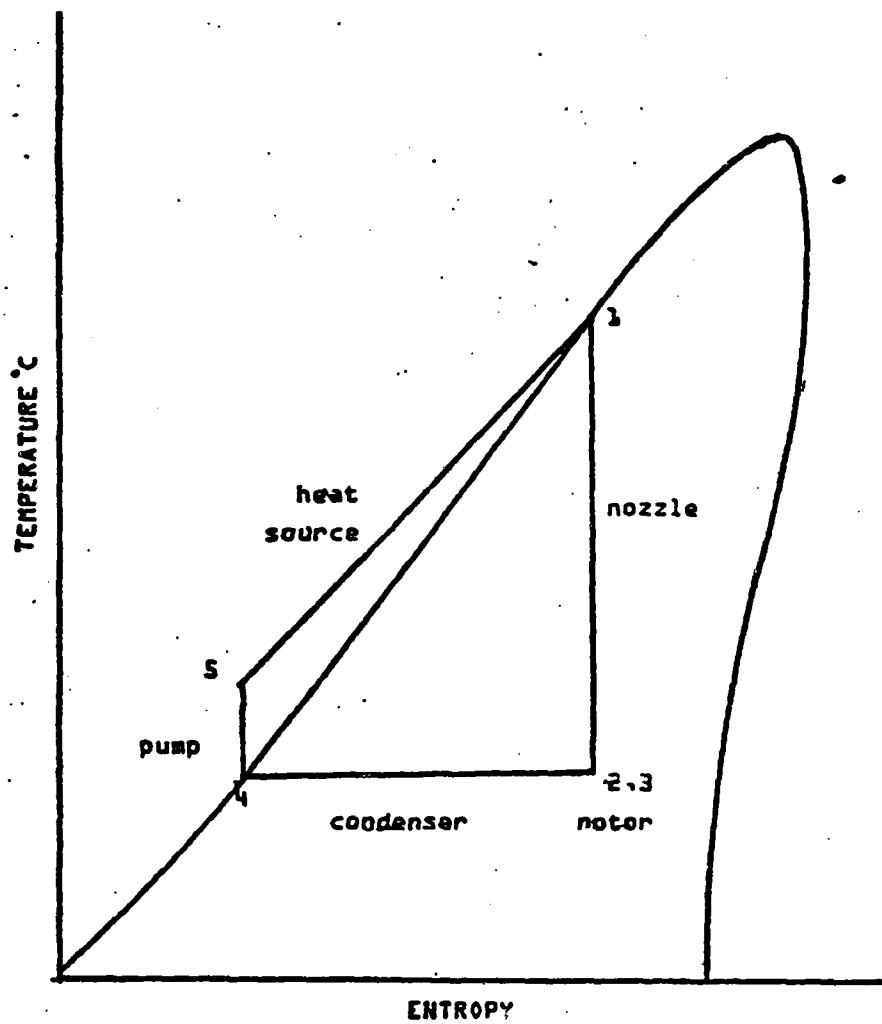


Figure 7. Dual-Phase Single-Component T-S Diagram

density than the vapor, the liquid velocities are relatively low as compared to the velocity of the vapor. Thus the output of the turbine will again have high torque at low rpm. After leaving the turbine the vapor mixture is condensed and the condensate is pumped back to the heat source. The cycle is shown on a T-S diagram in Figure 6. The state points are numbered to correspond with Figure 5. The dual-phase nozzle expansion takes the fluid from a saturated liquid, point 1, to a dual-phase flow, point 2. The flow is decelerated in the rotor; condensed, point 4; and pumped back to nozzle inlet pressure at point 5. The liquid is then reheated by the source fluid to point 1.

Another application of the one-component two-phase cycle is the wet-to-dry cycle. If the initial temperature of the working fluid is sufficiently high and the saturation curve has a positive saturated liquid slope the working fluid can be expanded to dry vapor. Figure 8 is the T-S diagram for a wet-to-dry cycle. The fluid is expanded from saturated liquid at point 1 to saturated vapor at point 2. The vapor drives an impulse rotor and leaves the rotor slightly superheated at point 3. The vapor is condensed to point 4 and pumped back to the nozzle inlet pressure at point 5.

C. ADVANTAGES

The advantage of the dual-phase cycle with respect to marine application is the ability to achieve low shaft speed

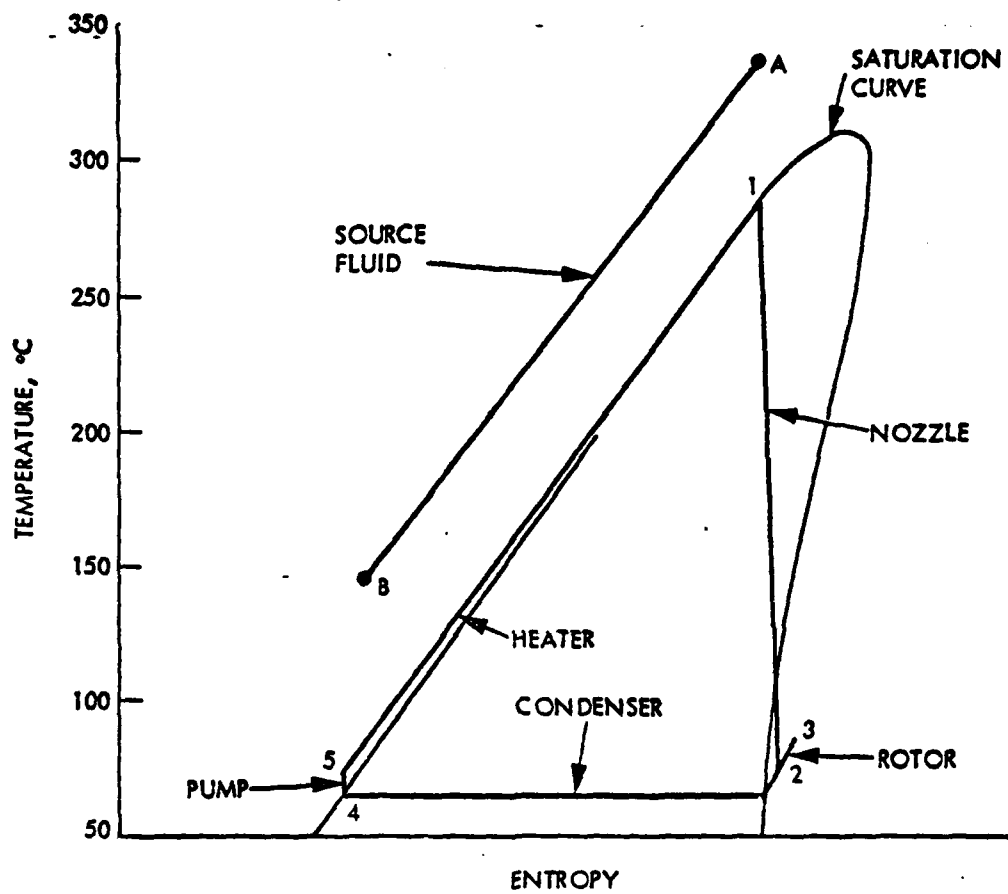


Figure 8. Wet-to-Dry T-S Diagram

in a small turbine engine. For example a steam turbine of 150-kw shaft horsepower using the temperature given in Figure 2 would have a speed of about 60,000 rpm. A comparable two-phase turbine would operate at approximately 10,000 rpm. There is also an efficiency advantage. At temperature corresponding to Figure 2, a steam Rankine cycle would have an efficiency of approximately 28% where-as the two-phase cycle would have an efficiency of 37%. This is assuming equal turbine efficiencies. The two-phase cycle also allows for control of turbine speed because the vapor/liquid mixture ratio can be varied to change the nozzle exit velocity. This is a capability unavailable in a conventional steam turbine.

Both of the dual-phase concepts can be thought of as a form of a regenerated Rankine cycle. The dual-phase cycle by control of liquid/vapor mixture ratio enhances the overall power system controlability. The T-S relationship for a dual-phase two-component engine cycle compared to a Rankine cycle is shown in Figure 9.

Two design studies References 1 and 2, have shown potential advantages in the two-phase engine cycle as compared to the conventional Rankine cycle for marine propulsion. The following advantages were noted:

1. High Efficiency - Full load output power performance gains ranging from 20 to 50 percent was found.
2. Direct Drive - Direct drive at speeds ranging from 90-4500 rpm was found possible with a single-stage turbine.

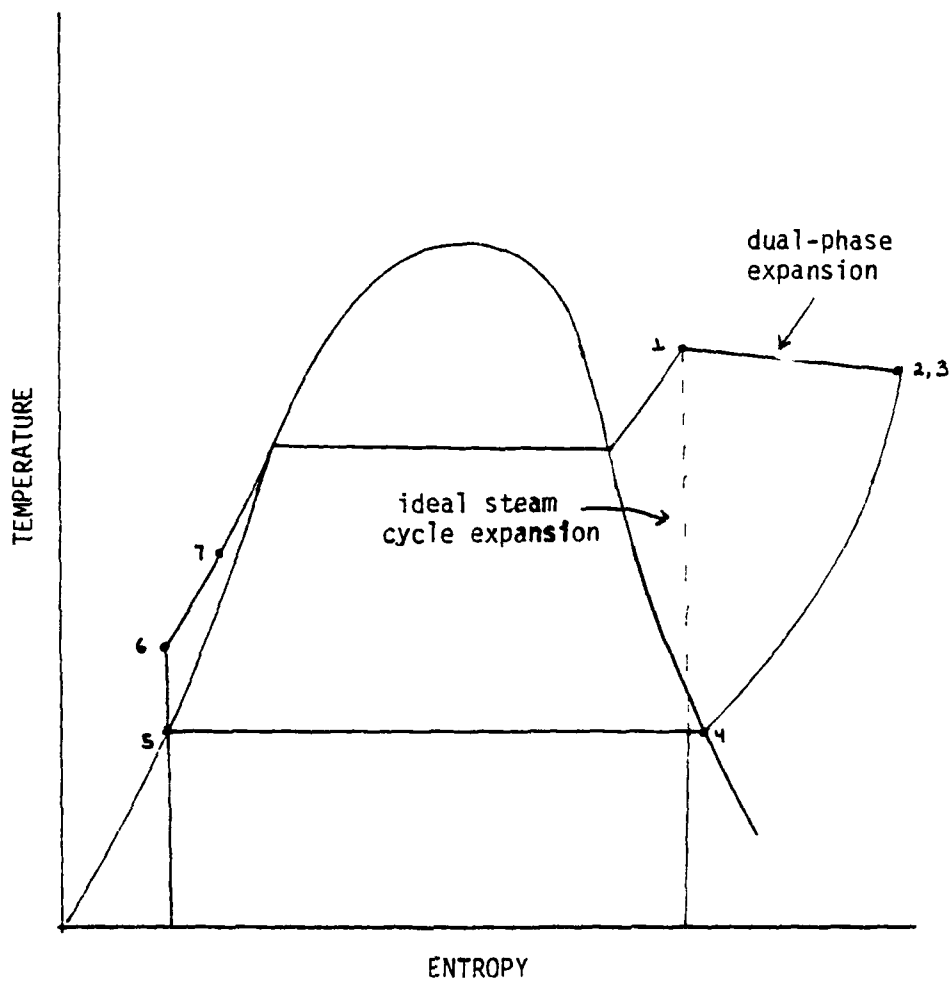


Figure 3. T-S Comparison of Dual-Phase Two-Component Cycle & Rankine Cycle

3. Reduced Volume - Volume reduction of 30 percent were estimated.

4. High Part-Load Efficiency - Variable mass ratio enabled part-load (cruise) efficiency gain of as much as 100 percent.

II. DUAL-PHASE NOZZLE THEORY

The flow phenomenon of a two-phase mixture has been analyzed in Reference 1. It is repeated as follows. The problem is illustrated in Figure 10. A spatially uniform two-component mixture of liquid drops and gas enters a nozzle at high pressure and low velocity and expands to low pressure and high velocity. The objective of the analysis is to determine, for a specified pressure the drop diameter D and the temperatures T_g and T_L , velocities V_g and V_L and flow rates \dot{m}_g and \dot{m}_L of the gas and liquid phases, respectively, at each station in the nozzle given the initial values of D , T_g , T_L , V_g , V_L , the total flow rate, and the properties of the fluids.

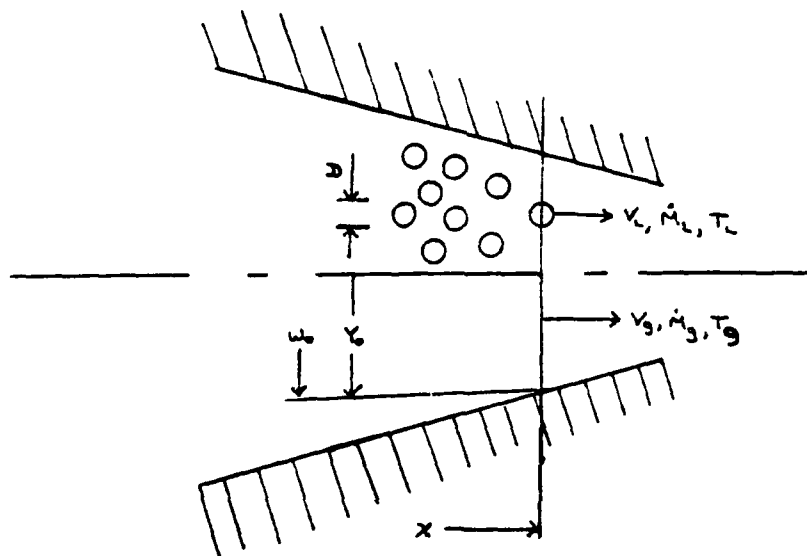


Figure 10. Dual-Phase Nozzle Flow
Geometry & Nomenclature

The five relations employed to compute the five unknowns D , T_g , T_L , V_g , and V_L , are (1) the momentum equation for the mixture, (2) the energy equation for the mixture, (3) the drop drag equation, (4) the drop heat transfer equation, and (5) the drop breakup criterion. Solubility and vapor pressure relations provide the flow rate ratio \dot{m}_g/\dot{m}_L .

A. ASSUMPTIONS

The assumptions employed in the two-component analysis are as follows:

1. The liquid is uniformly dispersed as spherical drops all of the same diameter.
2. The drops break up to limit the Weber number to 6.
3. There are no external forces acting on the two-phase mixture other than pressure and wall shear, and there is no heat transfer to or from the mixture.
4. The flow is one-dimensional.
5. The drops are large enough for the surface curvature to have negligible effect on the vapor pressure of the liquid and for the surface energy to be negligible.
6. The drops are isothermal.
7. The gas mixture obeys the additive-pressure law.
8. The partial pressure of the predominantly liquid component in the liquid is given by Raoult's Law.
9. The concentration of the predominantly gaseous component in the liquid is given by Henry's Law.

10. The volume of the liquid solution is equal to the sum of the volumes of the pure liquids.

Assumption 1 restricts the analysis to nozzles having spatially uniform injection of the liquid into the gas and operating at gas-to-liquid volume ratios greater than unity. Assumption 2, the drop breakup criterion, states that drop diameter is limited to a value D for which $W_e = \rho_g V_s^2 D / 2\sigma = 6$. Thus

$$D_{max} = \frac{12\sigma}{\rho_g V_s^2} \quad (1)$$

where ρ_g is the gas density, V_s is the slip velocity $V_g - V_L$, and σ is the liquid surface tension. The form of Eq. (1) is physically reasonable in that the Weber number is proportional to the ratio of stagnation pressure $\rho_g V_s^2 / 2$ to surface tension pressure $4\sigma/D$. Hence, a drop would be expected to flatten and breakup at a sufficiently high value of W_e . This has been verified experimentally and the critical Weber number found to be 6, within a factor of about two. An additional restriction is that for actual breakup to occur, the time spent at a Weber number exceeding 6 must be longer than the natural period of oscillation of the drop, $\pi(\rho_L D^3 / 4)^{1/2} / 4$, where ρ_L is the density of the liquid. This requirement is met only in two-phase nozzles longer than about 10 in. and Assumption 2 may cause the analysis to overestimate the exit velocity by increasing amounts as the nozzle length decreases below 10 in.

Assumption 3 excludes magnetohydrodynamic and mechanical body forces. The exclusion of wall heat transfer is correct for the insulated nozzles of interest for power systems. In addition, the relatively high velocity results in short residence lines in the nozzle proper.

Assumption 4 is closely met in practical nozzles since good performance requires small wall angles, large throat radius of curvature, and uniformly distributed injection of the fluids at the nozzle entrance.

Assumption 5 is valid for the drop sizes of 0.001 to 0.010 in. produced by the Eq. (1) breakup criterion. Assumption 6 is valid because of the rapid internal circulation in drops. Assumption 7 introduces negligible error in most cases of practical interest since the vapor pressure of the liquid is small and needs only to be evaluated approximately.

Assumptions 8, 9, and 10 are either valid, or cause little error, for fluids of low miscibility, which are the fluids of interest.

3. DERIVATION OF EQUATIONS FOR FREE-STREAM FLOW

1. Continuity

Referring to Figure 1, the nozzle flow area A is equal to the gas flow area $\dot{m}_g / \rho_g V_g$ plus the liquid flow area $\dot{m}_L / \rho_L V_L$. Thus

$$A = \dot{m}_g \left(\frac{1}{\rho_g V_g} + \frac{r}{\rho_L V_L} \right) \quad (2)$$

where r is the mass mixture ratio \dot{m}_L / \dot{m}_g .

2. Momentum

By Assumption 3, the only force acting on the free-stream flow is that due to the pressure gradient. If \dot{M} is the momentum flux at flow area A, the change in momentum flux across pressure increment dp is

$$d\dot{M} = -A dp \quad (3)$$

The momentum flux can be written as the sum of the momentum fluxes of the gas and liquid. Thus,

$$\dot{M} = \dot{m}_g V_g + \dot{m}_l V_l \quad (4)$$

If the flow were allowed to continue at constant pressure, V_g and V_l would become equal to each other at the mass-weighted mean velocity \bar{V} . Since for this process, $d\dot{M} = 0$, the value of \bar{V} is given by

$$(\dot{m}_g + \dot{m}_l)\bar{V} = \dot{m}_g V_g + \dot{m}_l V_l \quad (5)$$

or

$$\bar{V} = \frac{V_g + r V_l}{1 + r} \quad (6)$$

Thus, the momentum flux can be written

$$\dot{M} = (\dot{m}_g + \dot{m}_l)\bar{V} \quad (7)$$

Since $\dot{m}_g + \dot{m}_l$ is constant, the change in momentum flux is

$$d\dot{M} = (\dot{m}_g + \dot{m}_l)d\bar{V} \quad (8)$$

Substituting Eqs. (8) and (2) into Eq. (3), $d\bar{V}$ can be written

$$d\bar{V} = -\frac{1}{1+r} \left(\frac{1}{\rho_g V_g} + \frac{r}{\rho_l V_l} \right) dp \quad (9)$$

The slip ratio is defined as

$$s = V_s/\bar{V} = (V_s - V_L)/\bar{V} \quad (10)$$

This equation can be combined with Eq. (6) to give V_g and V_L in terms of \bar{V} :

$$V_s = \left(1 + \frac{rs}{1+r}\right) \bar{V} = a\bar{V} \quad (11)$$

$$V_L = \left(1 - \frac{s}{1+r}\right) \bar{V} = b\bar{V} \quad (12)$$

The gas density can be expressed as

$$\rho_g = W_g p / RT_g \quad (13)$$

where W_g is the effective molecular weight of the gas mixture and R is the universal gas constant. Eq. (13) is the definition of the effective molecular weight W_g , which is the quantity that gives the actual gas density when substituted in Eq. (13).

Substituting Eqs. (11) - (13) into Eq. (9), the differential momentum equation is

$$2\bar{V}d\bar{V} = d\bar{V}^2 = -\frac{2}{1+r} \left(\frac{RT_g}{\sigma W_g p} + \frac{r}{b\rho_L} \right) dp \quad (14)$$

The quantities a and b are slowly varying because s is typically only 0.1 to 0.3 and slowly varying. The quantities r , T_g , W_g and ρ_L are also slowly varying. Integrating

Eq. (14) over a pressure increment Δp , for which a , b , r , T_g , W_g , and ρ_L are constant to within the desired accuracy, the change in \bar{V}^2 is

$$\Delta \bar{V}^2 = - \frac{2}{1+r_m} \left(\frac{RT_{g,m}}{a_m W_{g,m}} \log_e \frac{p + \Delta p/2}{p - \Delta p/2} + \frac{r_m \Delta p}{b_m \rho_{L,m}} \right) \quad (15)$$

All quantities other than pressure can be taken outside the integral and evaluated at their mean values (denoted by subscript m) corresponding to the mid-interval pressure p . Thus,

$$\begin{aligned} \Delta \bar{V}^2 = & - \frac{2}{1+r_m} \\ & \times \left(\frac{RT_{g,m}}{a_m W_{g,m}} \int_{p-\frac{\Delta p}{2}}^{p+\frac{\Delta p}{2}} \frac{dp}{p} + \frac{r_m}{b_m \rho_{L,m}} \int_{p-\frac{\Delta p}{2}}^{p+\frac{\Delta p}{2}} dp \right) \end{aligned} \quad (16)$$

Performing the integrations,

$$\Delta \bar{V}^2 = - \int_{p-\frac{\Delta p}{2}}^{p+\frac{\Delta p}{2}} \frac{2}{1+r} \left(\frac{RT_g}{a W_{g,p}} + \frac{r}{b \rho_L} \right) dp \quad (17)$$

Equation (17) is the final form of the momentum equation.

3. Energy

The enthalpy change of the mixture between state 1 (the beginning of pressure interval Δp) and state 2 (the end

of the interval) can be evaluated in two steps: (1) phase change at p_1, T_{g1}, T_{L1} and (2) change to p_2, T_{g2}, T_{L2} , at fixed composition.

The enthalpy change for step 1 is

ΔH_1	=	amount of A vaporized	X	enthalpy required to vaporize and heat unit mass of A from T_L to T_{gL}
	+	amount of B vaporized	X	enthalpy required to vaporize and heat unit mass of B from T_{L1} to T_{g1}
	+	amount of A and B	X	kinetic energy required to accelerate unit mass from V_{L1} to V_{g1}

or

$$\begin{aligned} \Delta H_1 = & (\dot{m}_{g2} - \dot{m}_{g1}) [L_{a1} + c_{a1} (T_{g1} - T_{i1})] \\ & + (\dot{m}_{g2} - \dot{m}_{g1}) [L_{b1} + c_{b1} (T_{g1} - T_{i1})] \\ & + (\dot{m}_{g2} - \dot{m}_{g1}) (V_{g1}^2 - V_{i1}^2) / 2 \end{aligned} \quad (18)$$

where L and c are latent heat and specific heat, respectively.

Introducing more compact notation.

$$\begin{aligned} \Delta H_1 = & \Delta \dot{m}_g (L_{a1} + c_{a1} \delta_1 T) \\ & + \Delta \dot{m}_g (L_{b1} + c_{b1} \delta_1 T) + \frac{\Delta \dot{m}_g \delta_1 V^2}{2} \end{aligned} \quad (19)$$

The enthalpy change for step 2 is evaluated from the temperature, pressure, and velocity changes, with properties evaluated at mean T and p for the interval.

$$\begin{aligned}\Delta H_2 = & \dot{m}_{s_2} [c_{s_m} (T_{s_2} - T_{s_1}) + \frac{1}{2} (V_{s_2}^2 - V_{s_1}^2)] \\ & + \dot{m}_{i_2} [c_{i_m} (T_{i_2} - T_{i_1}) + \frac{p_2 - p_1}{\rho_{i_m}} \\ & + \frac{1}{2} (V_{i_2}^2 - V_{i_1}^2)]\end{aligned}\quad (20)$$

$$\begin{aligned}= & \dot{m}_{s_2} \left(c_{s_m} \Delta T_s + \frac{\Delta V_{s_2}^2}{2} \right) \\ & + \dot{m}_{i_2} \left(c_{i_m} \Delta T_i + \frac{\Delta p}{\rho_{i_m}} + \frac{\Delta V_{i_2}^2}{2} \right)\end{aligned}\quad (21)$$

By Assumption 3, no work is done by the free-stream flow and no heat is transferred to it. Hence,

$$\Delta H_1 + \Delta H_2 = 0 \quad (22)$$

Substituting Eqs. (19) and (21) into Eq. (22) and solving for ΔT_g gives the energy equation for the mixture:

$$\begin{aligned}\Delta T_g = & -\frac{1}{c_{g_m}} \left[\frac{\Delta V_g^2}{2} + r_2 \left(c_{i_m} \Delta T_i + \frac{\Delta p}{\rho_{i_m}} + \frac{\Delta V_{i_2}^2}{2} \right) \right. \\ & + \frac{\Delta \dot{m}_{s_2} \delta_1 V^2}{2 \dot{m}_{s_2}} + \frac{\Delta \dot{m}_{s_2}}{\dot{m}_{s_2}} (L_{s_1} + c_{s_g} \delta_1 T) \\ & \left. + \frac{\Delta \dot{m}_{i_2}}{\dot{m}_{i_2}} (L_{i_1} + c_{i_g} \delta_1 T) \right]\end{aligned}\quad (23)$$

4. Drag

Although no force other than pressure acts on the free-stream flow as a whole, a drag force exists between

the phases. Hence, a second momentum equation must be written using as the control volume the boundary between the phases.

The two forces acting on each liquid drop are the buoyancy due to the pressure gradient and the drag due to the relative gas velocity. The sum of these is equal to the mass times the acceleration of the drop. Thus, for a single drop.

$$\begin{array}{rcl}
 \text{dynamic pressure of} & & \text{drag} \\
 \text{relative gas flow} & \times & \text{coefficient} \times \\
 & & \text{frontal area} \\
 & & \text{of drop} \\
 \\
 - \text{volume} & \times & \text{pressure} \\
 \text{of drop} & & \text{gradient} \\
 & = & \text{mass} \\
 & & \text{drop} \times \text{acceleration} \\
 & & \text{of drop}
 \end{array}$$

or

$$\begin{aligned}
 \left(\frac{1}{2} \rho_g |v_g| v_g \right) C_D \frac{\pi D^2}{4} - \frac{\pi D^3}{6} \frac{dp}{dx} \\
 = \left(\frac{\pi D^3}{6} \rho_l \right) \left(v_l \frac{dv_l}{dx} \right)
 \end{aligned} \tag{24}$$

The absolute value sign in the first term makes the drag force positive when $v_g > v_L$ and negative when $v_g < v_L$.

Solving Eq. (24) for dv_L ,

$$dv_L = \frac{3 \rho_g |v_g| v_g C_D dx}{4 \rho_l v_l D} - \frac{dp}{\rho_l v_l} \tag{25}$$

Differentiating Eq. (12), dV_L can also be expressed in terms of s , r , and \bar{V} . Thus,

$$dV_L = b d\bar{V} + \bar{V} \left[\frac{s dr}{(1+r)^2} - \frac{ds}{1+r} \right] \quad (26)$$

Solving for ds ,

$$ds = \frac{b(1+r) d\bar{V}}{\bar{V}} + \frac{s dr}{1+r} - \frac{(1+r) dV_L}{\bar{V}} \quad (27)$$

Substituting dV_L from Eq. (25), noting that $d\bar{V} = d\bar{V}^2/2\bar{V}$, using Eq. (12), and writing for a finite increment, results in,

$$\Delta s = \frac{b_m(1+r_m) \Delta \bar{V}^2}{2\bar{V}_m^2} + \frac{(1+r_m) \Delta p}{b_m \rho_{L_m} \bar{V}_m^2} + \frac{s_m \Delta r}{1+r_m} - \frac{3\rho_{r_m} |s_m| s_m C_{D_m} (1+r_m) \Delta x}{4 b_m \rho_{L_m} D} \quad (28)$$

This is the drag equation employed when x is specified as a function of p .

Solving Eq. (28) for Δx yields the required alternative equation:

$$\Delta x = \frac{4D}{3\rho_{r_m} |s_m| s_m C_{D_m} \bar{V}_m^2} \left[\Delta p + \frac{b_m^2 \rho_{L_m} \Delta \bar{V}^2}{2} + \frac{b_m \rho_{L_m} \bar{V}_m^2}{1+r_m} \left(\frac{s_m \Delta r}{1+r_m} - \Delta s \right) \right] \quad (29)$$

5. Heat Transfer

Although no heat is transferred to the mixture as a whole, heat transfer exists between the phases. Hence, a second energy equation must be written using as the control volume the boundary between the phases.

The work dW done on the liquid is that due to drag by the gas. (Only work done by shear or shaft forces is included in dW when writing the First Law for a control volume). Multiplying Eq. (24) by the number flow rate of drops $\dot{N} = 6\dot{m}_L/\pi D^3 \rho_L$, the drag force F_d on that quantity of liquid is

$$F_d = \frac{\dot{N}}{8} \rho_g |V_r| V_r C_{D0} D^2 = \frac{\dot{m}_1}{\rho_1} \frac{dp}{dx} + \dot{m}_1 V_1 \frac{dV_1}{dx} \quad (30)$$

The work done on the liquid is

$$-dW = F_d dx = \dot{m}_1 \left(\frac{dp}{\rho_1} + \frac{dV_1^2}{2} \right) \quad (31)$$

The heat dQ transferred from the liquid is made up of two parts: (1) the convective cooling due to the temperature difference between the liquid and gas and (2) the evaporative cooling due to the latent heat supplied to the liquid vaporized. The convective cooling is

$$-dQ_c = h A_s \dot{N} (T_1 - T_g) dt \quad (32)$$

where h is the heat-transfer coefficient, $A_d = \pi D^2$ is the surface area of a drop, and $dt = dx/V_L$ is the time required to traverse dx . Thus,

$$-dQ_c = \frac{6h\dot{m}_l(T_g - T_l)dx}{D\rho_l V_l} \quad (33)$$

The evaporative cooling is

$$-dQ_e = L_e d\dot{m}_e + L_v d\dot{m}_v \quad (34)$$

The change in enthalpy of the liquid over the pressure increment dp is

$$dH = \dot{m}_l \left(c_l dT_l + \frac{dp}{\rho_l} + \frac{dV_l^2}{2} \right) \quad (35)$$

Substituting Eqs. (31), (33), (34), and (35) into the steady-flow energy equation $dQ - dW = dH$, the result is

$$\frac{6h\dot{m}_l \delta T dx}{D\rho_l V_l} - L_e d\dot{m}_e - L_v d\dot{m}_v = \dot{m}_l c_l dT_l \quad (36)$$

where $\delta T = T_g - T_l$.

Writing for a finite interval, the final form of the drop heat-transfer equation is

$$\Delta T_l = \frac{1}{c_{l_m}} \left[\frac{6h\delta_m T \Delta x}{D\rho_{l_m} V_{l_m}} - L_{e_m} \frac{\Delta \dot{m}_{e_l}}{\dot{m}_{l_m}} - L_{v_m} \frac{\Delta \dot{m}_{v_l}}{\dot{m}_{l_m}} \right] \quad (37)$$

Equations (1), (17), (23), (23), and (37) are the five equations that must be solved simultaneously to obtain the values of the five dependent variables D , T_g , T_L , V_g , and V_L as a function of the independent variable p . To carry out the solution all quantities in the equations must be expressed in terms of these six variables.

C. WALL SHEAR AND BOUNDARY LAYER

For a two-phase nozzle, the momentum flux of the frictionless nozzle flow is that given by

$$\dot{M} = \dot{m}_t \bar{V}$$

The mean mixture density corresponding to the mean velocity \bar{V} is

$$\rho' = \frac{\dot{m}_t}{A\bar{V}} = \frac{\rho_l}{1 + r_a} \quad (38)$$

where r_a is the ratio of gas flow area to liquid flow area $\rho_l V_l / \rho_g V_g$.

From the definition of the momentum thickness, the value of θ at a station where the nozzle wall radius is y_0 is given by

$$\dot{M} - \dot{M}_f = 2\pi y_0 \theta \frac{\dot{m}_t \bar{V}}{A} = 2\pi y_0 \rho' \bar{V}^2 \theta \quad (39)$$

where \dot{M}_f is the momentum flux of the real flow with friction.

The skin-friction coefficient can be defined using the same quantities as single-phase flow

$$C_f = \frac{2\tau_w}{\rho \bar{V}^2} \quad (40)$$

where τ_w is the wall shear. It will be shown that a valid C_f value can be provided.

The boundary-layer momentum equation then becomes

$$d\theta = \frac{C_f}{2} dx - \theta \left[\frac{1 + (\delta^*/\theta)}{\bar{V}} d\bar{V} + \frac{1}{\rho \bar{V}} d(\rho \bar{V}) + \frac{1}{R_w} dR_w \right] \quad (41)$$

where δ^* is the displacement thickness, i.e., the distance the wall must be moved outward to give the same flow rate as with frictionless flow.

Assuming a $\frac{1}{2}$ power velocity profile and no density variation, the shape factor δ^*/θ is obtained from

$$\frac{\delta^*}{\theta} = \frac{\int_0^\delta [1 - (y/\delta)^{1/2}] dy}{\int_0^\delta (y/\delta)^{1/2} [1 - (y/\delta)^{1/2}] dy} = \frac{9}{7}$$

where δ is the velocity thickness of the boundary layer.

Noting that $d\bar{V}$ can be written $d\bar{V}^2/2\bar{V}$, and that $d(\rho \bar{V}) = d(m_t/A)$, the finite-difference form of Eq. (41) is

$$\Delta\theta = \frac{C_{f_m}}{2} \Delta x - \theta_m \left(\frac{8}{7\bar{V}_m} \Delta\bar{V}^2 - \frac{1}{A_m} \Delta A + \frac{1}{y_{om}} \Delta y_o \right)$$

Wall shear in homogeneous two-phase flow has been found to be equal to that which would exist with pure liquid at equal velocity and boundary-layer thickness, multiplied by the wetted wall fraction:

$$\tau_w = \frac{C_{f_l}}{2} \rho_l V_l^2 \frac{A_l}{A} = \frac{C_{f_l} \rho_l V_l^2}{2(1+r_a)} \quad (42)$$

where C_{f_l} is the skin friction coefficient for liquid at a Reynolds number of

$$R_\delta = \frac{\rho_l V_l \delta}{\mu_l}$$

For a 1/7-power profile, the velocity thickness,

$$\delta = \frac{72}{7} \theta$$

A convenient relation for C_{f_l} as a function of R_δ is the Shultz-Grunow relation which can be written

$$C_{f_l} = \frac{0.208}{(\log_{10} R_\delta + 0.425)^{2.584}}$$

Comparison of Eqs. (40) and (42) shows that C_f can be written

$$C_f = \frac{rb}{1+r} C_{f_l}$$

Thus, the final form of the boundary-layer momentum equation is

$$\Delta\theta = \frac{r_m b_m}{1+r_m} \frac{C_{f_l} \Delta x}{2} - \theta_m \left(\frac{8\Delta\bar{V}^2}{7\bar{V}_m^2} - \frac{\Delta A}{A_m} + \frac{\Delta y_n}{y_{on}} \right)$$

Let \bar{V}_δ be the mean velocity of the flow including the boundary layer. Then, from Eq. (39),

$$\dot{m}_t \bar{V}_\delta = \dot{M}_t = \dot{m}_t \bar{V} - 2\pi y_o \rho \bar{V}^2 \theta$$

Hence, employing Eq. (38), the mean exit velocity including the boundary layer is

$$\bar{V}_\delta = \bar{V} \left(1 - \frac{2\pi y_o \theta}{A} \right)$$

By the definition of the displacement thickness, the flow rate is reduced by the throat displacement thickness δ_t^* to

$$\dot{m}_\delta = \dot{m}_t \left(1 - \frac{2\delta_t^*}{y_o} \right)$$

D. NOZZLE THEORY SUMMARY

The preceding equations form the basis for the mathematical model which is used to predict, based on inlet conditions, the exit velocity, and temperature of the mixture. These equations also form the basis for the model which provides the optimum nozzle shape given a set of inlet conditions. Some additional relationships are, however, required. These are:

1. Phase properties - to establish the mass ratio, mass flow rate ratio of gas to liquid, and the thermal conductivity of the mixture.

2. Liquid drop drag coefficients.
3. The liquid drop heat transfer coefficients.
4. Boundary layer momentum thickness and displacement thickness.
5. Skin friction coefficient.

These five additional relationships are developed in detail in Reference [1].

III. COMPUTER PROGRAM DUAL-PHASE NOZZLE

The computer program employed in this study is based on a program developed by Dr. G. Elliott of the Jet Propulsion Laboratory in Pasadena, California. The program was updated and converted for use on the Naval Postgraduate School computer. The Dual-Phase Two-Component program employs the theory in Section II. The program is written in Fortran computer language and can be compiled using a Watfiv or Fortran IV compiler.

This program has been utilized in dual-phase nozzle analysis and to provide values for comparison with the experimental results. To use the computer program the inlet conditions have to be specified. The flow conditions are: inlet pressure; mass ratio; inlet temperature of the gas and liquid; inlet velocity of the gas and liquid; total mass flow rate; and nozzle exit pressure. Section III B, shows specified details for data input.

There are two options that can be chosen. The first is prescribed pressure-versus-distance option MOP=0. The pressure profile $P(X)$ is selected corresponding to the adopted nozzle contour. If the pressure-versus-distance is used a $P(X)$ input table is required. This profile is developed from the actual measured pressure values in the experiments.

(See Appendix A for sample program.) The second option consist of an optimum nozzle contour option MOP=1. This option is useful only when the liquid drop diameter is constant.

The dual-phase two-component computer program is a structured program with thirteen subroutines controlled by a main program. This arrangement improved the programming process through better organization and programming notation.

The control point of the dual-phase two-component computer program is the "main section." It controls the flow path and operation of all input data, property tables, and calculations. It accomplishes this by calling the thirteen subroutines at the appropriate times, saving wanted data in files, and printing out desired information.

One of the most important subroutines which inputs information is the "INTRP" subroutines. INTRP controls the property table inputs. It reads in four two-dimensional tables and fourteen one-dimensional tables. These inputs are the properties of the gas and liquid phases of both components of the flow in the nozzle. The subroutine writes the values of these tables into a file and retrieves appropriate values from that file. INTRP can also interpolate for values used throughout the entire program.

Input of case data is controlled by subroutine "Sect 1." Identification information and case heading information is read and printed for each instance. Sect 1 also places the pressure vs. distance profile, if specified, in an array.

"Sect 2" through "Sect 6" are the subroutines which calculate the flow data. Sect 2 sets the initial conditions indicated in the input, and begins the iterations. Sect 3 computes initial flow rates of both components; the initial area of the nozzle; slip velocity; mean free stream velocity; and slip friction. Sect 4 computes the changes in flow parameters and new distances. It then begins to calculate new conditions such as flow rate, temperature, velocities, surface tension and mean area. Sect 5 is the binary cut convergence routine and computes mean boundary-layer parameters.

If a problem is diagnosed in any subroutines and "diagnose" is called, it will print all output parameters calculated to that point. It also does the same if there is a convergence problem.

There are two subroutines that output calculated data, subroutine "Write" and subroutine "Output." Subroutine "Write" will send output information to the printer for a hard paper copy. Subroutine "Output" reads and stores the output on a file.

The two-phase two-component program has been written with comment statements in the text of the program. These will allow for a more understandable and, therefore, a more easily modified program. For specific details on the content of these subroutines, see Appendix M.

The dual-phase nozzle program was tested for correct output. Sample data and results were obtained from

Dr. David Elliott, were inputted into the program, and executed. The output was compared to the sample data. The program produced duplicate results.

The program begins by storing fluid property tables and reading in all input data. All nozzle inlet conditions are computed. The program then proceeds half a pressure step at a time. At the middle of each pressure interval, the changes in quantities across the interval are computed using the properties interpolated from the table for that pressure, and for the existing temperature. The change in slip is found if the pressure profile $P(X)$ is specified. At the end of each pressure step, the flow conditions are updated and initial conditions are determined for the next step. The dropsize is reduced at the point when the Weber number exceeds six. The flow conditions are printed if the pressure is one selected for output. The computation continues until the last pressure step has been completed and flow conditions at the smallest flow area encountered are printed as the throat conditions.

A. PROPERTY TABLES

1. Heat capacity of component "A" vapor in BTU/LBM-R is a function of temperature and pressure. The two-dimensional tables are entered row-wise. At least two cards are necessary to specify a row and at least two rows must be entered.

Card 1: (format 6E12.6)

cols.

1-12 temperature (R)

13-24=1.0 if this is the last temp for
this table

Card 2: (format 6E12.6)

cols.

1-12 pressure (psi)

13-24 heat capacity (BTU/LBM-R)

25-36 pressure (psi)

37-48 heat capacity (BTU/LBM-R)

49-60 pressure (psi)

61-72 heat capacity (BTU/LBM-R)

The maximum entries of temperature are 35 values. For each value of temperature, the maximum number of entries of pressure and heat capacities are 35 values. Each row of this table will be terminated with the pressure and heat capacity equal to 10^5 . These two values are not counted in the maximum of 35 entries/rows allowed.

The program shown in Appendix B can be used to determine the values of heat capacities. The program structures its output in the format needed for the table input. It uses input data obtained from Reference 3. The input data must be placed in a two-dimensional table. This table is used in the program to interpolate the values needed for output. Input data must be formatted as follows:

Card 1-16 (format 10F7.4)

cols.

1-7 temperature (R)

at this temp the following is entered:

8-14 heat capacity at .01 P

15-21 heat capacity at .4 Pa

22-28 heat capacity at .7 Pa

29-35 heat capacity at 1.0 Pa

34-42 heat capacity at 4.0 Pa

43-49 heat capacity at 7.0 Pa

50-56 heat capacity at 10.0 Pa

57-63 heat capacity at 40.0 Pa

64-70 heat capacity at 70.0 Pa

The temperature must be entered with
increasing value.

2. Heat capacity of component "B" gas, BTU/LBM-R is a function of temperature and pressure. This two-dimensional table has the same format as part A1 above.

3. Molecular weight of component "A" vapor is a function of temperature and pressure. The two-dimensional tables are entered row-wise. At least two cards are necessary to specify a row and at least two rows must be entered.

Card 1: (format 6E12.6)

cols.

1-12 temperature (R)

13-24=1.0 if this is the last temp for
this table

Card 2: (format 6E12.6)

cols.

1-12 pressure (psi)

13-24 molecular weight

25-36 pressure (psi)

37-48 molecular weight

49-60 pressure (psi)

61-72 molecular weight

The maximum number entries of temperature are 35 values. For each value of temperature, the maximum entries of pressure and molecular weight are 35 values. Each row of this table will be terminated with the pressure and molecular weight equal to 10^5 . These two values are not counted in the maximum of 35 entries/rows allowed.

The program shown in Appendix C can be used to determine the values of molecular weight. The program formats its output in the format needed for the table input of the two-component two-phase computer program. It uses input data obtained from Reference 3. The input data must be placed in a two-dimensional table. This table is used in the program to interpolate the values needed for output. The program used in Figure 8 can only be used with ideal gases. Input data must be formatted as follows:

Card 1-16 (format 10F7.4)

cols.

1-7 temperature (R)

at this temp the following is entered:

8-14 density at .01 P

15-21 density at .4 Pa

22-28 density at .7 Pa

29-35 density at 1.0 Pa

34-42 density at 4.0 Pa

43-49 density at 7.0 Pa

50-56 density at 10.1 Pa

57-63 density at 40.0 Pa

64-70 density at 70.0 Pa

The temperatures must be entered with
increasing value.

4. Molecular weight of component "B" gas is a function of temperature and pressure. This two-dimensional table has the same format as part A3 above.

5. There are fourteen one-dimensional tables. The one-dimensional tables are entered in the following format (for Z(T)):

Card 1: (format 6E12.6)

cols.

1-12 Ti-2, oR

13-24 Ti-2

25-36 T_i-1 $2 \leq i \leq 50$

37-48 Z_i-01

49-60 T_i

61-72 Z_i and $T_j-1 < T_j$ for $2 \leq j \leq 50$

Each table is terminated by two consecutive entries of 10^5 , i.e., $T_k \equiv Z_k \equiv 10^5$ (1.0E5 right adjusted in the field).

The fourteen one-dimensional tables are (in order requested):

1. CAL(T) heat capacity of component A liquid,
BTU/LBM or
2. CBL(T) heat capacity of component B liquid,
BTU/LBM or
3. LA(T) latent heat of vaporization for com-
ponent A, BTU/LBM
4. LB(T) latent heat of vaporization for com-
ponent B, BTU/LBM
5. PBO(T) vapor pressure of component B
6. ROAL(T) density of liquid component A,
LBM/FT³
7. ROBL(T) density of liquid component B,
LBM/FT³
3. KAG(T) thermal conductivity of component
A gas, BTU/FT HR or
9. KBG(T) thermal conductivity of component
B gas, BTU/FT HR or

10. VIAL(T) viscosity of liquid component A,
LBM/FT HR
11. VIBL(T) viscosity of liquid component B,
LBM/FT HR
12. VIAG(T) viscosity of gas component A,
LBM/FT HR
13. VIBG(T) viscosity of gas component B,
LBM/FT HR
14. SIG(T) surface tension of liquid component
B, DYNE/CM

Appendix D gives sample property table for input.

B. CASE INPUT

A blank card must separate the property table from the data set decks following.

Card 1: (format 4A4, A2, 3A4, 7A4, A32, 2I6)
cols.

1-18 date

19-30 case number, may be any

alphanumeric data

31-60 identification

61-66 NS, number of pressure steps per
printout

(right justified integer).* Use -1 if new
property tables follows.

67-72 NP, number of printouts (right
justified)

Card 2: (All integers right justified in field)

(format 1116)

cols.

1-6 MBU, =0 constant droplet size, =1 drop
breakup

7-12 MOP, =0 X(P) table to be supplied, =1
X(P) is to be optimized

13-18 MGEO, =0 circular, =1 annular

19-24 NDS, maximum number of S iterations

25-30 NSO, maximum number of So iterations

31-36 NB, maximum number of TB iterations

37-42 NNS, first setting of step counter

Card 3: (format 5E12.6)

cols.

1-12 DP, pressure step size, negative for
decreasing, psi

13-24 RC, mass flow ratio

25-36 PHI, angle of annular nozzle axis, deg

37-48 RAXO, annular nozzle axis, in.

49-60 EMT, total flow rate *P final = $P_0 +$
 $(NS*NP-NNS-1)*DP+DP1$

Card 4: (format 6E12.6)

cols.

1-12 H, inverse Henry's Law constant, psi

13-24 ALAM, Lagrangian multiplier

25-26 DPL, first pressure step size
37-48 WAL, molecular weight of liquid a
49-60 WBL, molecular weight of liquid b
61-72 SA, Sutherland constant for component
A, oR

Card 5: (format 6E12.6)

cols.

1-12 SB, Sutherland constant for component
B, oR
13-24 DO, initial drop diameter, in.
25-36 PO, initial pressure, psia
37-48 TGO, initial gas temperature, oR
49-60 TLO, initial liquid tempeature, oR
61-72 VGO, initial gas velocity, FT/S

Card 6: (format 6E12.6)

cols.

1-12 VLO, Initial liquid velocity, FT/S
13-24 THOO, initial momentum thickness of
outer wall boundary layer, in.
25-36 THIO, initial momentum thickness of
inner wall boundary layer, in.
37-48 EDS, convergence criterion for S
49-60 ESO, convergence criterion for So
61-72 EB, convergence criterion for T, oR
If MOP=0, the following cards are present:
in 10 Card 7: (format 7A4, A2) cols.

1-30 X(P) table identification (and
alphanumeric data).

Card 8: (format 6E12.6)

cols.

1-12 pressure, pi-2, psia

13-24 distance, xi-2, in.

25-36 pressure, pi-1 $3 \leq i \leq 75$

37-48 distance, xi-1

49-60 pressure, pi

61-72 distance, xi

The last two entries are 1.0E5 and 1.0E5 right adjusted in their fields. The table must be monotonic increasing or decreasing. New property tables may be used by putting -1 in cols. 61-66 of Card 1, and following this with new property tables and data sets. Appendix E is a sample input data.

C. OUTPUT

For each case, the case identification is printed followed by the input parameters. If MOP=0, the X(P) table forms a part of this output. The following output then appears.

1. X distance, in.
2. P pressure, psia
3. R mass flow ratio
4. vb mean free-stream velocity, ft/s
5. a flow area, in. ²

- 6. tb gas temperature, or
- 7. tl liquid temperature, or
- 8. vg gas velocity, ft/s
- 9. vl liquid velocity, ft/s
- 10. vs slip velocity $vg - vl$, ft/s
- 11. s slip fraction vs/vb
- 12. d drop diameter, in.
- 13. rv ratio of gas volume flow to liquid volume flow
- 14. ra ratio of gas flow area to liquid flow area
- 15. alpha mass fraction of component a dissolved in liquid
- 16. beta mass fraction of component b vapor in gas
- 17. mg gas flow rate, lbm/s
- 18. ml liquid flow rate, lbm/s
- 19. rog gas density, lbm/ft³
- 20. rol liquid density, lbm/ft³
- 21. wag molecular weight of component a gas
- 22. wbg molecular weight of component b gas
- 23. wg mean molecular weight of gas
- 24. pa partial pressure of component a, psia
- 25. pb partial pressure of component b, psia
- 26. la latent heat of vaporization of component a, btu/lbm
- 27. lb latent heat of vaporization of component b, btu/lbm

28. sigma liquid surface tension, dyne/cm
29. cgm specific heat of gas (at midpoint of pressure step), btu/lbm of
30. clm specific heat of liquid, btu/lbm of
31. vigm viscosity of liquid, lbm/ft hr
32. vilm viscosity of gas, lbm/ft hr
33. kgm thermal conductivity of gas, btu/hr ft of
34. rem reynolds number of flow over drops
35. cdm drag coefficient of drops
36. hm heat transfer coefficient of drops, btu/hr ft²
37. rb mass flow ratio after velocity and temperature equalization
38. ab flow area after equalization, in. ²
39. tb temperature after equalization, or
40. rvb volume flow ratio after equalization
41. alphb alpha after equalization
42. betab beta after equalization
43. mgb gas flow rate after equalization, lbm/s
44. mlb liquid flow rate after equalization, lbm/s
45. vilb liquid viscosity after equalization, lbm/ft hr
46. rogb gas density after equalization, lbm/ft³
47. rolb liquid density after equalization, lbm/ft³
48. wagb molecular weight of component a gas after equalization

49. wbgb molecular weight of component b gas after equalization
50. wgb mean molecular weight of gas after equalization
51. pab partial pressure of component a gas after equalization, psia
52. pbb partial pressure of component b gas after equalization, psia
53. g separator friction parameter
54. ref separator film reynolds number
55. yo distance from nozzle axis to outer wall, in.
56. wom angle of outer wall relative to axis, deg
57. tho momentum thickness of outer boundary layer, in.
58. delo velocity thickness of outer boundary layer, in.
59. delso displacement thickness of outer boundary layer, in.
60. redom reynolds number of outer boundary layer
61. cfom skin friction coefficient of outer boundary layer
62. twom shear stress on outer wall, psi
63. vbd mean velocity including boundary layer, ft/s
64. wim angle of inner wall relative to axis, deg
65. thi momentum thickness of inner boundary layer, in.
66. deli velocity thickness of inner boundary layer, in.
67. delsi displacement thickness of inner boundary layer, in.

68. redim reynolds number of inner boundary layer
69. cfim skin friction coefficient of inner boundary
layer
70. twin shear stress on inner wall, psi
71. nna number of iterations required to optimize X(P)
72. nis number of iterations required to converge on
s or so
73. nib number of iterations required to converge on
tb

IV. EXPERIMENTAL SYSTEM

The experimental system can be grouped into three subsystems. These are:

- a) nozzle
- b) air supply system
- c) liquid injection system

Each subsystem is described in the following sections.

Figure 11 is an overall system schematic.

A. NOZZLE

The nozzle has a convergent-divergent flow passage. It is 12 inches long with a variable exit area. The exit area can be varied from .45313 square inches to .84375 square inches. The pivot point is located 1 inch above the throat. This causes the throat to vary when the exit is varied. Since the change in the throat is negligible, it will be considered to be constant. It has a throat area of .45 square inches. The inlet area is 1.625 square inches. The throat is located 4 inches from the inlet. The nozzle is constructed by sandwiching two 1/2 inch thick machined aluminum nozzle profile plates between 1/2 inch plexiglas plates (Fig. 12). The aluminum nozzle plates are located at the end of a 30 inch long entry section and are easily adjustable. Figure 13 shows a close up of the aluminum section of the nozzle.

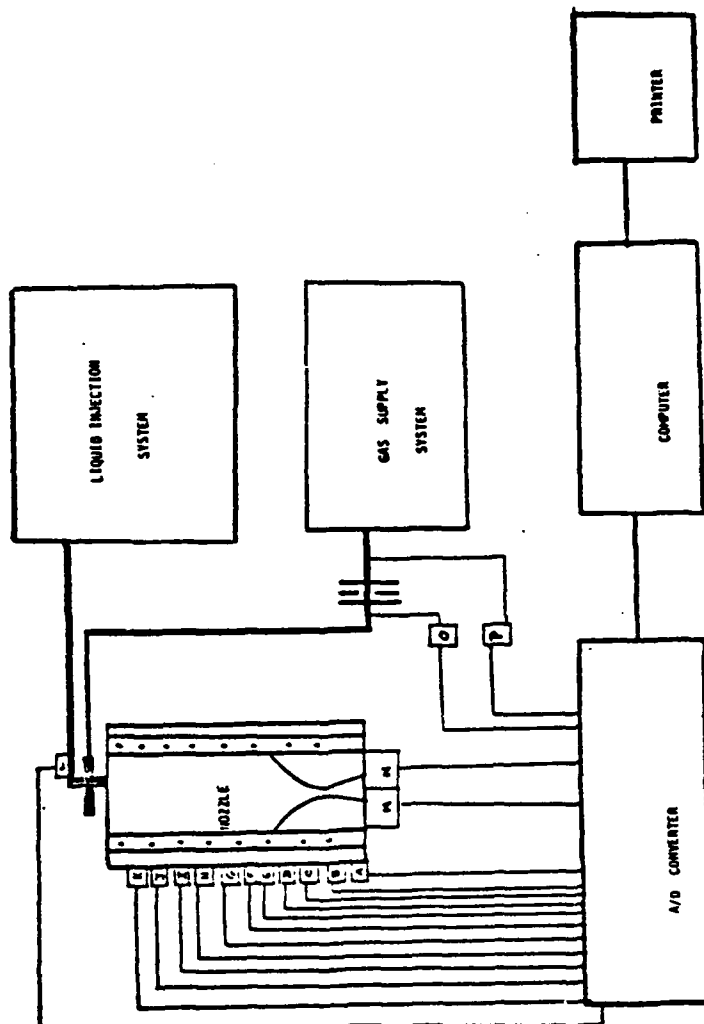


Figure 11. Experimental System Schematic

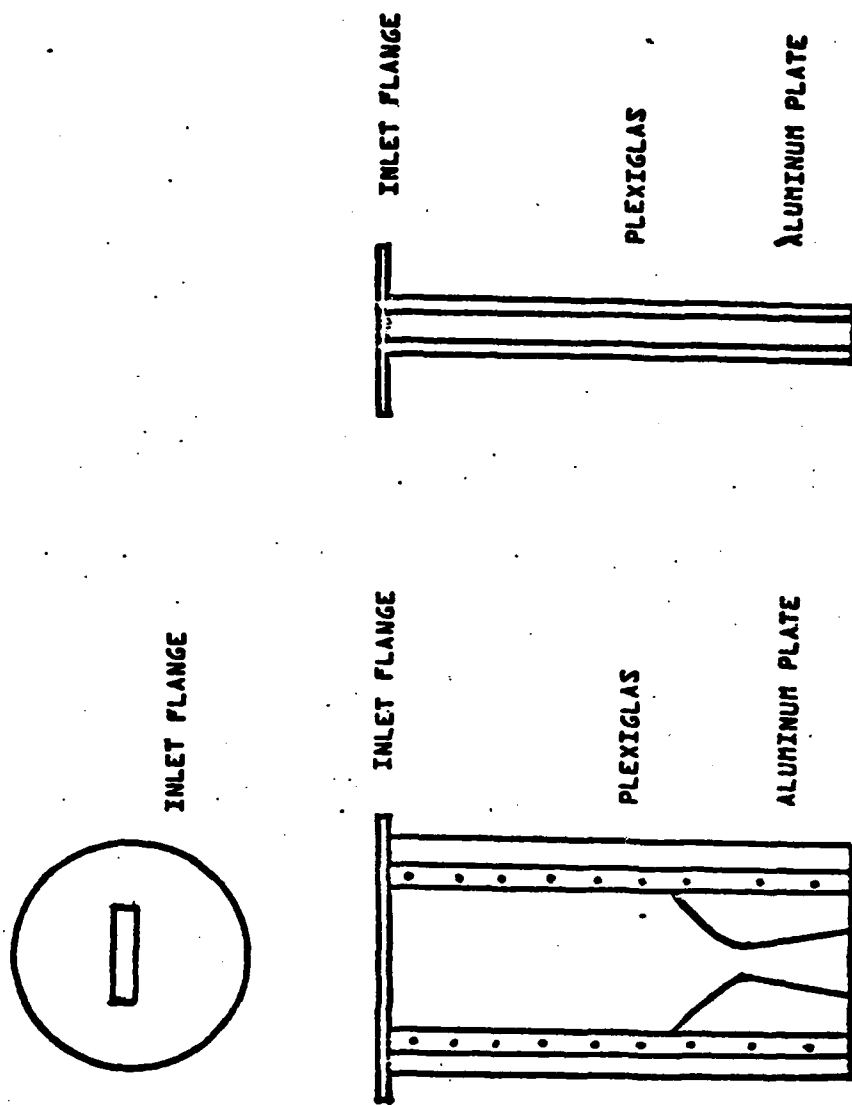
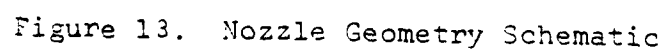


Figure 12. Nozzle Assembly Drawing



The inlet section provides the necessary space for pressure taps, liquid injection tubes, and other instrumentation and configuration options.

B. AIR SYSTEM

The air system is shown in Figure 14. Two 117 cubic feet tanks supplied with compressed air from an Ingersoll Rand two-stage air compressor provides the air storage volume required to support the nozzle's operation. Each storage tank has a pneumatically control Norgren gate valve at its exit. These valves are operated by a nitrogen actuator and controlled by a pressure regulator. The nitrogen is regulated to 40 psi control pressure which will open the Norgren valve.

The nozzle is supplied with air via a 3" i.d. pipe. The air supply to the nozzle is controlled by a solenoid actuated nitrogen operated 3 inch ball valve. The nitrogen is supplied via a regulator. By varying the nitrogen supply pressure to the ball valve, supply air pressure to the nozzle can be controlled. Air flow to the nozzle is measured with a standard ASME orifice plate. Figure 15 shows the dimensions of the orifice. The orifice is a model D-10512 with a 0.920 inch bore.

C. LIQUID INJECTION SYSTEM

The liquid injection system, Figure 16, is supplied by house water and is further pressurized by an Aurora electric

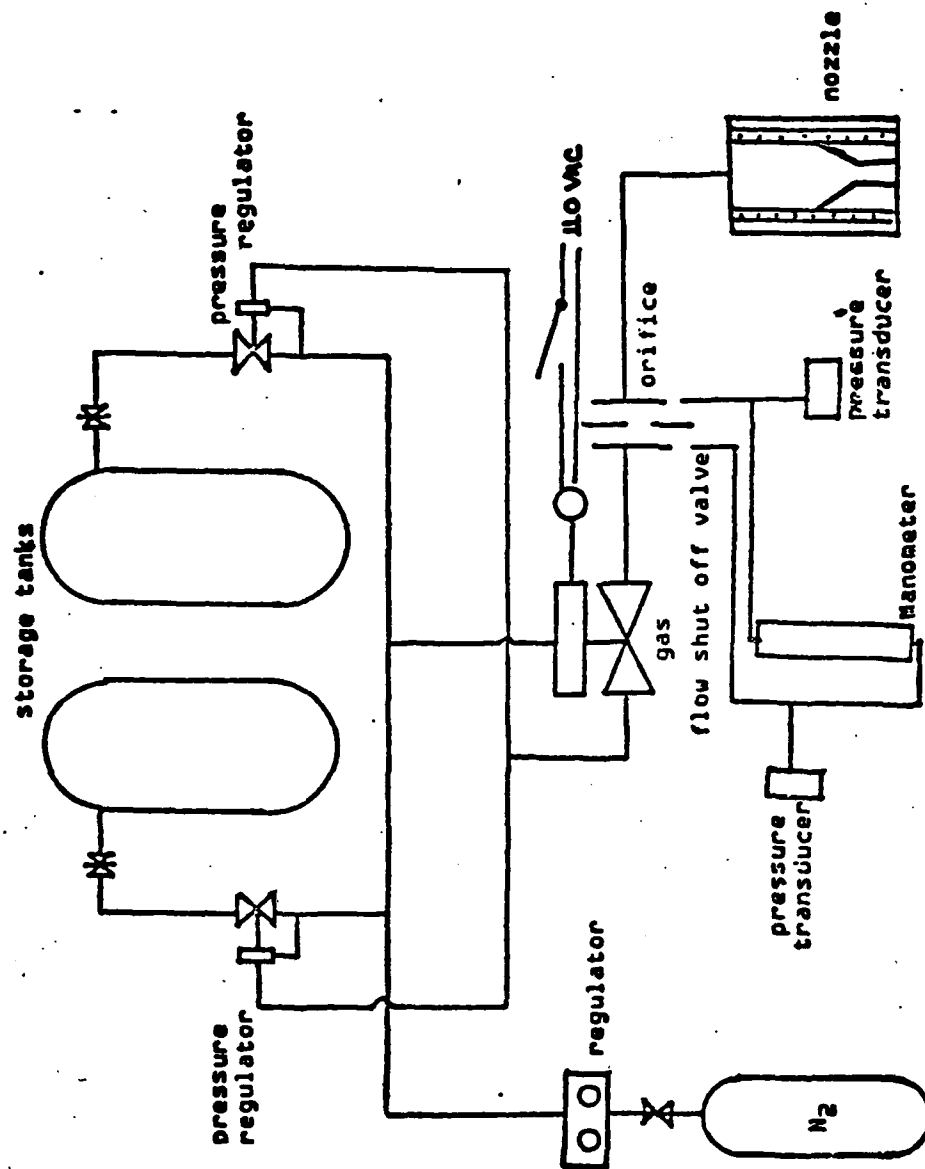


Figure 14. Orifice Schematic

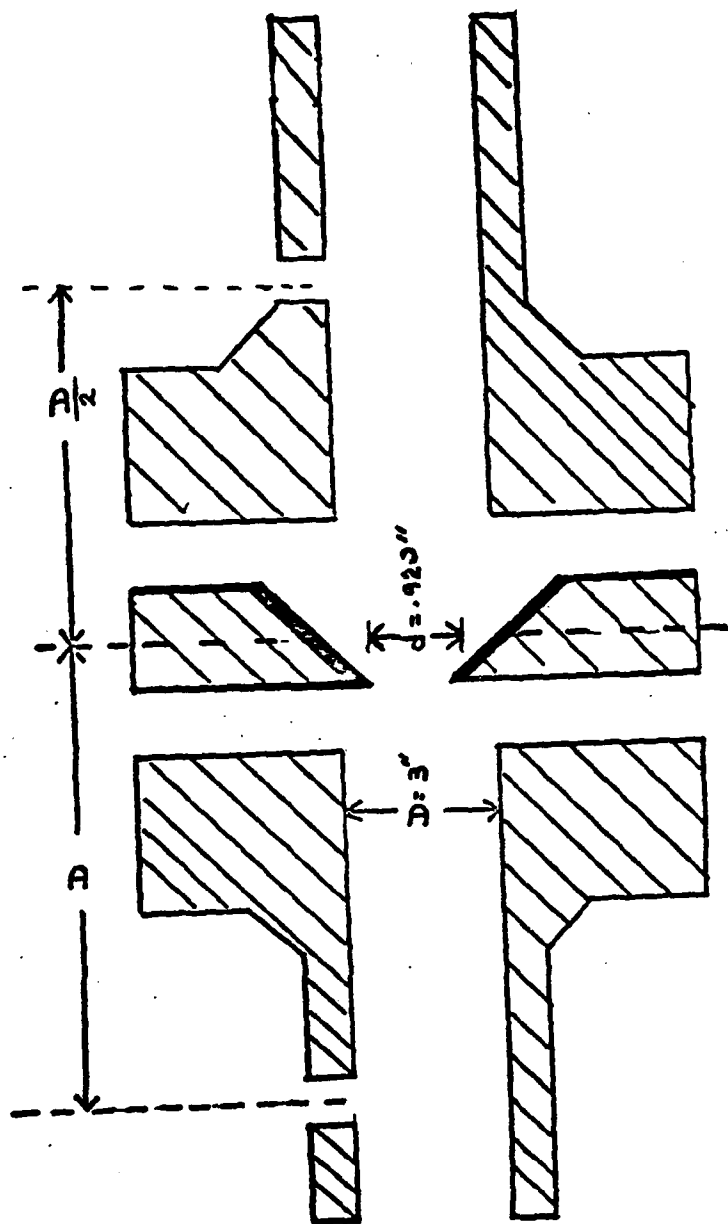


Figure 15. Air System Schematic

pump model/type 317696. The pump is rated at 50 gallons per min. The pump supplies pressurized water to the injection tube via two flowmeters (rotometers). The flowmeters are F&P co. precision bore flowrator tubes. One rotometer is rated at 1 to 12 gallons per min. and the other at .6 gallons per min.

The liquid injector is a 0.25 inch brass tube inserted in the 3" i.d. air supply pipe just upstream of the flange connection to the test section. The injector tube is drilled with sixteen 1/16th inch diameter holes facing the test section entrance. The drilled holes were made as small as possible consistent with achieving a significant liquid mass flow rate.

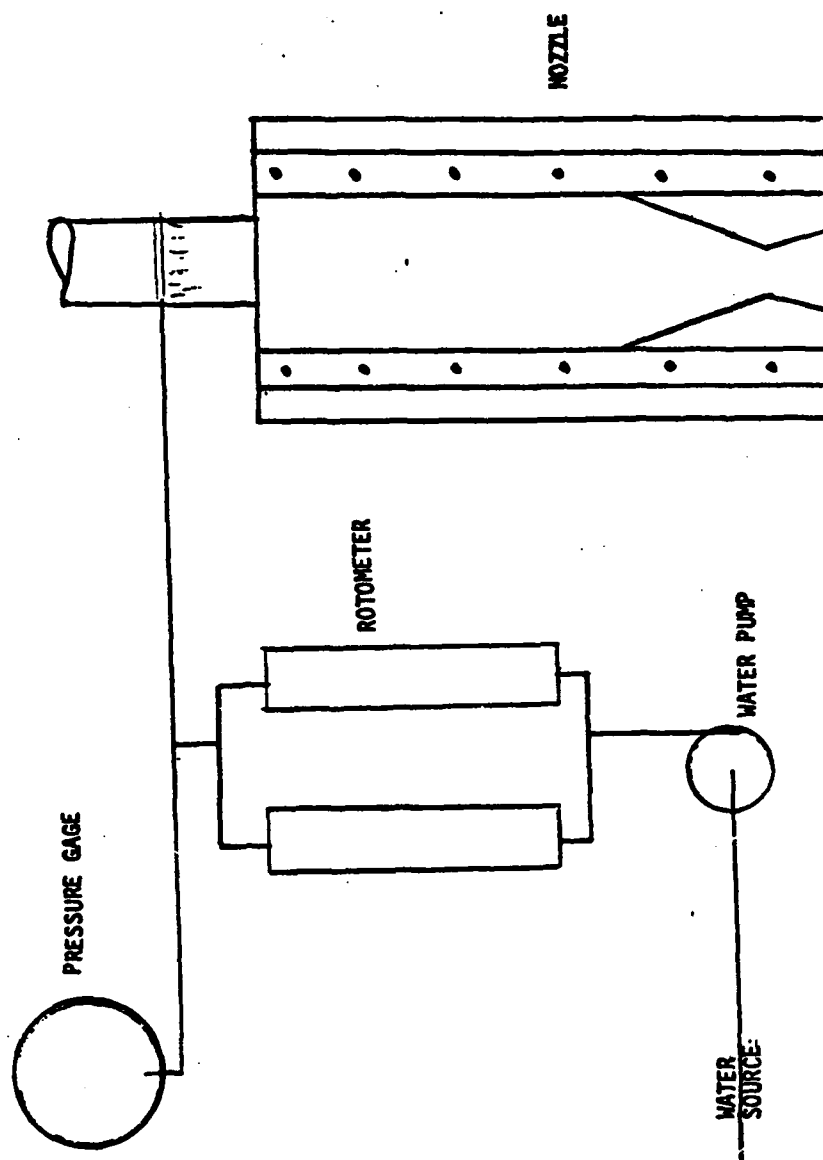


Figure 16. Liquid Injection System Schematic

V. INSTRUMENTATION SYSTEM

The instrumentation system is designed for automated data acquisition, analysis, and display. A schematic is depicted in Figure 17. Where ever possible, test operations and sequencing are under direct computer control. Each parameter measured involves an appropriate transducer, excitation source, and calibration procedure. The major instrumentation sub-systems are:

- (A) Pressure Measuring Transducers
- (B) Nozzle Thrust Force Block
- (C) Flow Measurement Devices
- (D) Data Acquisition/Control System

A. PRESSURE MEASURING TRANSDUCERS

Pressure measurements are made in fourteen locations throughout the experimental apparatus. Eleven Micro Switch 140PC pressure transducers model PK 87633 are placed on the nozzle assembly to measure pressure at various axial positions. Specifications for this model transducer are shown in Figure 18. The first pressure tap is located at one half inch from the inlet along the axis of the nozzle. The remainder are placed at one inch intervals toward the nozzle exit. These pressure taps are connected to the pressure transducers via a 1/4" o.d. plastic tubing. The

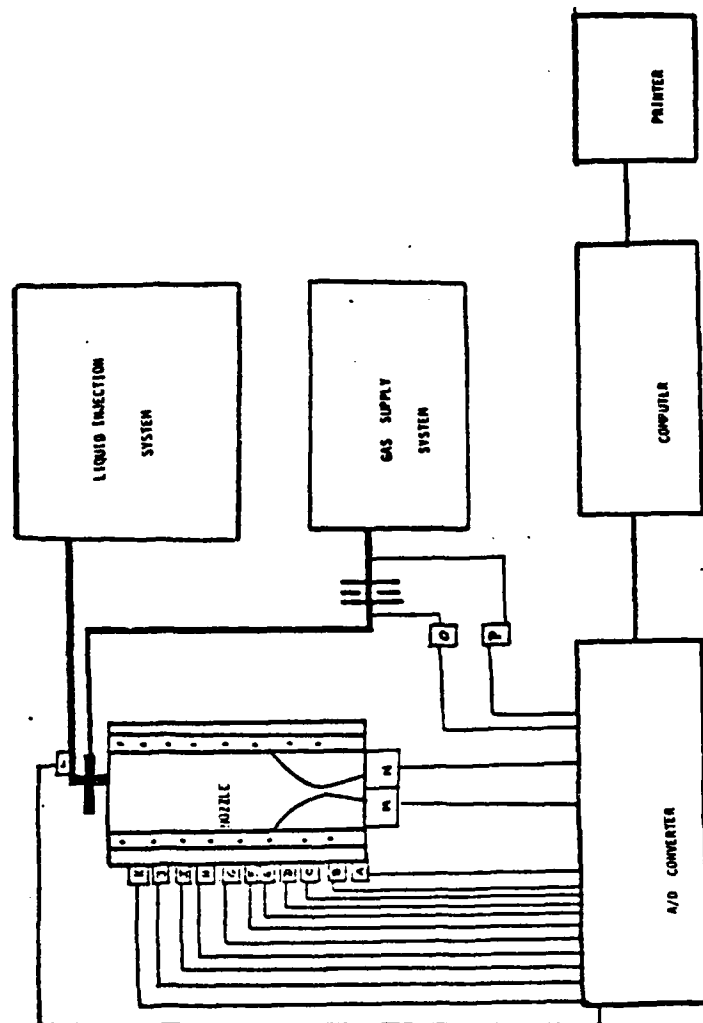
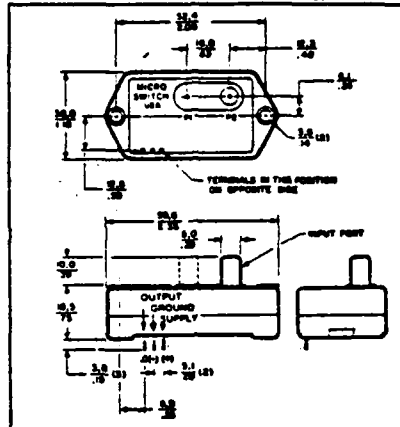


Figure 17. Instrumentation System Schematic

MOUNTING DIMENSIONS (for reference only)



140PC SPECIFICATIONS at 0.5 0.01VDC, 25°C

ALL LISTINGS

PARAMETER	PRESSURE RANGE (psi)	Min.	Typ.	Max.	UNITS
F.S.O. (Full Scale Output)*	All	4.85	5.00	5.15	Volts
Null Offset	All	0.95	1.00	1.05	Volts
Excitation	All	8.00		20.0	VDC
Output Current Source	All	10.0			mA
Supply Current (10K ohm load)	All	5.0	8.0		mA
Overpressure	0-1 0-5 0-15 0-15/0-30(L)			20 25 45 60	psi
Operating Temperature		-55°C to +125°C (-65°F to +257°F)			

ELECTRICAL AND PRESSURE CONNECTIONS

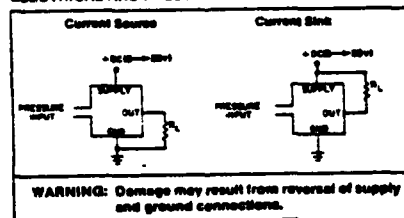


Figure 18. 140PC Pressure Transducer Specifications

pressure transducers are mounted on the side panel of the nozzle. See Figure 19 for locations. Each transducer has an identification letter corresponding to its position. Table I is the ID code for the transducers.

At the nozzle inlet a Micro Switch 200PC pressure transducer model PK 87690 is placed in the inlet line to measure the dual-phase mixture inlet pressure. Specifications for this model transducers are shown in Figure 19. Gas supply pressure is measured using a 200PC pressure transducer located at the orifice outlet. This transducer is also used to help measure differential pressure across the orifice. Another 200PC transducer is placed at the inlet of the orifice. The data acquisition program calculates the orifice differential pressure and then the air mass rate using the output from the above two transducers.

The pressure transducers provide output voltage proportional to applied pressure. These operate from a single, positive supply voltage ranging from 8 to 20 VDC. The supply voltage was maintained at 15VDC.

Each pressure transducer sends back a signal proportional to the input pressure. The signal is converted in the A/D converter to a digital signal which can be read by the HP9826. The pressure transducers were calibrated using a known pressure source. Appendix F depicts a sample calibration program written for a HP9826 computer. This program reads ten



* If S.D. is the square difference between end points (full and full pressure outputs).

ELECTRICAL AND PRESSURE CONNECTIONS



Table I. ID Code for Instrumentation

<u>Letter Designation</u>	<u>Description</u>
A	10.5" from inlet
B	9.5" from inlet
C	8.5" from inlet
D	7.5" from inlet
E	6.5" from inlet
F	5.5" from inlet
G	4.5" from inlet
H	3.5" from inlet
I	2.5" from inlet
J	1.5" from inlet
K	0.5" from inlet
L	inlet nozzle pressure
M	force-block signal
N	force-block excitation voltage
O	orifice exit pressure
P	orifice inlet pressure

$$\text{Pressure} = A + B(\text{volts}) + C(\text{volts})^2 + D(\text{volts})^3$$

pressure transducer	A	B	C	D
A	-6.049	3.926	-.17336	.01121
B	-7.059	4.234	-.22921	.01415
C	-.6491	3.934	-.16025	.00945
D	-7.056	4.692	-.34118	.01898
E	-7.009	4.152	-.20717	.01237
F	-6.389	3.933	-.16055	.00951
G	-6.589	3.956	-.16427	.00961
H	-6.588	3.908	-.15806	.00942
I	-6.595	3.949	-.16270	.00957
J	-6.353	3.927	-.16509	.00998
K	-6.203	3.846	-.14697	.00880
L	-20.207	11.157	-.08591	.00492
O	-20.668	11.068	-.07562	.01118
P	-18.827	11.928	-.17715	.01101

Figure 20. List of Polynomial Coefficients
for Pressure Transducer

values of pressure for each 140PC transducer and gives the mean and standard deviation for each from 0 to 45 psi in 5 psi increments. Appendix G depicts sample output of the program for a given pressure. Appendix H illustrates the program used to calibrate the 200PC pressure transducers. The program works as above, except values are taken from 0 to 60 psi at 5 psi increments.

Data obtained during calibration source pressure is plotted for each pressure transducer (Appendix I). Each plot was curve fitted with a third order polynomial. Figure 20 shows coefficients of the polynomial. These polynomials are used in the data acquisition/control program to convert transducer readings to pressure readings.

B. NOZZLE THRUST FORCE-BLOCK

The thrust produced by the nozzle was employed to deduce the nozzle exit velocities. The thrust was measured by instrumenting a target plate in the exit flow field. Appropriate screens were installed to prevent liquid "bounce back." The jet momentum force on the target is acquired by a balance beam system shown in Figure 21. A Kistler-Morse force block provides an analog signal proportional to the nozzle jet momentum.

The calibration of the force-block was completed by placing known weights on the force-block side of the balance

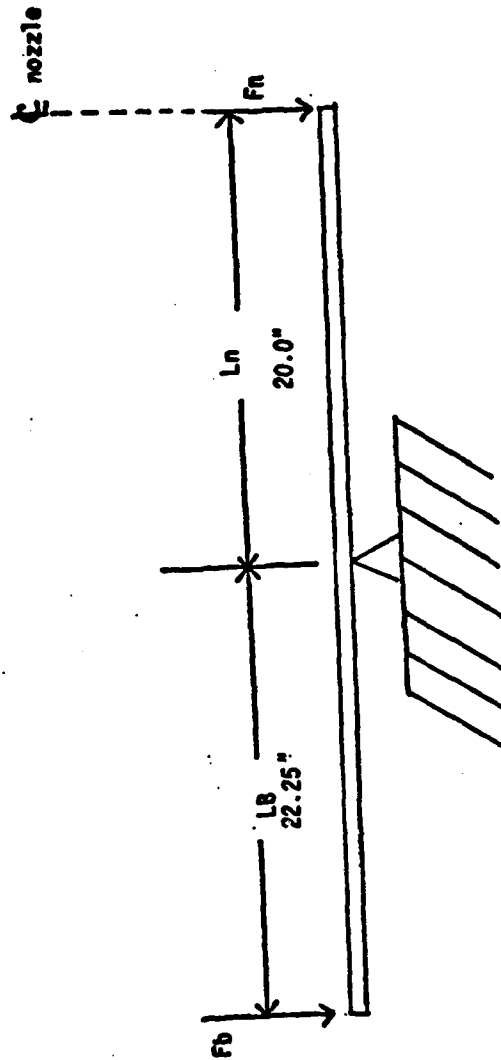


Figure 21. Force-Block Balance Beam Diagram

beam, (Figure 21), and recording the voltage produced.

Appendix J shows results of this calibration. These results were curve fitted with a third order polynomial. The following is the volt-to-force conversion polynomial:

$$FB = (1/453.6)*(-234.396+48715.28*VOLTS-11261.54*VOLTS+33899.3*VOLTS)$$

$$FN = FB*LB/LN$$

$$FN = \text{FORCE AT NOZZLE EXIT (LBF)}$$

$$FB = \text{FORCE AT FORCE BLOCK (LBS)}$$

$$LB = \text{LENGTH FROM PIVOT TO FORCE BLOCK(IN.)}$$

$$LN = \text{LENGTH FROM PIVOT TO DIRECTIONAL OBSTRUCTION(IN.)}$$

C. FLOW MEASUREMENT

1. Air mass flow measurement for the system achieved by measuring the inlet and outlet pressure on a D-10512 orifice with a 0.92 inch bore. For details on orifice see Section III. Air flow is calculated in the data acquisition program. The program uses the following equation obtained from References 4 and 5.

$$M_{\text{air}} = K A_2 Y \sqrt{2G_c \rho_1 (P_1 - P_2)} \quad \text{where}$$

$$K = CE$$

$$E = 1/(1-B^4)^{1/2}$$

$$B = d/D$$

$$C = 0.60$$

$$A_2 = \text{AREA OF ORIFICE} = \frac{\pi D^2}{4} \quad \text{FT}^2$$

$G_c = 32.2 \text{ FT/SEC}^2$

$\rho_1 = \text{DENSITY LBM/FT}^3$

$P_1 = \text{PRESSURE AT INLET IN LBF/FT}^2$

$P_2 = \text{PRESSURE AT LUTLET IN LBF/FT}^2$

$M_{\text{air}} = \text{AIR MASS FLOW RATE LBM/SEC}$

The discharge coefficient C is the factor that accounts for losses through the orifice. Since the values of C varies from .62 to .60 for R_d number from 10^4 to 10^7 with $\beta = .3$, C will be considered constant.

2. Water flow measurement was made using two rotometers. Calibration of the rotometers were made by measuring the time for a given weight of fluid flow. The mass flow rate was calculated and plotted versus the rotometer reading. Appendix K is the plot of the results. The plot was curve fitted with a third order polynomial. The following is that polynomial:

$$MH20 = -.0063268 + .002097278 * RR - .00000658 * RR^2 + 1.1X10 * RR^2$$

where $RR = \text{rotometer reading}$

D. DATA ACQUISITION/CONTROL SYSTEM

The heart of the data acquisition system is a HP9826 small computer. The HP9826 communicates via a Hewlett Packard 3497A data acquisition/control system. This system gathers data from the pressure transducers and nozzle thrust force-block. It converts the analog signal to digital data, and stores the data in memory. Figure 22 shows a pressure transducer to A/D converter channel connection, and Figure 23

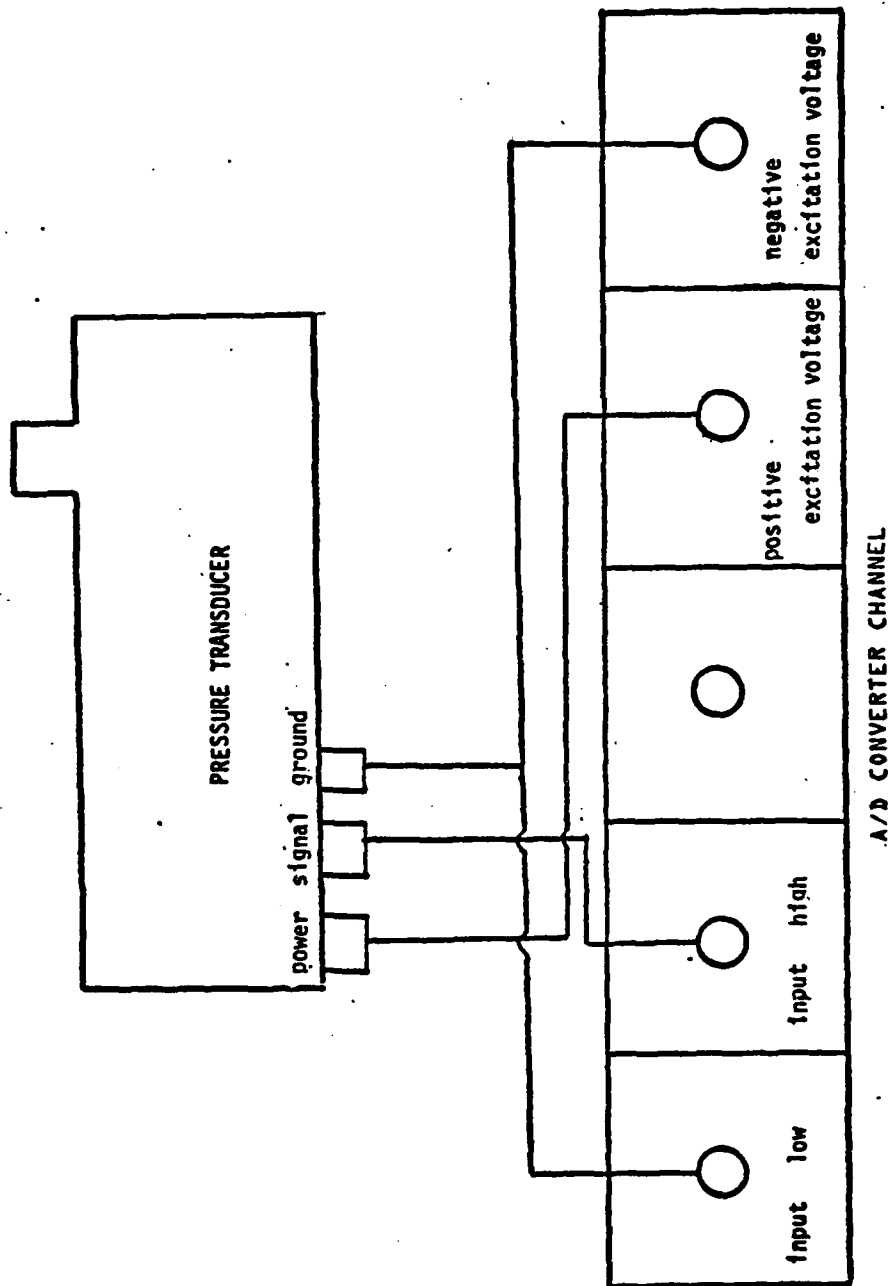


Figure 22. Transducer-to-Channel Connection

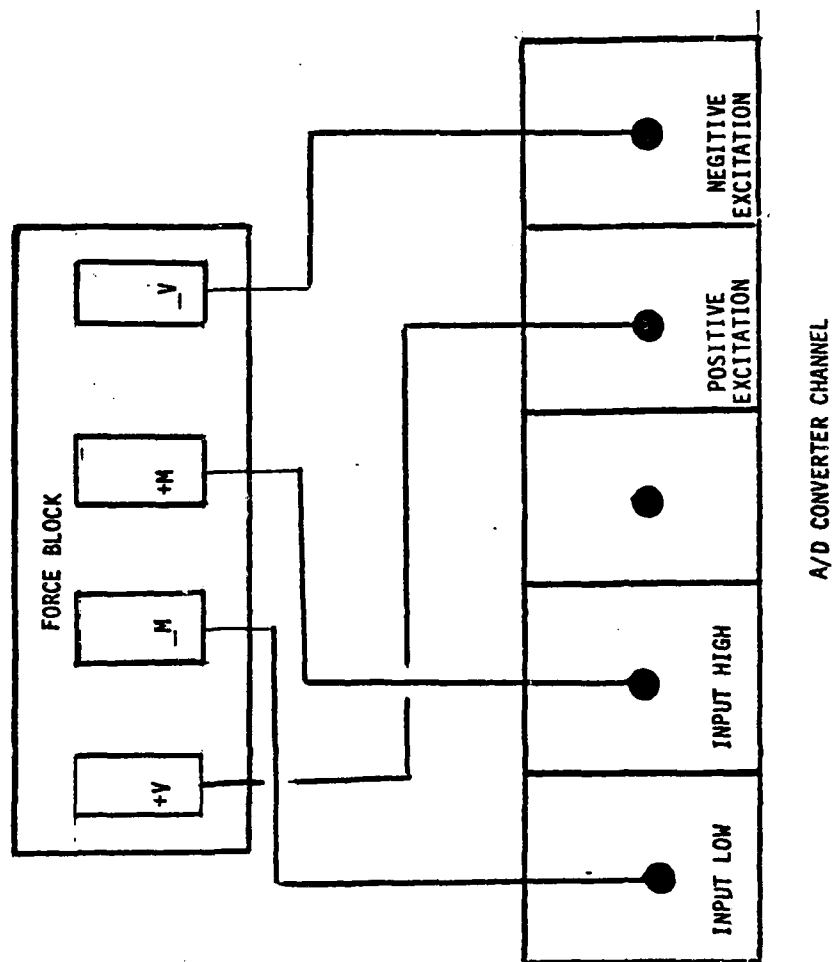


Figure 23. Force-Block-to-Channel Connection

CHANNEL 7	CHANNEL 8	CHANNEL 9	CHANNEL 16	CHANNEL 17	CHANNEL 18
6	H	I		EXCITATION VOLTAGE	
CHANNEL 4	CHANNEL 5	CHANNEL 6	CHANNEL 13	CHANNEL 14	CHANNEL 15
D	E	F	O	P	M,N
CHANNEL 1	CHANNEL 2	CHANNEL 3	CHANNEL 10	CHANNEL 11	CHANNEL 12
A	B	C	J	K	L
BUS CONNECTION TO A/D CONVERTER					

Figure 24. Bus Connection-to-A/D Converter Connections

the force-block to A/D converter channel connection. Figure 24 depicts the current connection to the data acquisition/control system bus from the sensors.

The data acquisition and analysis program is a fully automated system which integrates the use of the HP9826, HP3497A, and all sensor devices. The program can be divided into three sections:

1. Data Acquisition - Data is acquired from 15 different sensor devices. All system parameters are initialized and configuration types are inputted into the computer memory for future use. Ten readings are made on each sensor and a mean value for each is calculated and stored in memory.
2. Data Conversion - Input data conversion is completed by using calibration information obtained on each pressure transducer and force-block. The program takes the stored mean values from memory, and converts the input voltage to a pressure or force using the third order calibration polynomial for each sensor.
3. Data Analysis - This is the heart of the program in that all performance parameters are calculated and printed. It uses all of the above information to calculate mass flow rate of water, mass flow rate of air, total mass flow rate, thrust, mixture

ratio, exit velocity and all pressure informa-

tion. The following equations are also used:

$$\text{Exit Velocity} = (\text{Thrust} / \text{Total Mass Flow Rate}) * g_c (\text{ft/sec})$$

$$\text{Mixture Ratio} = \dot{m}_{\text{water}} / \dot{m}_{\text{air}}$$

$$\text{Total Mass Flow Rate} = \dot{m}_{\text{water}} + \dot{m}_{\text{air}} \left(\frac{\text{lbm}}{\text{sec}} \right)$$

Appendix L is the data acquisition program and sample output.

VI. EXPERIMENTAL RESULTS

Six test runs were made using the equipment described in the previous sections. It was found that the inlet pressure to the nozzle remained nearly constant for each setting. The testing was conducted by maintaining a constant air control valve setting and varying the mass flow rate of water. The regulator outlet pressure of the nitrogen bottle which controls the air valve was set at 30 psi and 15 psi. These settings correspond to nozzle inlet pressure of 35 ± 1.0 psi and 29 ± 1.0 psi respectively. For each nozzle inlet pressure, experimental data was obtained at varying mass ratios of water-to-air in the range of 2 to 13. These experiments were conducted with three discharge area ratios (A_{inlet}/A_{out}). Those ratios are: 3.586, 2.600, and 1.9259. The results of the six experimental tests are displayed in Tables II through VII.

Inlet air was limited to a maximum value of 40 psi. The 140PC pressure transducer's maximum output pressure is 40 psi, (see information on 140PC transducer Figure 21). The experimental system has the ability to reach higher pressures. The present configuration may be operated to 70 psig.

Table II. Experimental Data Exit Area = .45313 sq. in.
Inlet Pressure = 29+1.5psi

NOZZLE GEOMETRY 0.45313 SQ.IN*****PRESSURE 29.00000 PSI*****									

*****MASS FLOW RATE*****	*****	*****	*****	*****	*****	*****	*****	*****	*****
TOTAL	*****	*****	*****	*****	*****	*****	*****	*****	*****
0.06050	0.01830	0.04220	2.30000	1998.00000	1950.39900	3.75000	3.75000	3.75000	3.75000
0.10150	0.01840	0.08346	4.62000	1243.30000	1300.00000	3.92000	3.92000	3.92000	3.92000
0.13650	0.01790	0.11860	6.64000	969.69990	1001.89900	4.11000	4.11000	4.11000	4.11000
0.16630	0.01770	0.14860	8.39000	806.10000	830.00000	4.16000	4.16000	4.16000	4.16000
0.19220	0.01770	0.17450	9.83000	710.69990	760.30000	4.24000	4.24000	4.24000	4.24000
0.21490	0.01750	0.19730	11.26000	643.19990	695.50000	4.29000	4.29000	4.29000	4.29000
0.23560	0.01750	0.21810	12.39000	602.50000	642.00000	4.41000	4.41000	4.41000	4.41000
0.25550	0.01760	0.23790	13.51000	550.00000	620.89990	4.37000	4.37000	4.37000	4.37000

Table III. Experimental Data Exit Area = .45313 sq. in.
Inlet Pressure = 35±1.5psi

NOZZLE GEOMETRY		0.45313 SQ. IN.		PRESSURE		35.00000 PSI			

Table IV. Experimental Data Exit Area = .62500 sq. in.
Inlet Pressure = 29 \pm 1.5psi

MUZZLE GEOMETRY 0.62500 SQ.IN*****PRESSURE 29.00000 PSI*****									
*****MASS FLOW RATE*****				*****WATER RATIO*****		*****EXPERIMENTAL REAL EXIT VELOCITY*****		*****THRUST*****	
TOTAL	AIR					VELOCITY	VELOCITY		
0.06040	0.01820	0.04220	2.32000	1834.50000	1880.10000				3.44000
0.10150	0.01810	0.08346	4.61000	1159.00000	1201.19900				3.66000
0.13660	0.01800	0.11860	6.59000	937.80000	953.60000				3.98000
0.16650	0.01790	0.14860	8.32000	802.19990	803.10000				4.15000
0.19220	0.01770	0.17450	9.85000	688.60000	749.19990				4.21000
0.21510	0.01780	0.19730	11.10000	643.19990	702.10000				4.30000
0.23550	0.01740	0.21810	12.60000	589.89990	670.10000				4.32000

Table V. Experimental Data Exit Area = .62500 sq. in.
Inlet Pressure = 35±1.5psi

NOZZLE GEOMETRY 0.62500 SQ.IN*****PRESSURE 35.00000 PSI									

*****MASS FLOW RATE*****									
*****AIR*****									
*****WATER*****									
*****0.02900 0.04220*****									
*****MASS RATIO*****									
*****2.02000 2165.89900 2210.30000*****									
*****EXPERIMENTAL REAL EXIT VELOCITY*****									
*****THRUST*****									
*****4.24000*****									
0.13910	0.02050	0.11860	5.77000	1097.10000	1153.10000	4.74000			
0.16900	0.02050	0.14860	7.27000	940.50000	998.39990	4.94000			
0.19980	0.02030	0.17450	8.60000	832.89990	885.30000	5.04000			
0.21760	0.02030	0.19730	9.72000	765.39990	801.60000	5.17000			
0.23820	0.02010	0.21810	10.87000	720.50000	753.39990	5.32000			
0.25770	0.01890	0.23790	12.03000	674.80000	700.30000	5.40000			

Table VI. Experimental Data Exit Area = .84375 sq. in.
Inlet Pressure = 29±1.5psi

NOZZLE GEOMETRY		0.84375 SQ.IN		*****PRESSURE		29.00000 PSI		*****	
*****MASS FLOW RATE*****		*****		*****		*****		*****	
TOTAL	AIR	WATER	MASS RATIO	EXPERIMENTAL	REAL	EXPERIMENTAL	REAL	VELOCITY	THRUST
0.06110	0.01890	0.04220	2.23000	1586.69900	1653.39900				3.01000
0.10210	0.01860	0.08346	4.448003	1069.10000	1101.60000				3.39000
0.13700	0.01840	0.11860	6.44000	850.39990	900.89990				3.61000
0.16680	0.01820	0.14860	8.19000	754.69990	791.10000				3.91000
0.19250	0.01800	0.17450	9.69000	655.60000	720.10000				3.92000
0.21560	0.01820	0.19730	10.84000	596.80000	675.30000				3.99000
0.23610	0.01800	0.21810	12.14000	576.10000	638.19990				4.22000
0.25560	0.01770	0.23790	13.46000	570.50000	610.50000				4.52000

Table VII. Experimental Data Exit Area = .84375 sq. in.
Inlet Pressure = 35±1.5psi

NOZZLE GEOMETRY		0.84375 SQ. IN.		*****PRESSURE		35.00000 PSI	
*****		*****		*****		*****	
*****MASS FLOW RATE*****		*****		*****		*****	
TOTAL		AIR		WATER		*****	
*****		*****		*****		*****	
0.06350		0.02130		0.04220		*****	
*****		*****		*****		*****	
0.10440		0.02090		0.08346		*****	
*****		*****		*****		*****	
0.13900		0.02040		0.11860		*****	
*****		*****		*****		*****	
0.16890		0.02010		0.14860		*****	
*****		*****		*****		*****	
0.19460		0.02010		0.17450		*****	
*****		*****		*****		*****	
0.21690		0.01960		0.19730		*****	
*****		*****		*****		*****	
0.23810		0.02000		0.21810		*****	
*****		*****		*****		*****	
0.25780		0.01980		0.23790		*****	
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VII. DISCUSSION

The results of this investigation are in two parts; the experimental test results and the computer outputs based on initial conditions similar to those of the experiments.

The key variables are: the nozzle overall area ratio A_R , liquid/air mass flow ratio, nozzle exit velocity, nozzle supply pressure, nozzle thrust, and the nozzle axial pressure profiles. Figures 25 through 34 present the exit velocity vs. mass ratio results for the experiment and computer output. Figure 29-34 illustrates the comparison of the experimental results and the corresponding computer output. Figure 35 illustrates the variation of nozzle thrust with mass ratio for different area ratios and inlet pressures. Figures 36 and 37 present the axial pressure profiles.

In all cases the exit plane velocity decreases as the liquid/air mass ratio is increased. In the low mass ratio range (i.e., less than ≈ 5) the velocity decrease is very pronounced. Past a mass ratio value of ≈ 10 the velocity of the mixture becomes relatively insensitive to the mixture ratio. This is as would be expected. For the same inlet conditions there is a fixed amount of energy available for conversion to its kinetic form. In the nozzle process the energy is conserved and thus an increasing mass ratio is

manifested in a decreasing exit plane velocity. In all cases the agreement between the experimental tests and the analytic predictions are within $\approx 10\%$.

There also was apparent a consistent trend with respect to the nozzle area ratio and the exit velocity. As the nozzle exit area was increased the exit velocity of the mixture decreased. The effect seems to be more pronounced at the lower mass ratios. This trend was confirmed by observations of the actual flow field in the nozzle. If the exit area was increased significantly (beyond the max area used in those tests) there was evident an abrupt and severe separation within the diverging portion of the nozzle passage. This was accompanied by a drastic decrease in exit velocity as evidenced by the output from the thrust target. It appears that the diverging portion of the nozzle, at a certain point, starts to behave as a subsonic diffuser and hence experiences an adverse pressure gradient. This reasoning is in part confirmed by the pressure profiles identified in Figures 36 and 37.

The relationship between measured thrust and mass ratio (Figure 35) indicates a slightly increasing trend. This may be explained by considering the following. Each set of data points (at constant inlet air pressure and constant nozzle exit area) is developed by varying the mass ratio. This, in turn, is achieved by increasing the liquid rate by increasing the liquid supply pressure. The net result is an

increase in the liquid inlet velocity or an increase in the inlet energy level. Thus a particular data set is not truly at a constant inlet energy level but is increasing.

Perhaps the major discrepancy between the experimental test results and the computer analytic model is evidenced in the axial pressure profiles of Figures 36 and 37. It is clearly evident that at a certain point the nozzle passage reverts from a nozzle to a diffuser. This transition point occurs slightly downstream of the throat and the pressure starts to increase.

Unfortunately the computer analytic model requires a pressure profile as an input. Furthermore the pressure profile must be continuously decreasing. Thus the profile as obtained from the experiment are not directly useable. The situation is examined with the aid of Figure 38. Curve A is a typical axial pressure distribution as obtained from the experiment. Curve B is a pressure distribution used by Elliott in the application of his computer program. Pressure profiles C and D were arbitrarily defined and the exit plane velocity for each was calculated. Velocity variation in the range of 1% is evident. It appears that the final exit velocity is relatively insensitive to the actual pressure profile in the nozzle.

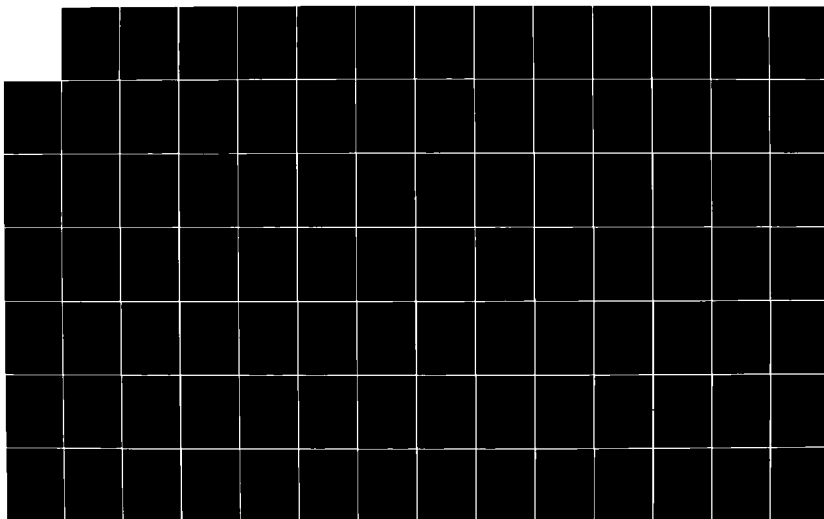
In light of the preceeding difficulty of matching the experimental pressure profile to the computer model and the

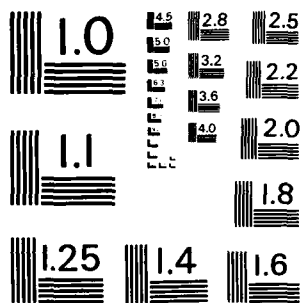
AD-A125 730 DUAL-PHASE NOZZLE FLOW(U) NAVAL POSTGRADUATE SCHOOL
MONTEREY CA T C MOLLIE OCT 82

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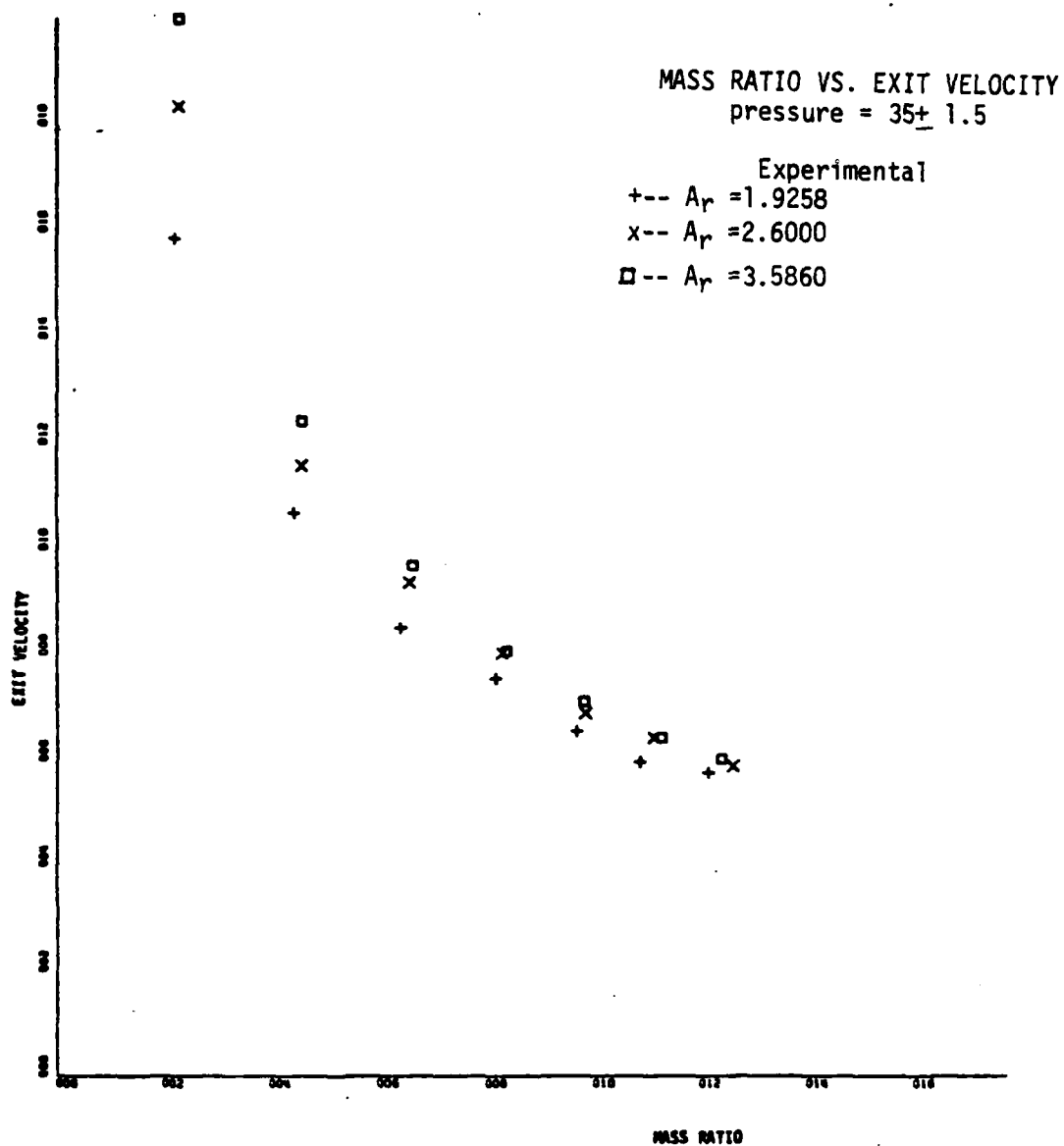




MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS - 1963 - A

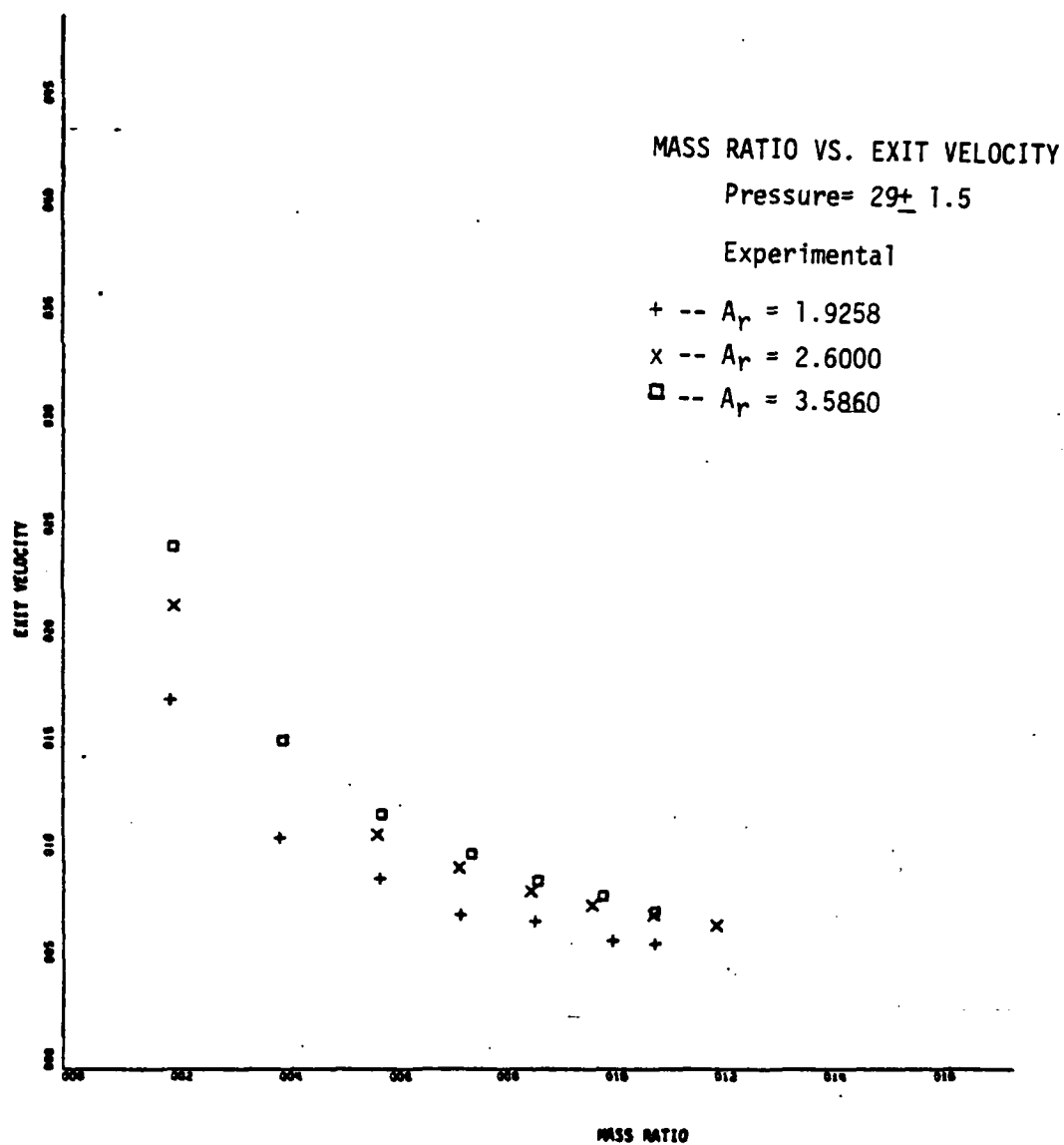
apparent flexibility of the type of pressure profile it was decided to employ a form of profile D of Figure 38. Thus instead of using the actual experimental pressure data as an input in the computer model, a profile resembling curve D was developed for each inlet pressure case.

It is apparent that within certain rather wide limits the Elliott computer model yields results that correspond within 10% to results obtained from the experiment. The general trends have been confirmed and their behavior has revealed nothing unexpected. The conclusion of Elliott [Ref. 1] has been largely substantiated. "It is very difficult to design a bad or a good dual-phase nozzle."



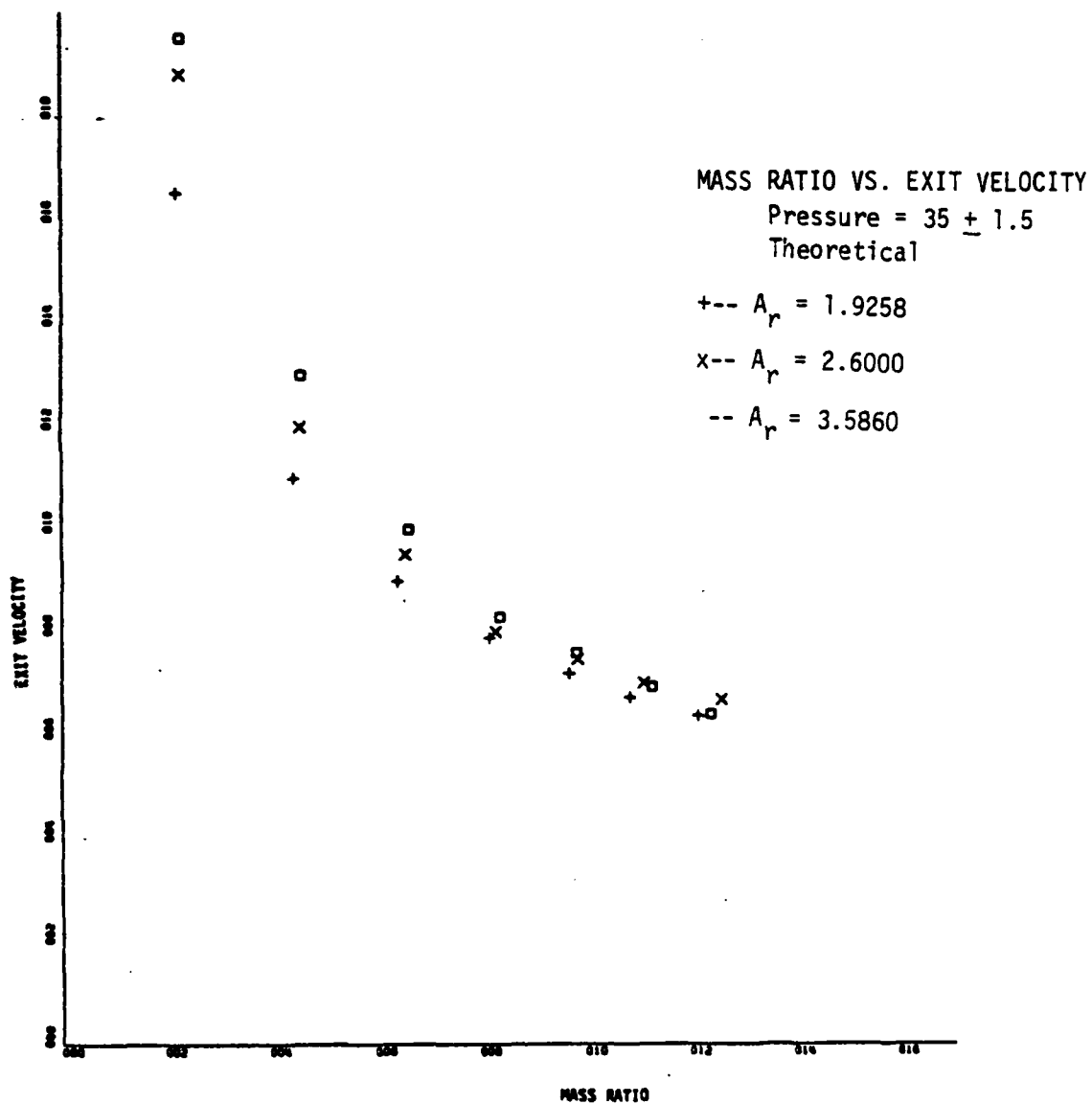
X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=2.00E+02 UNITS INCH.

Figure 25. Mass Ratio vs. Exit Velocity at
Pressure = 35 ± 1.5 psi Experimental



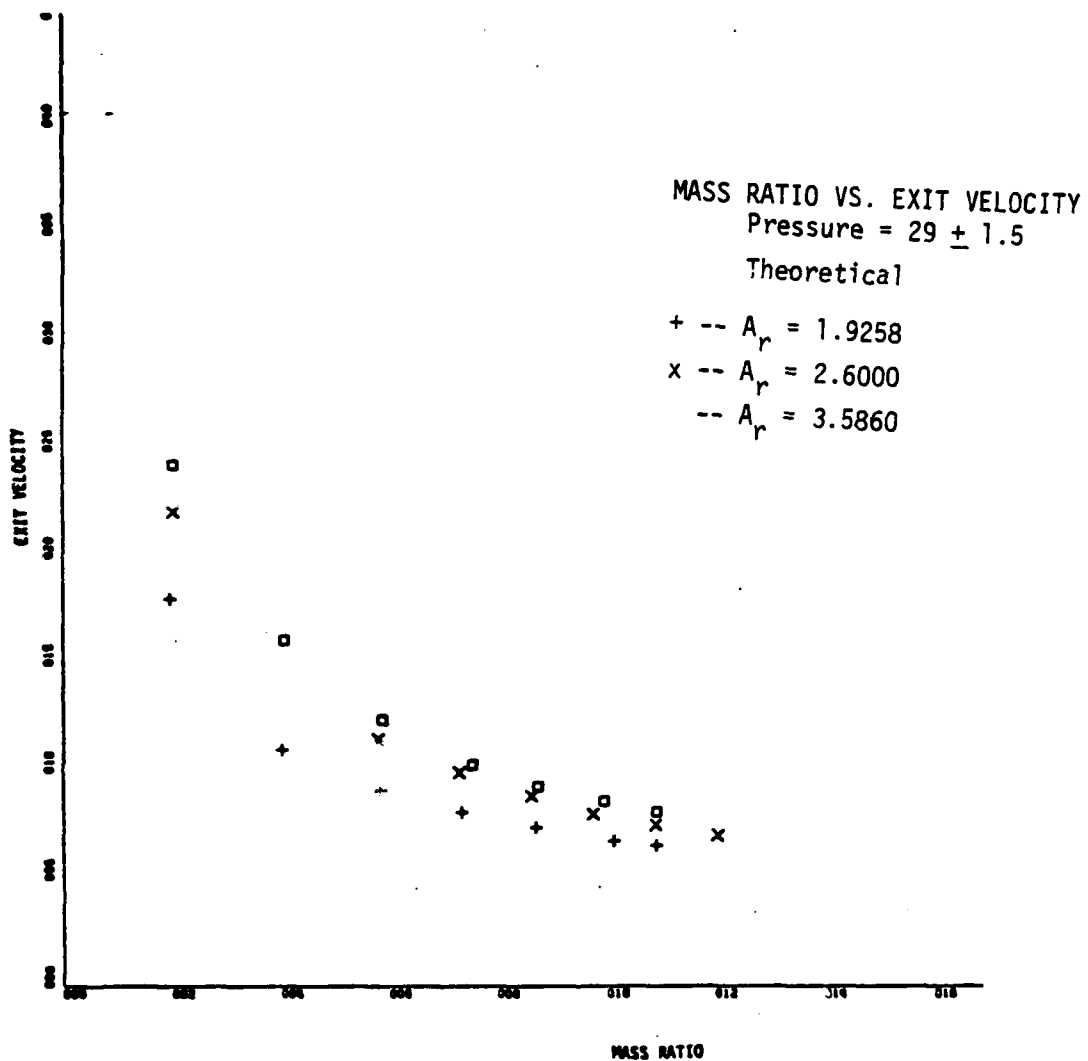
X-SCALE = 2.00×10^0 UNITS INCH.
Y-SCALE = 5.00×10^2 UNITS INCH.

Figure 26. Mass Ratio vs. Exit Velocity at
Pressure = 29 ± 1.5 psi Experimental



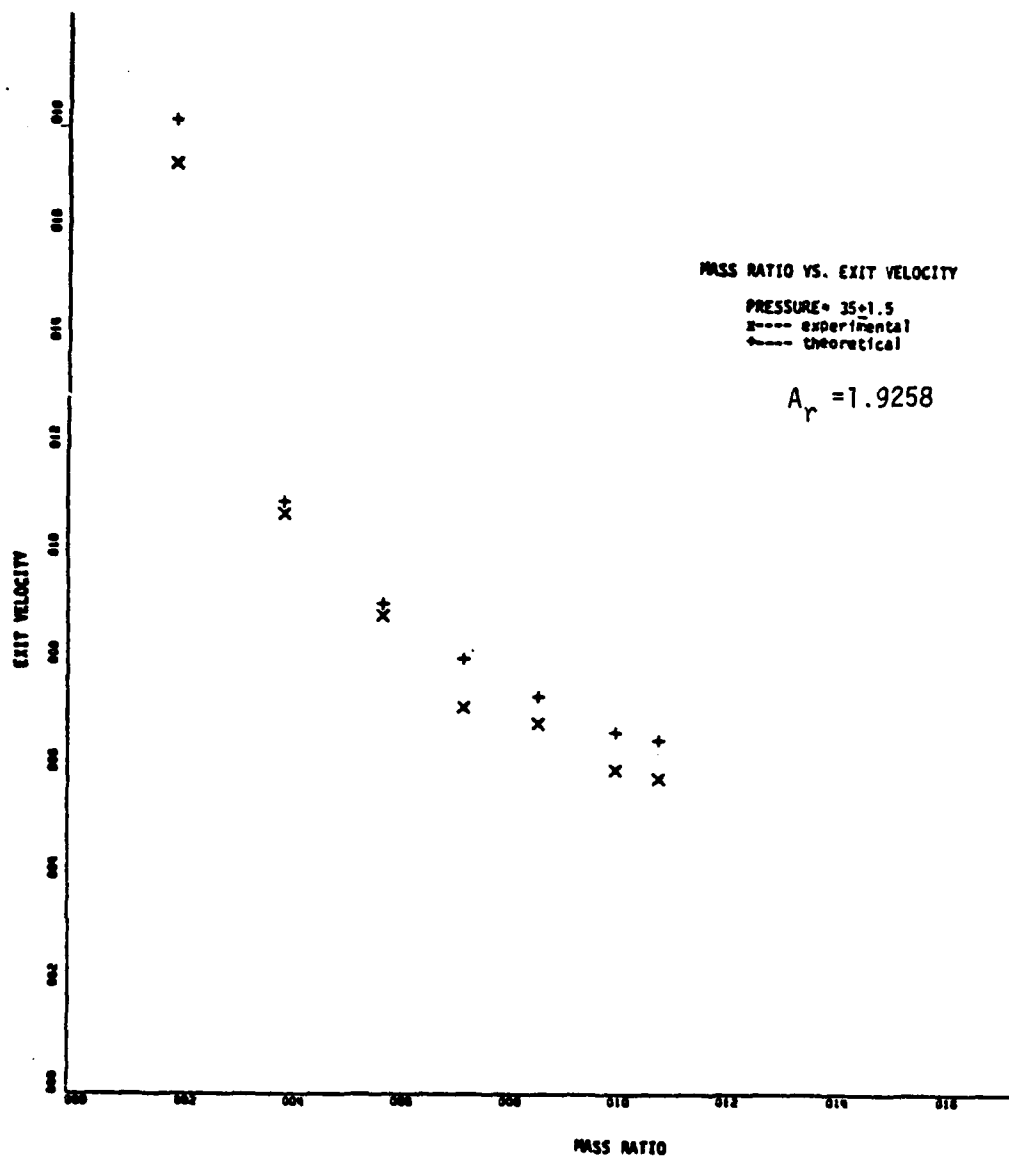
X-SCALE=2.00E+00 UNITS INCH.
 Y-SCALE=2.00E+02 UNITS INCH.

Figure 27. Mass Ratio vs. Exit Velocity at
 Pressure = 35 ± 1.5 psi Theoretical



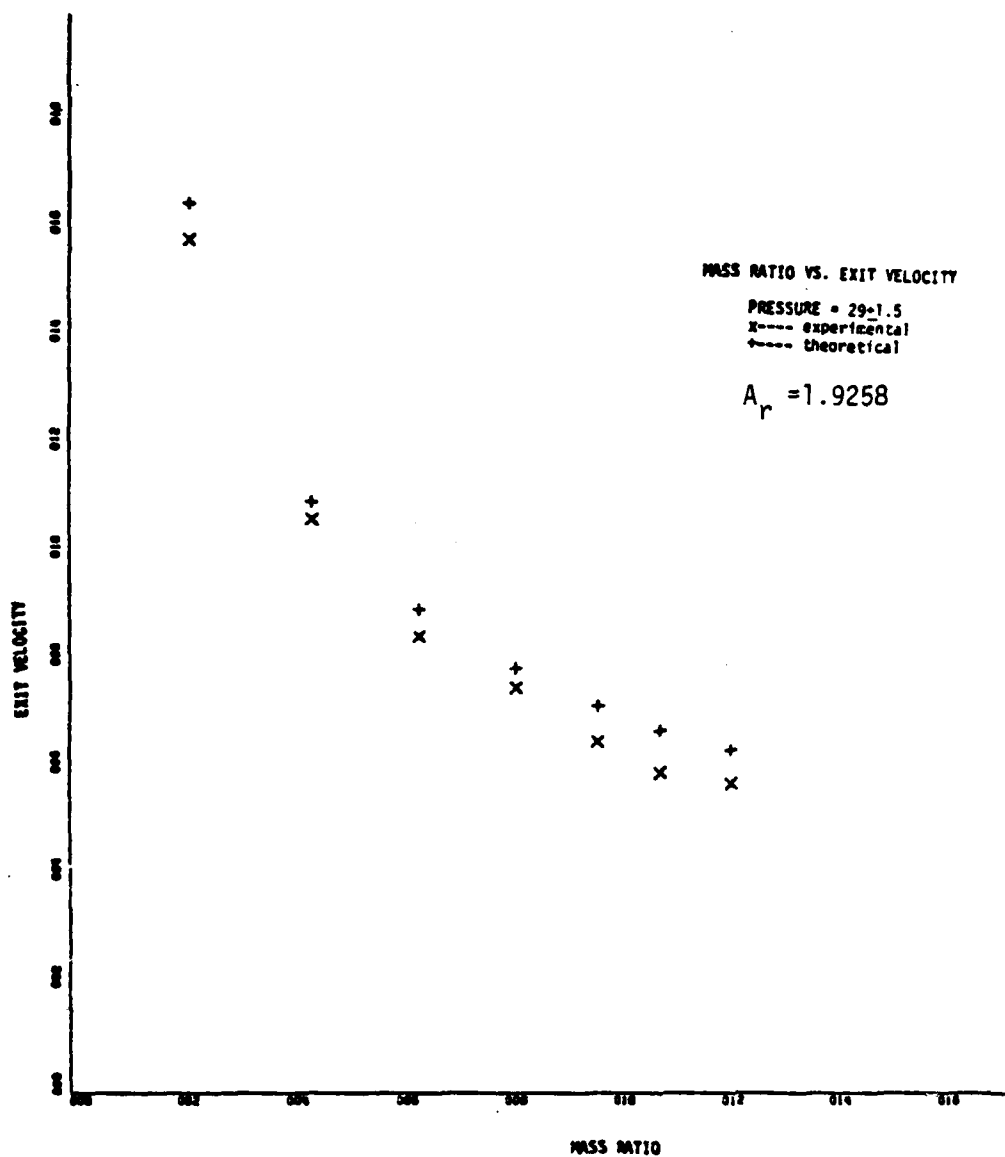
X-SCALE=2.00E+00 UNITS INCH.
 Y-SCALE=5.00E+02 UNITS INCH.

Figure 28. Mass Ratio vs. Exit Velocity at
 Pressure = 29 ± 1.5 psi Theoretical



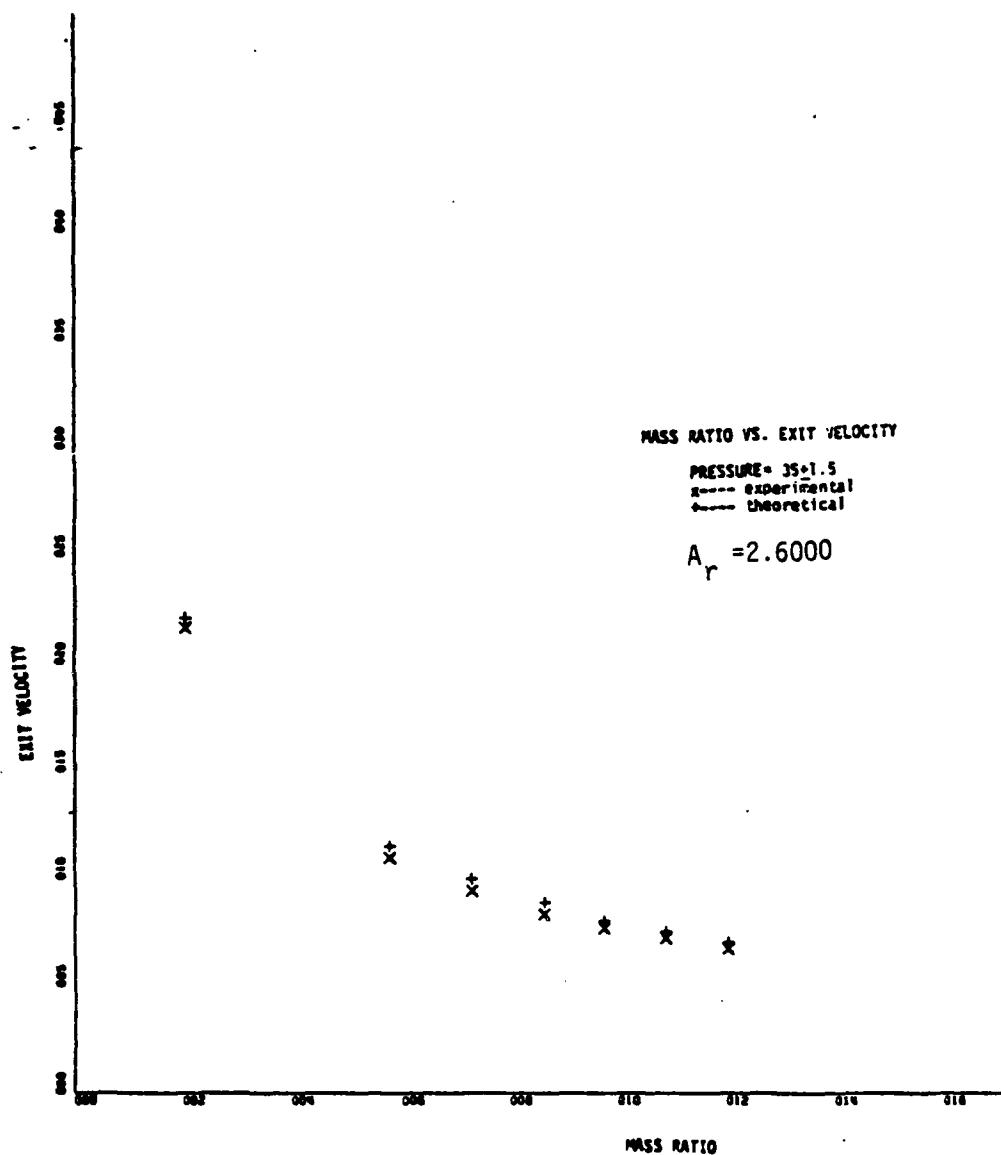
X-SCALE=2.00E+00 UNITS INCH.
 Y-SCALE=2.00E+02 UNITS INCH.

Figure 29. Mass Ratio vs. Exit Velocity at
 Pressure = 35±1.5psi Area = .34375



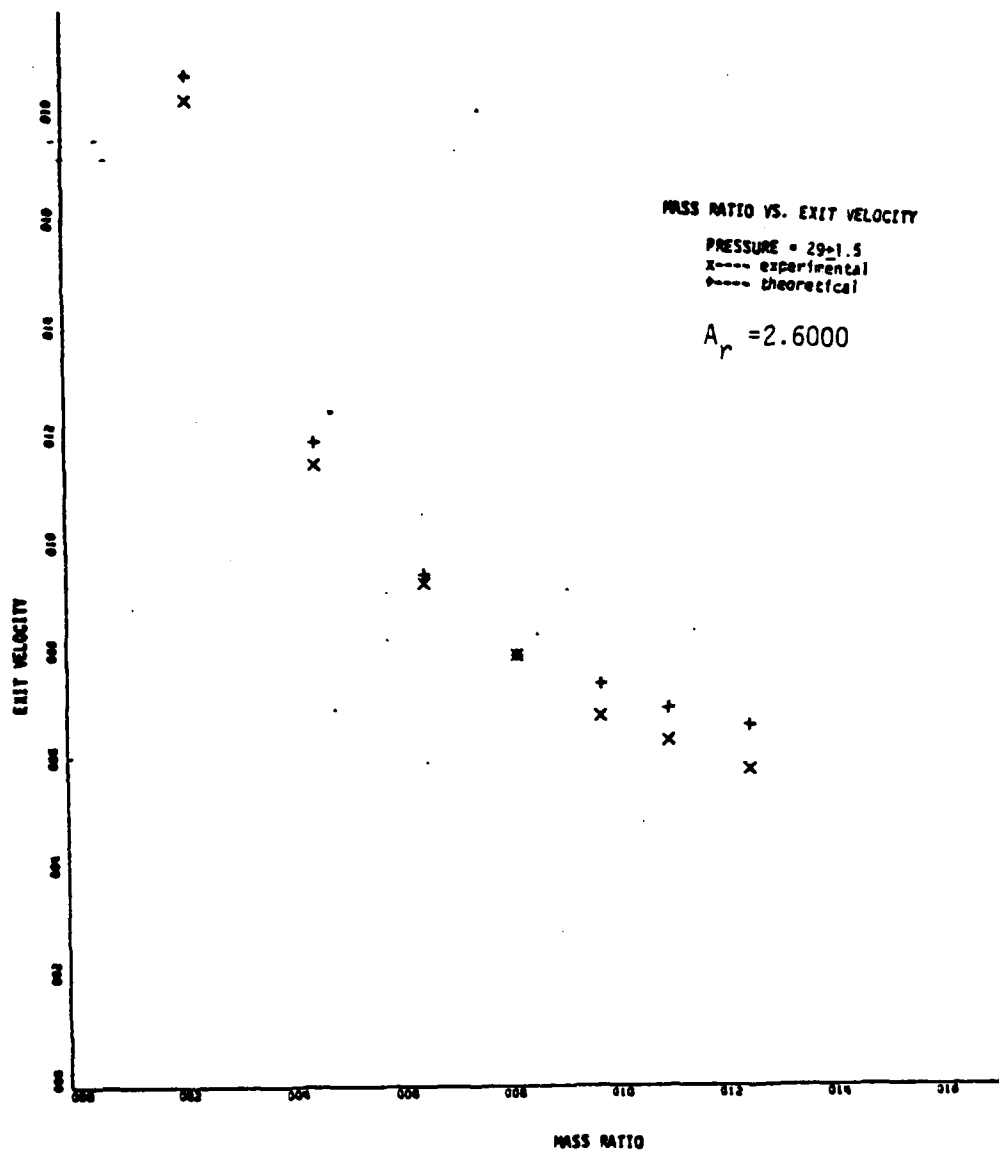
X-SCALE=2.00E+00 UNITS INCH.
 Y-SCALE=2.00E+02 UNITS INCH.

Figure 30. Mass Ratio vs. Exit Velocity at
 Pressure = 29±1.5psi Area = .84375



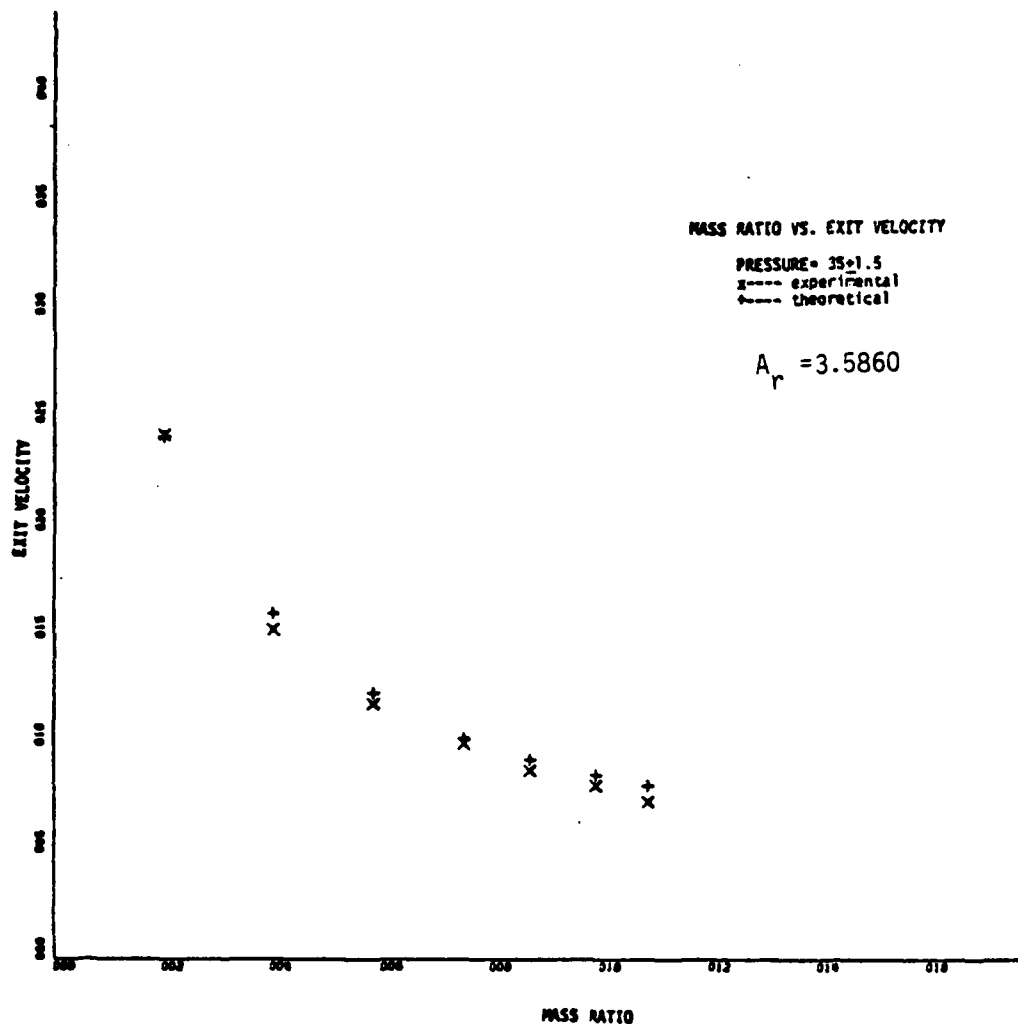
X-SCALE = $2.00E+00$ UNITS INCH.
Y-SCALE = $5.00E+02$ UNITS INCH.

Figure 31. Mass Ratio vs. Exit Velocity at
Pressure = 35 ± 1.5 psi Area = .62500



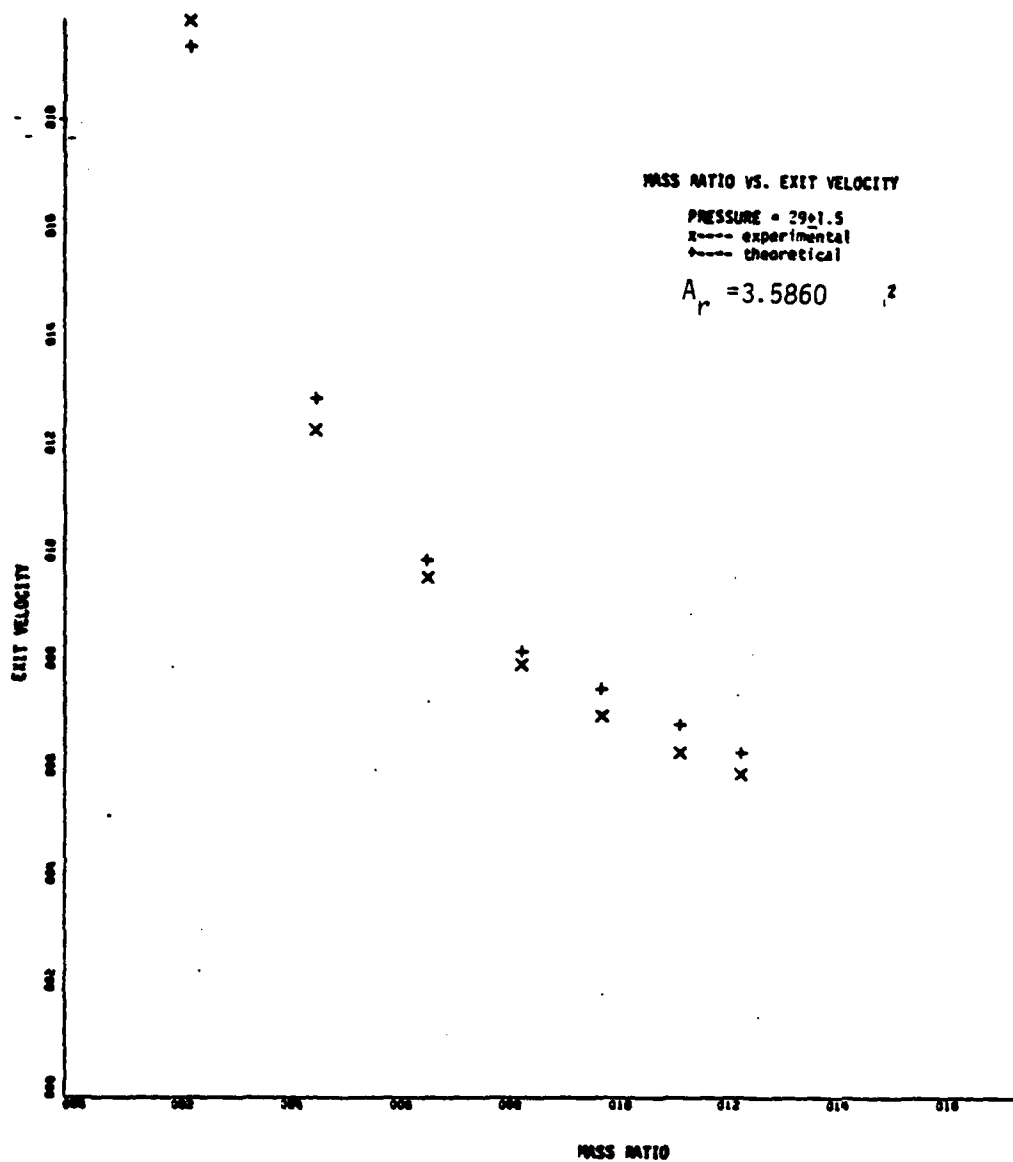
X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=2.00E+02 UNITS INCH.

Figure 32. Mass Ratio vs. Exit Velocity at
Pressure = 29±1.5psi Area = .62500



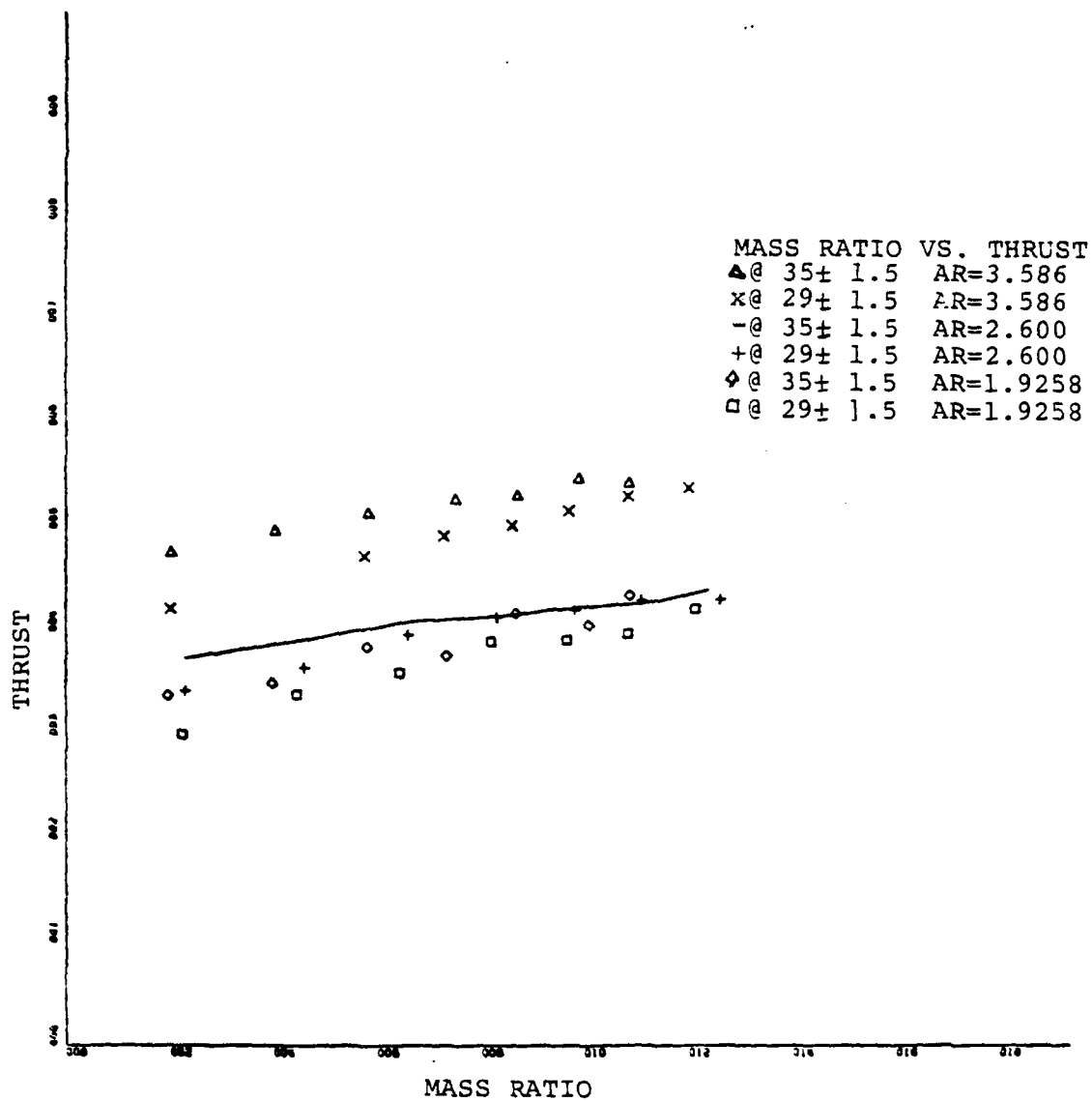
X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=5.00E+02 UNITS INCH.

Figure 33. Mass Ratio vs. Exit Velocity at
Pressure = 35±1.5psi Area = .45313



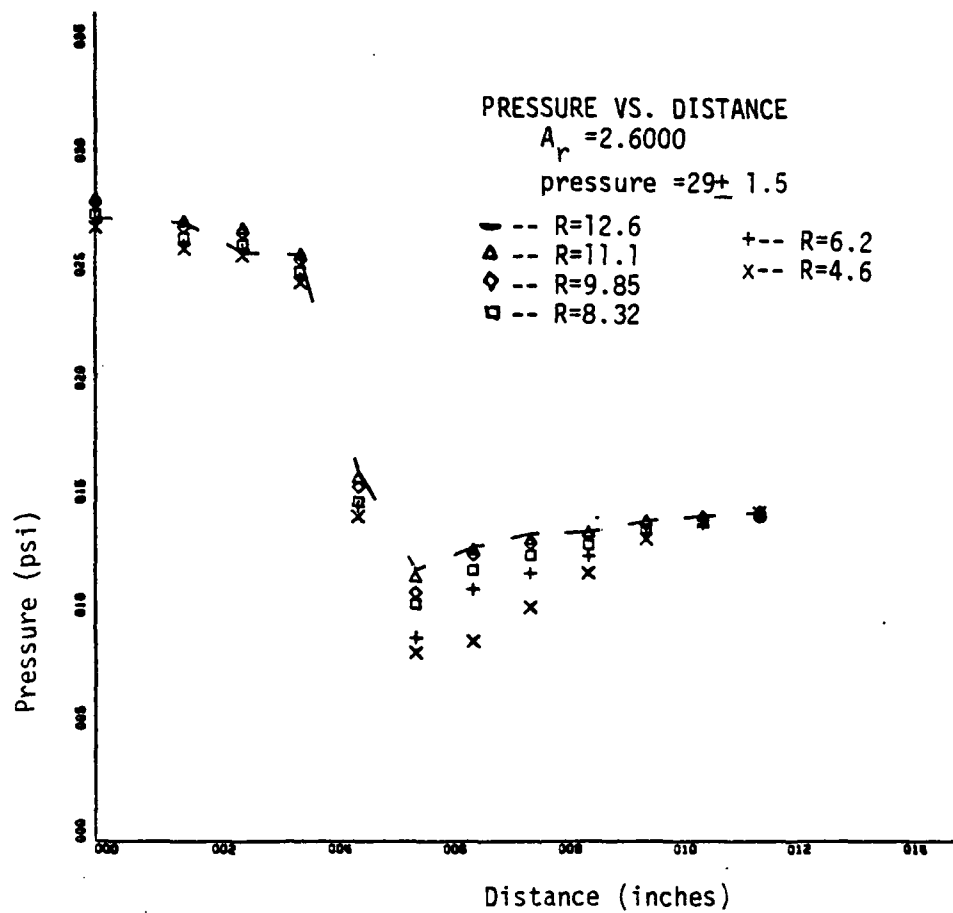
X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=2.00E+02 UNITS INCH.

Figure 34. Mass Ratio vs. Exit Velocity at
Pressure = 29±1.5psi Area = .45313



X-SCALE=2.00E+00 UNITS INCH.
Y-SCALE=1.00E+00 UNITS INCH.

Figure 35. Mass Ratio vs. Thrust Curve



X-SCALE=2.00E+00 UNITS INCH.
 Y-SCALE=5.00E+00 UNITS INCH.

Figure 36. Pressure vs. Distance at Pressure
 = 29 ± 1.5 Exit Area = .625 sq. in.

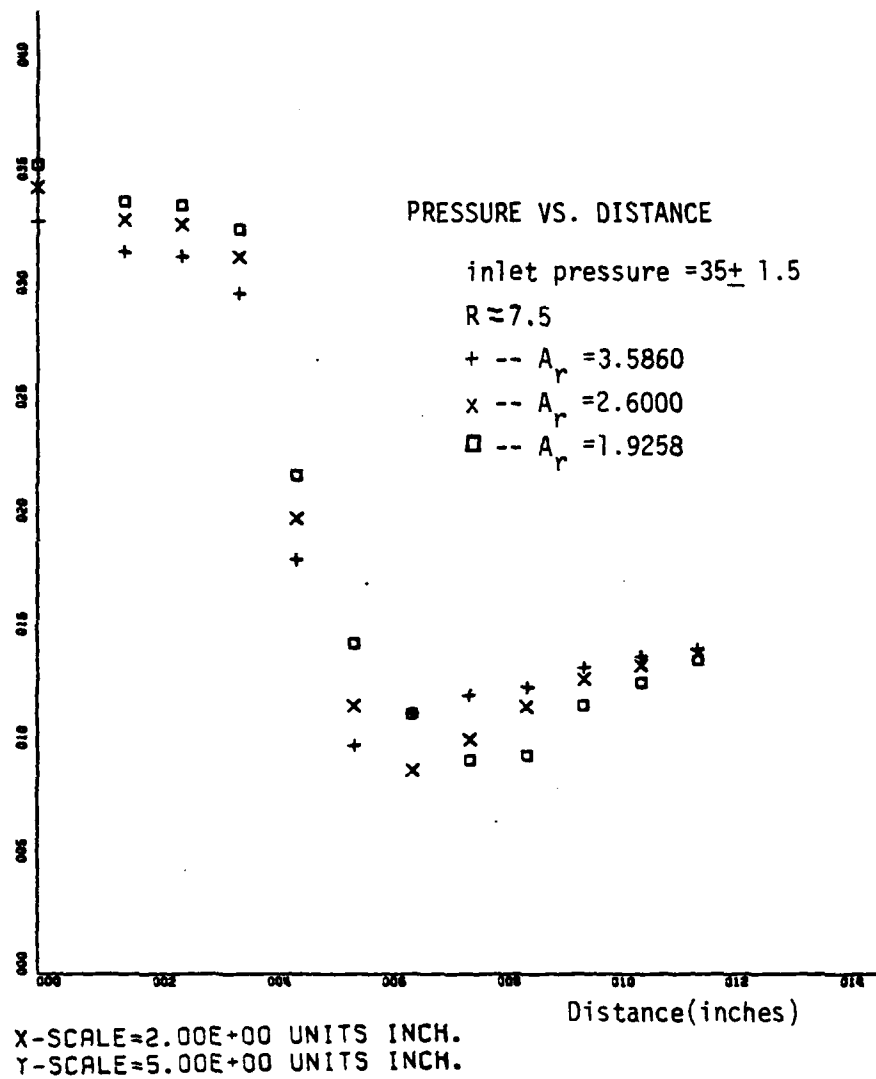


Figure 37. Pressure vs. Distance at
 $R \approx 7.5$ $p = 35\text{psi} \pm 1.5$

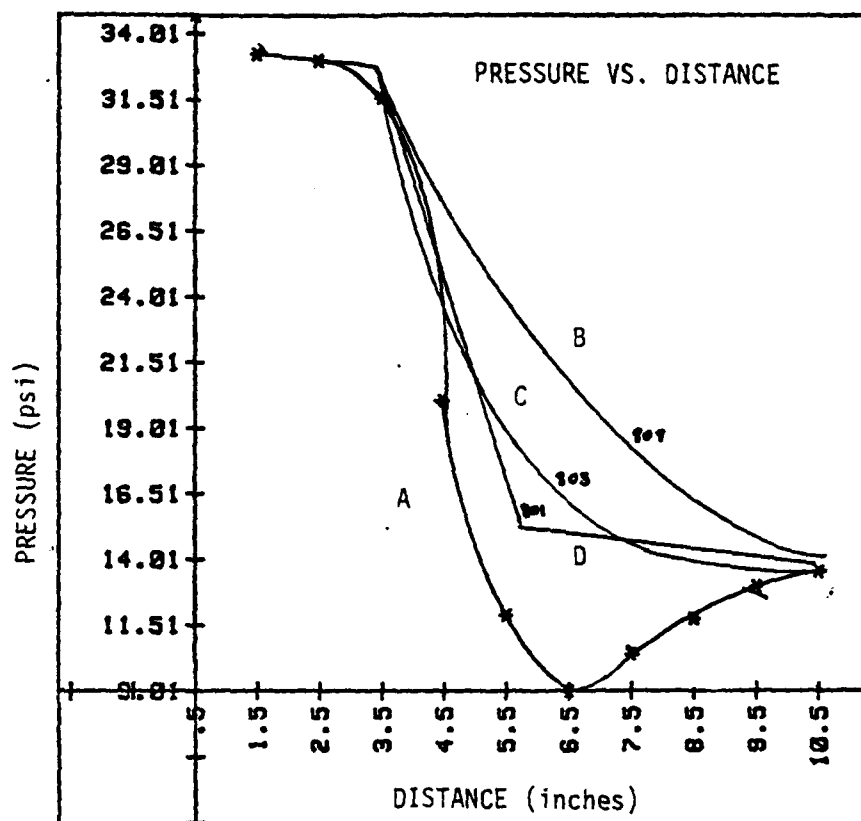


Figure 38. Various Pressure Profiles & Corresponding Exit Velocities

VIII. CONCLUSION

A set of experiments were conducted with an air-water mixture flowing at a mass ratio from 2 to 13. The experiments correlated well with a dual-phase two-component computer program. It appears that the program will permit prediction of the exit velocities to an accuracy of 10% for two-phase nozzles. At small nozzle area ratios, accuracies between theoretical and experimental are better; approximately 5%. All predicted velocities are higher than measured experimental values due to estimation of the drop size and the initial kinetic energy of the liquid at injection point.

It would be desirable to develop a drop size subroutine to better estimate varying drop size. The experimental system can be improved by velocity measurement devices at the inlet. Input of the gas and liquid velocities are necessary in the dual-phase two-component program and thus these measurement devices are vital for better accuracy.

Due to the insensitivity of the pressure distribution, it appears that any reasonable approximation to the pressure profile can be employed in the two-phase two-component flow program. For a given nozzle exit area and inlet pressure, a pressure distribution curve can be obtained through the experimental system. This distribution can therefore be used to obtain good results.

APPENDIX A: SAMPLE P(X) PROGRAM

```

$JOB
1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34
10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34
REAL P(75),X(75),PX,FX
INTEGER N,K,NP1,NX
A=35.2654923
B=5.6858877
C=14.39
D=0.04731
E=.444AC3
NX=75
PI=35.0
P2=8.0
PREF=8.0
XREF=10.0
N=74
NP1=N+1
PX=(P1-P2)/380
CC 10 K=1,N
X(K)=7.7*(K-1)/(N-1)
DX=7.7-X(K)
IF (X(K).LE.0.7) P(K)=P2+PX*E*(855-A*(X(K)+.5)**B)
IF (X(K).GT.0.7) P(K)=P2+PX*E*(C*DX+D*DX**4.9)
IF (X(K).EQ.1) GO TO 10
FRAC=(P(K-1)-PREF)/(P(K)-PREF)
IF (FRAC.GE.0.0.AND.FPAC.LE.1) XXREF=X(K-1)+FRAC*(X(K)-X(K-1))
10 CONTINUE
FX=XREF/XXREF
DC 15 K=1,N
X(K)=FX*X(K)
15 X(NP1)=1E5
P(NP1)=1E5
CC 20 K=1,75.3
WRITE(6,100) P(K),X(K),P(K+1),X(K+1),P(K+2),X(K+2)
20 STOP
100 FORMAT(6E12.6)
END

```

APPENDIX B: SAMPLE HEAT CAPACITY PROGRAM

```

1JCR  REAL PRESS(9), TABLE(20,20),P(20),TEMP(20),ANS(20),D1,02,03,04,05,
1      D6,07,08,09,M,K,X
1      INTEGER N,J,I,C,K,X
1      DATA PRESS(9),01,04,07,10,0,40,0,70,0/
1      READ (5,100)((TABLE(J,I),I=1,10),J=1,10)
1      DO 10 N=1,9
1      PRESS(N)=PRESS(N)+14.696
1      CC CONTINUE
1      DO 20 J=1,16
1      DO 30 I=2,10
1      CC TABLE(J,I)=TABLE(J,I)*.0685590
1      CC CONTINUE
1      CC CONTINUE
1      M=C,0
1      DO 51 J=1,16
1      TEMP(J)=400.0+C*M
1      M=M+10.0
1      W=C,0
1      DO 90 I=1,11
1      P(I)=W
1      N=1
1      Q=N+1
1      W=100.0+W
1      IF (ABS(TEMP(J)-TABLE(N,1)).GT .001) GO TO 25
1      Q=C
1      GC TC 50
1      CONTINUE
1      IF ((TEMP(J) .GT. TABLE(N,1)) .AND. (TEMP(J) .LT. TABLE(Q,1)))
1      CONTINUE
1      N=1+N
1      Q=N+1
1      GC TC 15
1      CONTINUE
1      K=2
1      X=K+1
1      IF(P(1).GT. .0001)GO TO 49
1      X=C
1      GC TC 60

```

```

45 IF(ABS(P(I) - PRESS(K)).GT..001) GO TC 55
   X=C
   GC TC 60
55 CONTINUE
   IF (P(I) .GT. PRESS(K)) .AND. (P(I) .LT. PRESS(X)) GC TC 60
   K=I+K
   X=X+I
   GC TC 55
60 CONTINUE
   IF (C.NE. Q) .OR. (X .NE. 0))GO TC 61
   ANS(I)=TABLE (N,K)
   GC TC 90
61 CONTINUE
   IF (C .NE. C) GO TO 63
   D1 = TABLE(N,K)-TABLE(N,X)
   D7 = PRESS(K-1) -PRESS(X-1)
   D8 = PRESS(K-1)-P(I)
   D9 =(D8/D7)*D1
   ANS(I)=C+TABLE(N,K)
   GC TO 90
62 CONTINUE
   IF (X.NE. C) GO TO 70
   D1 = (TABLE(N,K)-TABLE(Q,K))
   D3 = TABLE(N,1)-TABLE(Q,1)
   D4 = TABLE(N,1) -TEMP(J)
   D5 =(D4/D3)*D1
   ANS(I) =C5+(TABLE(N,K))
   GC TC 90
70 CONTINUE
   D1 =TABLE (N,K)-TABLE(Q,K)
   D2=TABLE(N,X)-TABLE(Q,X)
   D3=TABLE(N,1)-TABLE(Q,1)
   D4=TABLE(N,1)-TEMP(J)

```


[illegible]

```

55      CONTINUE
      IF ((P(I) .GT. PRESS(K)) .AND. (F(I) .LT. PRESS(X))) GO TO 60
      K=I+K
      X=X+1
      GC TC 55
60      CONTINUE
      IF ((C.NE. 0) .OR. (X .NE. 0)) GO TO 61
      AN=(I)=TABLE(N,K)
      GC TC 50
61      CONTINUE
      IF ((C.NE. 0) GO TC 63
      D1 = (TABLE(N,K)-TABLE(N,X))
      D2 = (PRESS(X-1) -PRESS(X-1))
      D3 = PRESS(X-1)-P(I)
      D4 = ABS((C2/D3)*D1)
      AN(I)=D4+TABLE(N,K)
      GC TC 50
62      CONTINUE
      IF ((C.NE. 0) GO TO 70
      D1 = (TABLE(N,K)-TABLE(C,K))
      D2 = (TABLE(N,1)-TABLE(C,1))
      D3 = ABS((C2/D3)*D1)
      AN(I)=D3+TABLE(N,K)-D9
      GC TC 50
70      CONTINUE
      D1 = TABLE(N,K)-TABLE(C,K)
      D2 = TABLE(N,X)-TABLE(C,X)
      D3 = TABLE(N,1)-TABLE(C,1)
      D4 = TABLE(N,1)-TEMP(J)

```


APPENDIX D: SAMPLE PROPERTY TABLE

SENTPT							
400.0	.2494	100.0	.2534	200.0	.2571	CAG	0010
0.0	.2626	400.0	.2578	500.0	.2730	CAG	0020
300.0	.2779	700.0	.2642	800.0	.2767	CAG	0030
600.0	.3001	1000.0	.3053		1.ES	1.25CAG	0050
900.0						CAG	0060
410.0	.2488	100.0	.2532	200.0	.2569	CAG	0070
0.0	.2621	400.0	.2575	500.0	.2721	CAG	0080
300.0	.2773	700.0	.2643	800.0	.2718	CAG	0090
600.0	.2996	1000.0	.3030		1.ES	1.ES CAG	0100
900.0						CAG	0110
420.0	.2484	100.0	.2531	200.0	.2566	CAG	0120
0.0	.2616	400.0	.2575	500.0	.2712	CAG	0130
300.0	.2769	700.0	.2615	800.0	.2704	CAG	0140
600.0	.2990	1000.0	.3000		1.ES	1.ES CAG	0150
900.0						CAG	0160
430.0	.2484	100.0	.2529	200.0	.2564	CAG	0170
0.0	.2612	400.0	.2574	500.0	.2704	CAG	0180
300.0	.2768	700.0	.2611	800.0	.2709	CAG	0190
600.0	.2924	1000.0	.2970		1.ES	1.ES CAG	0200
900.0						CAG	0210
440.0	.2484	100.0	.2527	200.0	.2561	CAG	0220
0.0	.2607	400.0	.2551	500.0	.2695	CAG	0230
300.0	.2737	700.0	.2707	800.0	.2647	CAG	0240
600.0	.2898	1000.0	.2942		1.ES	1.ES CAG	0250
900.0						CAG	0260
450.0	.2484	100.0	.2526	200.0	.2559	CAG	0270
0.0	.2602	400.0	.2545	500.0	.2636	CAG	0280
300.0	.2726	700.0	.2718	800.0	.2625	CAG	0290
600.0	.2872	1000.0	.2915		1.ES	1.ES CAG	0300
900.0						CAG	0310
460.0	.2484	100.0	.2524	200.0	.2556	CAG	0320
0.0	.2597	400.0	.2618	500.0	.2678	CAG	0330
300.0	.2716	700.0	.2759	800.0	.2604	CAG	0340
600.0							

[illegible]

307.0	28.508	400.0	28.710	500.0	28.942	WA	1070
600.0	29.055	700.0	29.242	400.0	29.331	WA	1080
900.0	29.460	1000.0	29.500		1.25	1.25	1090
0.0							1100
0.0	28.010	100.0	28.175	200.0	28.310	WA	1110
300.0	28.472	400.0	28.553	500.0	28.851	WA	1120
600.0	28.958	700.0	29.127	400.0	29.200	WA	1130
900.0	29.312	1000.0	29.425		1.25	1.25	1140
0.0							1150
0.0	28.010	100.0	28.151	200.0	28.294	WA	1160
300.0	28.436	400.0	28.525	500.0	28.765	WA	1170
600.0	28.861	700.0	29.012	400.0	29.073	WA	1180
900.0	29.165	1000.0	29.255		1.25	1.25	1190
0.0							1200
0.0	28.010	100.0	28.151	200.0	28.266	WA	1210
300.0	28.403	400.0	28.543	500.0	28.880	WA	1220
600.0	28.768	700.0	28.932	400.0	29.040	WA	1230
900.0	29.020	1000.0	29.310		1.25	1.25	1240
0.0							1250
0.0	28.010	100.0	28.140	200.0	28.244	WA	1260
300.0	28.383	400.0	28.482	500.0	28.745	WA	1270
600.0	28.677	700.0	28.750	400.0	28.917	WA	1280
900.0	28.899	1000.0	29.140		1.25	1.25	1290
0.0							1300
0.0	28.010	100.0	28.123	200.0	28.222	WA	1310
300.0	28.127	400.0	28.422	500.0	28.517	WA	1320
600.0	28.547	700.0	28.748	400.0	28.700	WA	1330
900.0	28.750	1000.0	28.730		1.25	1.25	1340
0.0							1350
0.0	28.010	100.0	28.116	200.0	28.210	WA	1360
300.0	28.310	400.0	28.417	500.0	28.700	WA	1370
600.0	28.510	700.0	28.700	400.0	28.850	WA	1380
900.0	28.620	1000.0	28.850		1.25	1.25	1390
0.0							1400
0.0	28.010	100.0	28.104	200.0	28.190	WA	1410
300.0	28.300	400.0	28.400	500.0	28.690	WA	1420
600.0	28.500	700.0	28.690	400.0	28.800	WA	1430
900.0	28.600	1000.0	28.800		1.25	1.25	1440
0.0							1450
0.0	28.010	100.0	28.098	200.0	28.180	WA	1460
300.0	28.290	400.0	28.380	500.0	28.670	WA	1470
600.0	28.490	700.0	28.670	400.0	28.780	WA	1480
900.0	28.590	1000.0	28.780		1.25	1.25	1490
0.0							1500
0.0	28.010	100.0	28.086	200.0	28.160	WA	1510
300.0	28.270	400.0	28.360	500.0	28.650	WA	1520
600.0	28.470	700.0	28.640	400.0	28.760	WA	1530
900.0	28.570	1000.0	28.760		1.25	1.25	1540
0.0							1550
0.0	28.010	100.0	28.074	200.0	28.140	WA	1560
300.0	28.250	400.0	28.340	500.0	28.630	WA	1570
600.0	28.450	700.0	28.620	400.0	28.740	WA	1580
900.0	28.550	1000.0	28.740		1.25	1.25	1590
0.0							1600
0.0	28.010	100.0	28.060	200.0	28.120	WA	1610
300.0	28.230	400.0	28.320	500.0	28.610	WA	1620
600.0	28.430	700.0	28.600	400.0	28.720	WA	1630
900.0	28.530	1000.0	28.720		1.25	1.25	1640
0.0							1650
0.0	28.010	100.0	28.040	200.0	28.100	WA	1660
300.0	28.210	400.0	28.300	500.0	28.590	WA	1670
600.0	28.410	700.0	28.580	400.0	28.690	WA	1680
900.0	28.510	1000.0	28.690		1.25	1.25	1690
0.0							1700
0.0	28.010	100.0	28.020	200.0	28.080	WA	1710
300.0	28.190	400.0	28.280	500.0	28.570	WA	1720
600.0	28.390	700.0	28.560	400.0	28.680	WA	1730
900.0	28.490	1000.0	28.680		1.25	1.25	1740
0.0							1750
0.0	28.010	100.0	28.000	200.0	28.040	WA	1760
300.0	28.170	400.0	28.260	500.0	28.530	WA	1770
600.0	28.370	700.0	28.540	400.0	28.650	WA	1780
900.0	28.470	1000.0	28.650		1.25	1.25	1790

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APPENDIX E: SAMPLE INPUT DATA

APRIL 15, 1980		NOZZLE D, 2 MPA		920		1	
1	0	30	30	30	30	30	30
-0.3	20.0	0.0	0.0	0.0	0.0	5.28	
1010.0	85E-60.0	-0.3	28.02	18.02	208.0		
20.0	0.05	290.0	537.0	537.0	20.0		
10	0.005	0.005	0.0004	0.003	0.1		
IN. NCZLL F							
2500000	000000	289564+C3	138144+00	288590+03	276288+00		
284616+03	414332+C0	283025+03	552576+00	276898+03	690720+00		
266595+03	828864+C0	252206+03	967008+00	236879+03	110515+01		
222415+03	124330+C0	209777+C3	138144+01	145929+03	151958+01		
183834+03	165773+C0	172458+03	179537+01	161766+C3	193402+01		
151728+03	207216+C0	142310+03	221030+01	133482+03	224845+01		
125216+03	249655+C0	117431+03	262474+C0	110251+C3	278289+01		
103500+03	290102+01	972006+02	303917+01	913291+02	317771+01		
858612+02	321546+C0	807742+C2	345360+01	760066+02	359174+01		
716545+02	372339+01	675910+02	396803+01	638066+02	400618+01		
603065+02	414432+C0	570700+02	423246+01	540765+02	442061+01		
513102+02	445875+01	467553+C2	469690+01	463268+02	482504+01		
442206+02	497315+C0	422129+02	511133+01	403511+02	524947+C0		
380527+02	533762+C0	370764+02	552576+01	356211+02	565351+01		
342765+02	560205+C0	339228+02	594019+01	318810+02	597821+01		
308124+02	621648+01	298120+02	635463+01	264540+02	649777+01		
280228+C0	663091+C0	272169+02	676906+01	244000+02	690720+01		
257235+02	705535+C0	250505+02	713246+01	226035+02	732123+01		
237775+02	745078+01	231833+02	759792+01	206684+02	773607+01		
220445+02	787421+C0	215603+02	801235+01	204257+C2	815050+01		
204466+02	828864+C0	193329+02	842679+01	179296+02	856493+01		
185235+02	870307+C0	184252+02	884122+01	164525+C2	897936+01		
174360+02	911751+C0	169438+02	925565+01	149807+C2	939372+01		
159617+02	953194+01	154712+02	967008+01	140000+02	980823+01		
144504+02	994637+01	140000+02	100845+02	100000+06	100000+06		

APPENDIX F: SAMPLE 140PC CALIBRATION PROGRAM

```

10 DIM X(15,100),Sum(15),Mean(15),Sum2(15),Sd(15)
30 N=10
40 FOR I=1 TO N
50   OUTPUT 709;"AI1VT1"
60   ENTER 709;X(1,I)
70   OUTPUT 709;"AI2VT1"
80   ENTER 709;X(2,I)
90   OUTPUT 709;"AI3VT1"
100  ENTER 709;X(3,I)
110  OUTPUT 709;"AI4VT1"
120  ENTER 709;X(4,I)
130  OUTPUT 709;"AI5VT1"
140  ENTER 709;X(5,I)
150  OUTPUT 709;"AI6VT1"
160  ENTER 709;X(6,I)
170  OUTPUT 709;"AI7VT1"
180  ENTER 709;X(7,I)
190  OUTPUT 709;"AI8VT1"
200  ENTER 709;X(8,I)
210  OUTPUT 709;"AI9VT1"
220  ENTER 709;X(9,I)
230  OUTPUT 709;"AI16VT1"
240  ENTER 709;X(10,I)
250  OUTPUT 709;"AI11VT1"
260  ENTER 709;X(11,I)
261  OUTPUT 709;"AI12VT1"
262  ENTER 709;X(12,I)
263  OUTPUT 709;"AI13VT1"
264  ENTER 709;X(13,I)
267  OUTPUT 709;"AI17VT1"
268  ENTER 709;X(14,I)
270 NEXT I

```

```

280 PRINT "
290 PRINT "
300 I
310 PRINT "
320 PRINT
330 FOR I=1 TO N
340 PRINT USING "DD.DDDD";X(1,I),X(2,I),X(3,I),X(4,I),X(5,I),X(6,I),X(7,I),
X(8,I)
350 NEXT I
360 PRINT
370 PRINT "
380 PRINT "
390 PRINT "
400 FOR I=1 TO N
410 PRINT USING "DD.DDDD";X(9,I),X(10,I),X(11,I),X(12,I),X(13,I),X(14,I)
411 NEXT I
413 PRINT
414 PRINT "
415 PRINT "
420 PRINT
440 FOR J=1 TO 14
450 Sum(J)=0
460 FOR I=1 TO N
470 Sum(J)=Sum(J)+X(J,I)
480 NEXT I
490 Mean(J)=Sum(J)/N
510 NEXT J
512 FOR J=1 TO 14
513 FOR I=1 TO N
515 Sum2(J)=Sum2(J)+X(J,I)-Mean(J))*(X(J,I)-Mean(J))
516 NEXT I
518 Sd(J)=SQR(Sum2(J)/(N-1))
519 IF J=14 THEN GOTO 523
521 PRINT "channel
522 GOTO 525
523 PRINT "CHANNEL
525 NEXT J
526 END

```

CHANNELS							
1	2	3	4	5	6	7	8
<div style="display: flex; justify-content: space-between;"> <div> <p>CHANNEL</p> <p>9 10 11 12 13 17</p> </div> <div> <p>MEAN " "</p> <p>STANDARD DEVIATION "</p> </div> </div>							

APPENDIX C: SAMPLE OUTPUT TO CALIBRATION PROGRAM

THIS DATA IS FOR A PRESSURE OF 0 PSI

CHANNELS							
1	2	3	4	5	6	7	8
1.6588	1.8301	1.7702	1.6914	1.8429	1.7485	1.7900	1.8098
1.6589	1.8305	1.7705	1.6922	1.8431	1.7484	1.7901	1.8101
1.6598	1.8308	1.7715	1.6931	1.8446	1.7500	1.7918	1.8120
1.6605	1.8318	1.7717	1.6935	1.8444	1.7499	1.7916	1.8117
1.6603	1.8317	1.7722	1.6931	1.8446	1.7497	1.7917	1.8117
1.6598	1.8315	1.7716	1.6928	1.8439	1.7495	1.7909	1.8112
1.6596	1.8310	1.7712	1.6925	1.8437	1.7494	1.7909	1.8110
1.6597	1.8310	1.7714	1.6925	1.8441	1.7494	1.7910	1.8112
1.6593	1.8311	1.7712	1.6923	1.8438	1.7496	1.7907	1.8114
1.6594	1.8313	1.7709	1.6927	1.8438	1.7496	1.7906	1.8109

CHANNEL				
9	10	11	12	13
1.7941	1.7381	1.7211	1.8335	-.0119
1.7946	1.7389	1.7221	1.8336	-.0119
1.7961	1.7397	1.7239	1.8353	-.0119
1.7958	1.7400	1.7232	1.8341	-.0119
1.7960	1.7401	1.7237	1.8351	-.0119
1.7957	1.7402	1.7236	1.8347	-.0119
1.7955	1.7388	1.7233	1.8340	-.0119
1.7959	1.7394	1.7226	1.8342	-.0119
1.7958	1.7398	1.7233	1.8338	-.0119
1.7954	1.7394	1.7231	1.8341	-.0119

	MEAN	STANDARD DEVIATION
channel 1	1.65961	.000542524961023
channel 2	1.83108	.000528730134904
channel 3	1.77124	.000587272414548
channel 4	1.69261	.000583952052826
channel 5	1.84389	.000574359546703
channel 6	1.7494	.000537483849887
channel 7	1.79093	.0006254775953
channel 8	1.8111	.000697614984549
channel 9	1.79549	.000647130417905
channel 10	1.73944	.000668663675633
channel 11	1.72299	.000847807630172
channel 12	1.83424	.000607728009338
channel 13	-.0118729	8.1846740246E-6
channel 17	14.9017	.00368329562575

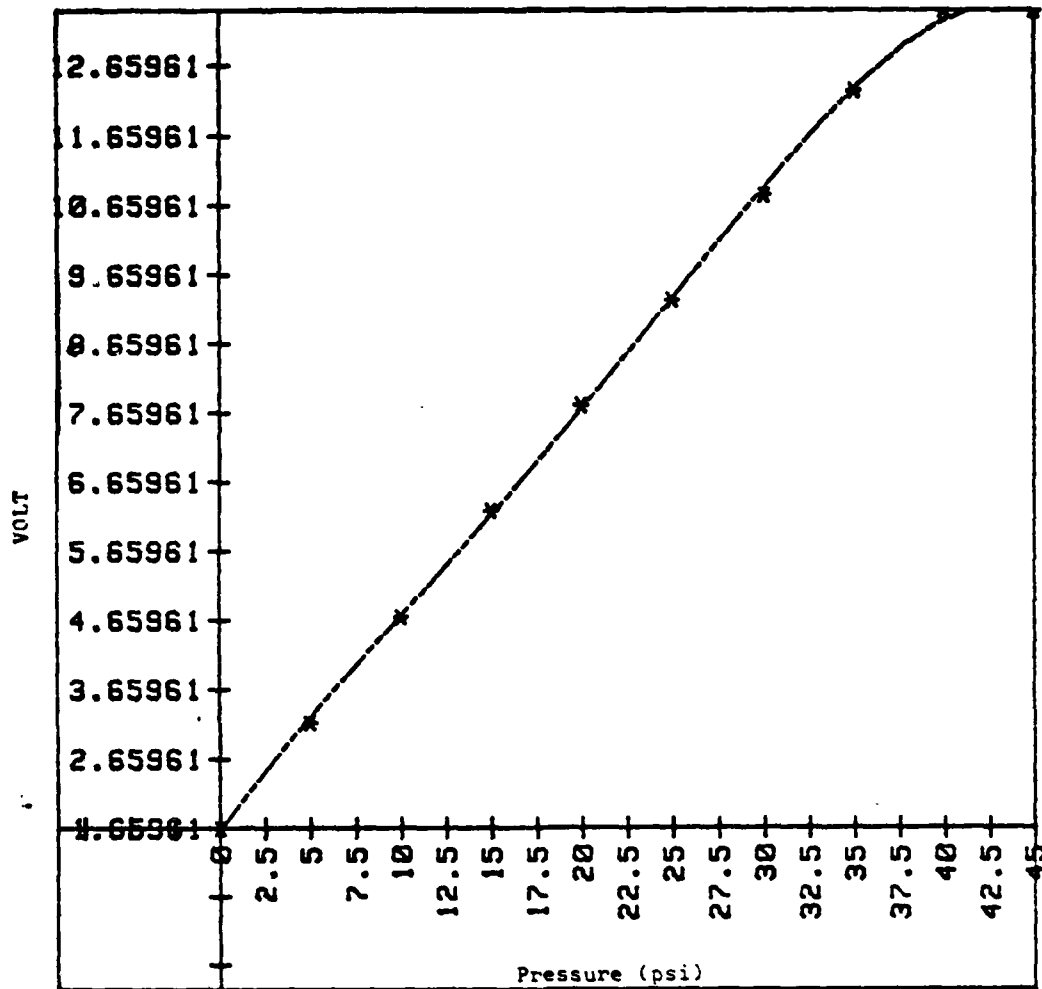
APPENDIX H: SAMPLE 200PC CALIBRATION PROGRAM

```

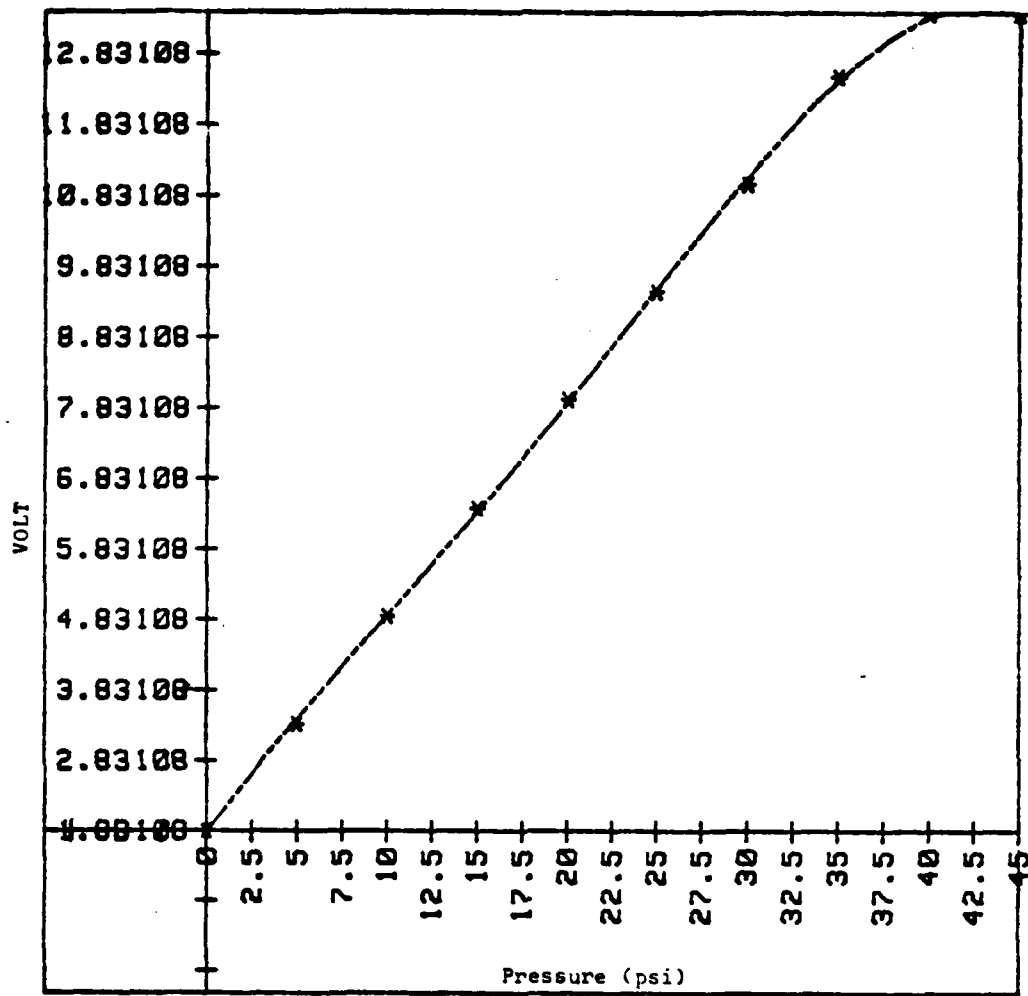
PRINT "THIS PROGRAM IS USED TO CALIBRATE 100 PSI PRESSURE TRANSDUCERS."
PRINT "ENTER THE AMOUNT OF READINGS YOU WISH THE PROGRAM TO AVERAGE"
DIM X(11,100),Sum(11),Mean(11),Sum2(11),Sd(11)
PRINT
INPUT N
Y=0
FOR I=1 TO 11
  PRINT
  PRINT
  PRINT "THIS RUN IS FOR A PRESSURE OF";Y;"PSI"
  PRINT
  PRINT
  PRINT "ONCE THE PRESSURE IS SET AT";Y;"PSI"
  PRINT "PUSH THE CONTINUE BUTTON AND WAIT FOR DATA."
  Y=Y+10
  PAUSE
  FOR J=1 TO N
    OUTPUT 709;"AI12VT11"
    ENTER 709;X(1,I)
    OUTPUT 709;"AI14VT11"
    ENTER 709;X(2,I)
    OUTPUT 709;"AI15VT11"
    ENTER 709;X(3,I)
  NEXT J

```

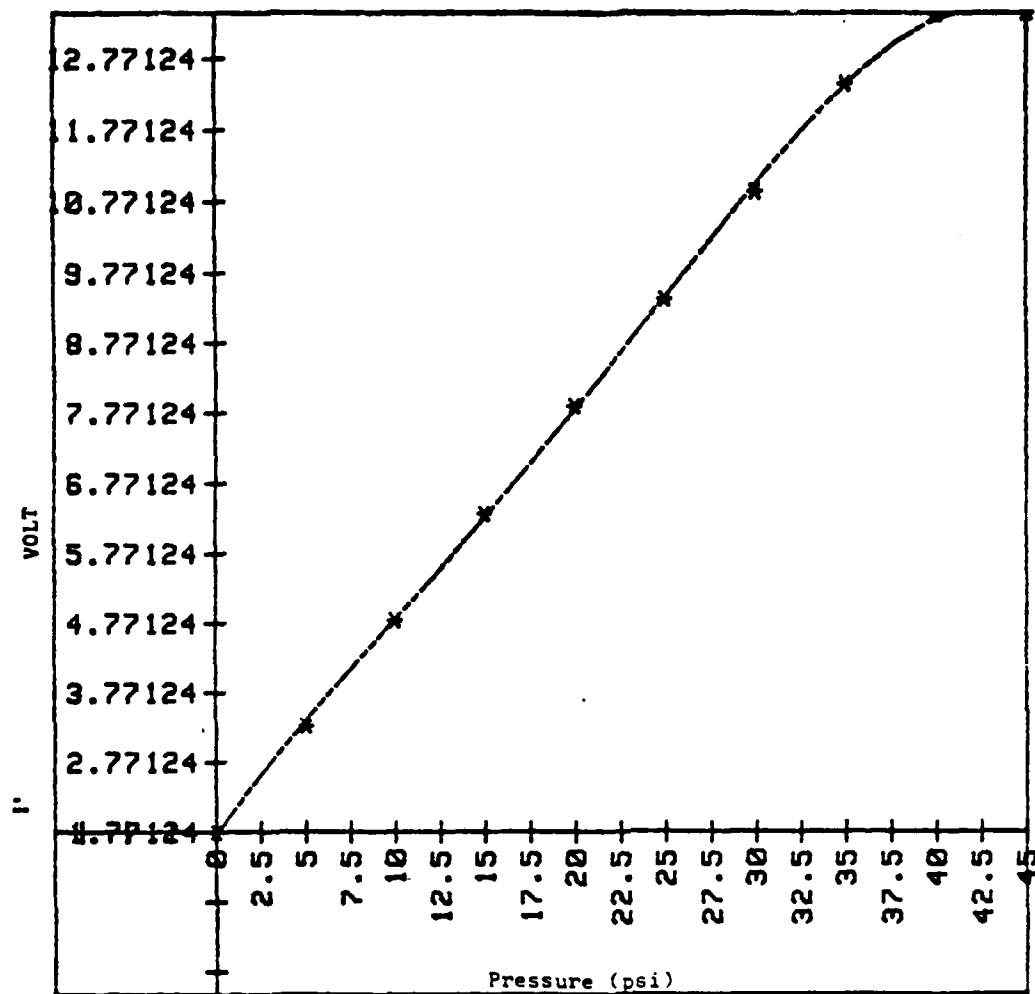

APPENDIX I: CALIBRATION PLOT OF PRESSURE TRANSDUCERS



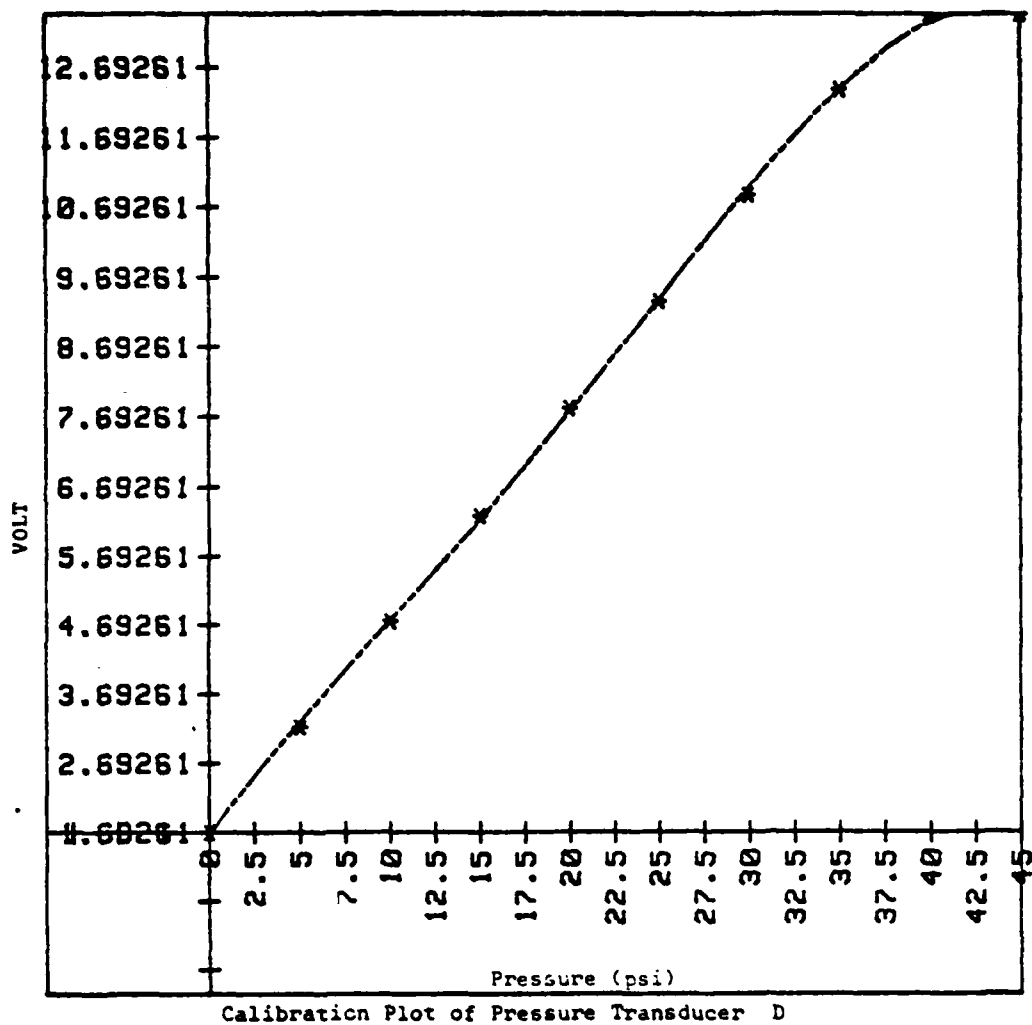
Calibration Plot of Pressure Transducer A



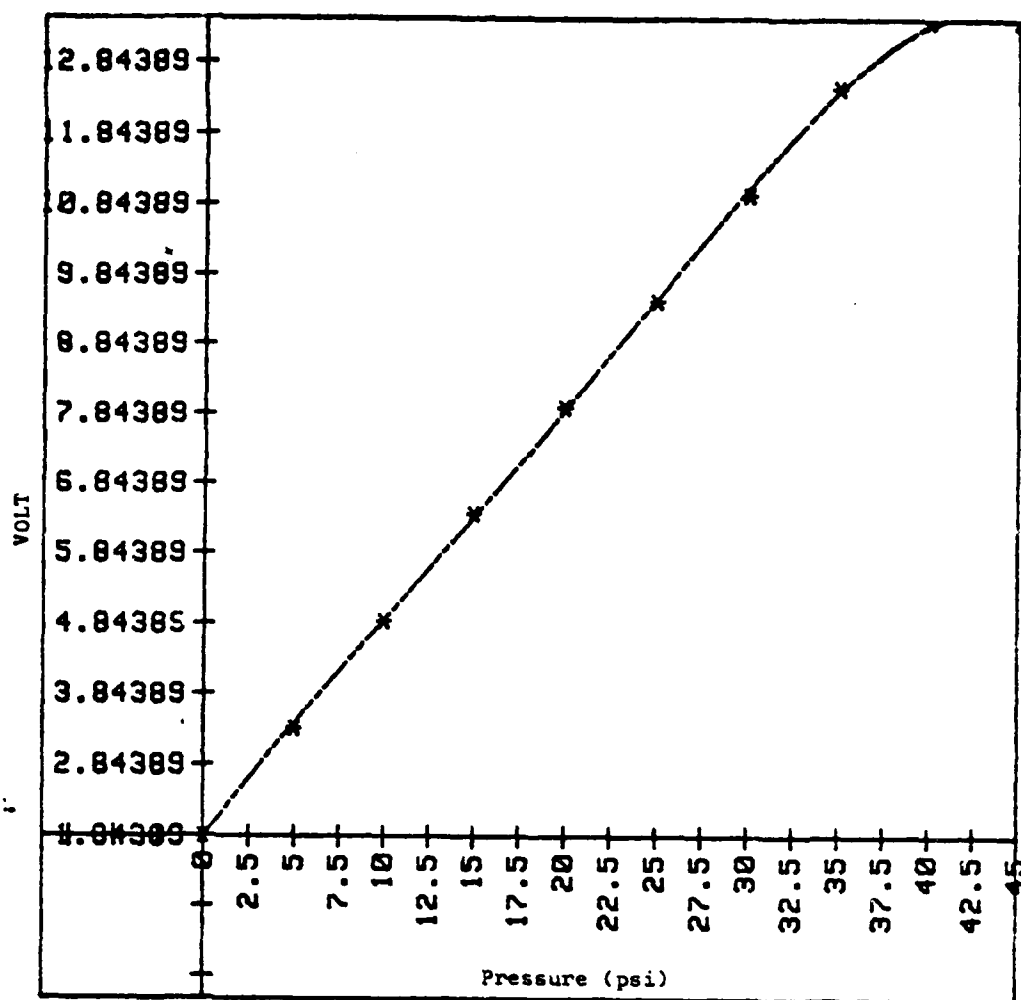
Calibration Plot of Pressure Transducer B



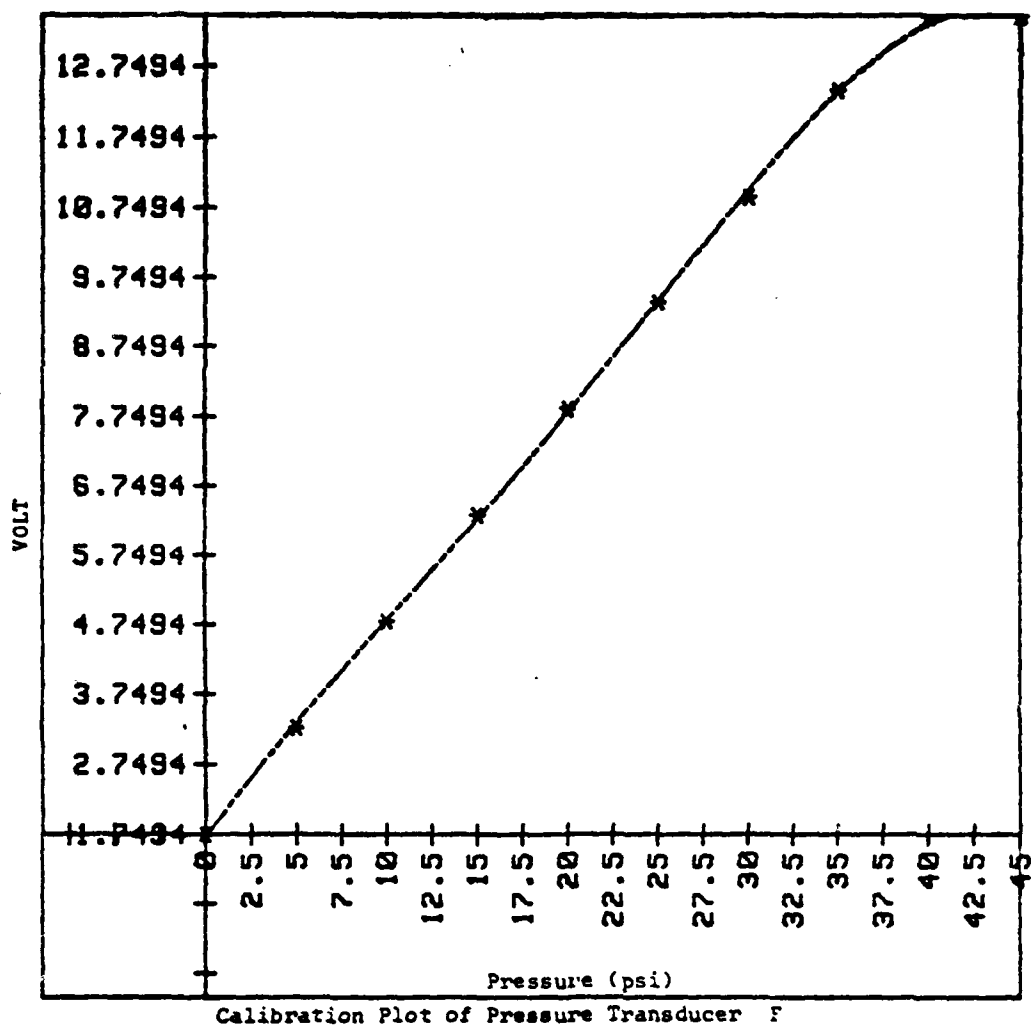
Calibration Plot of Pressure Transducer C

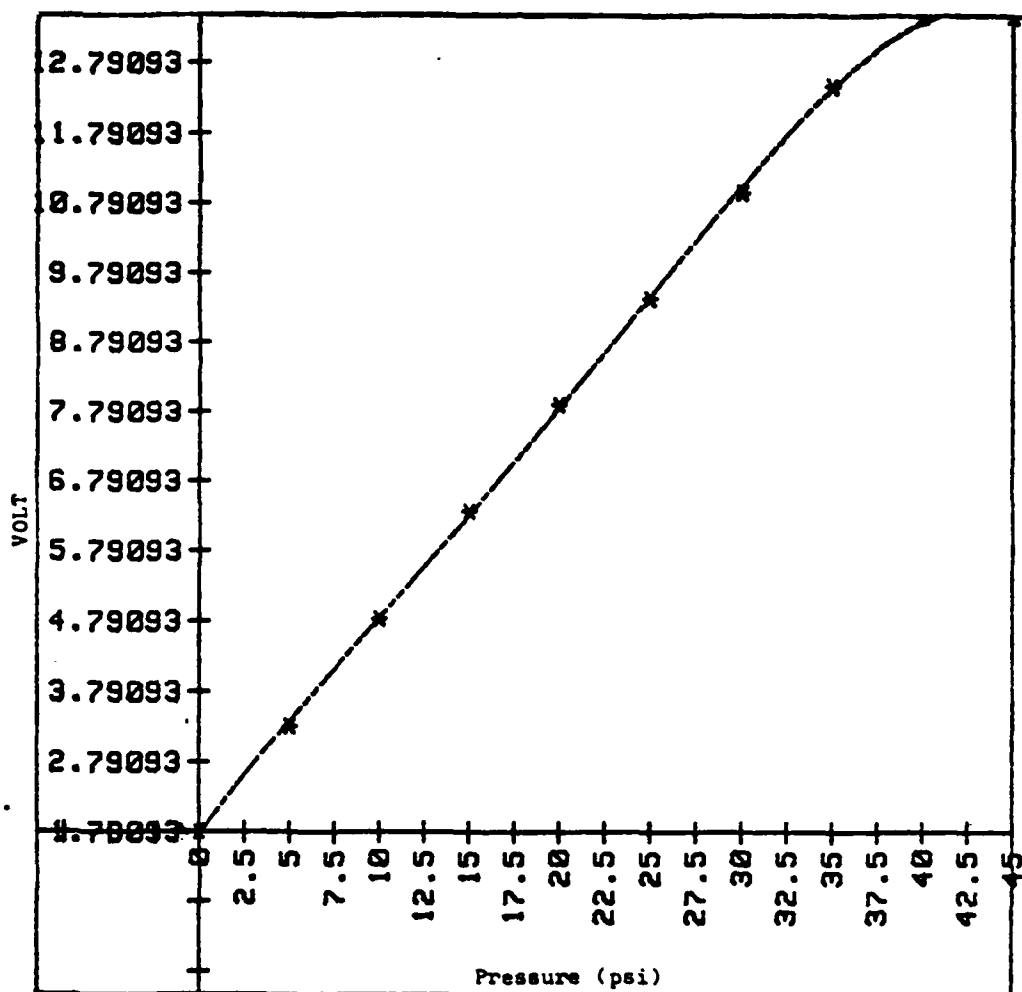


Calibration Plot of Pressure Transducer D

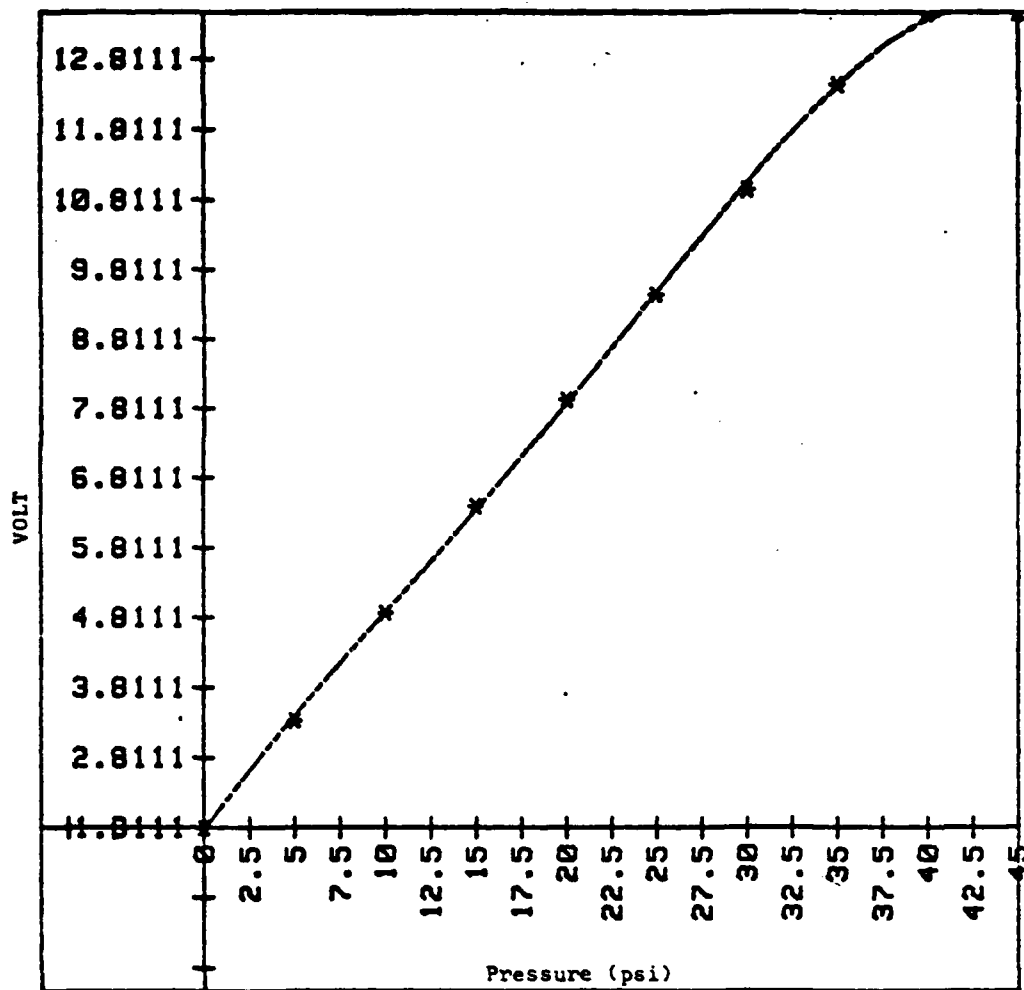


Calibration Plot of Pressure Transducer E

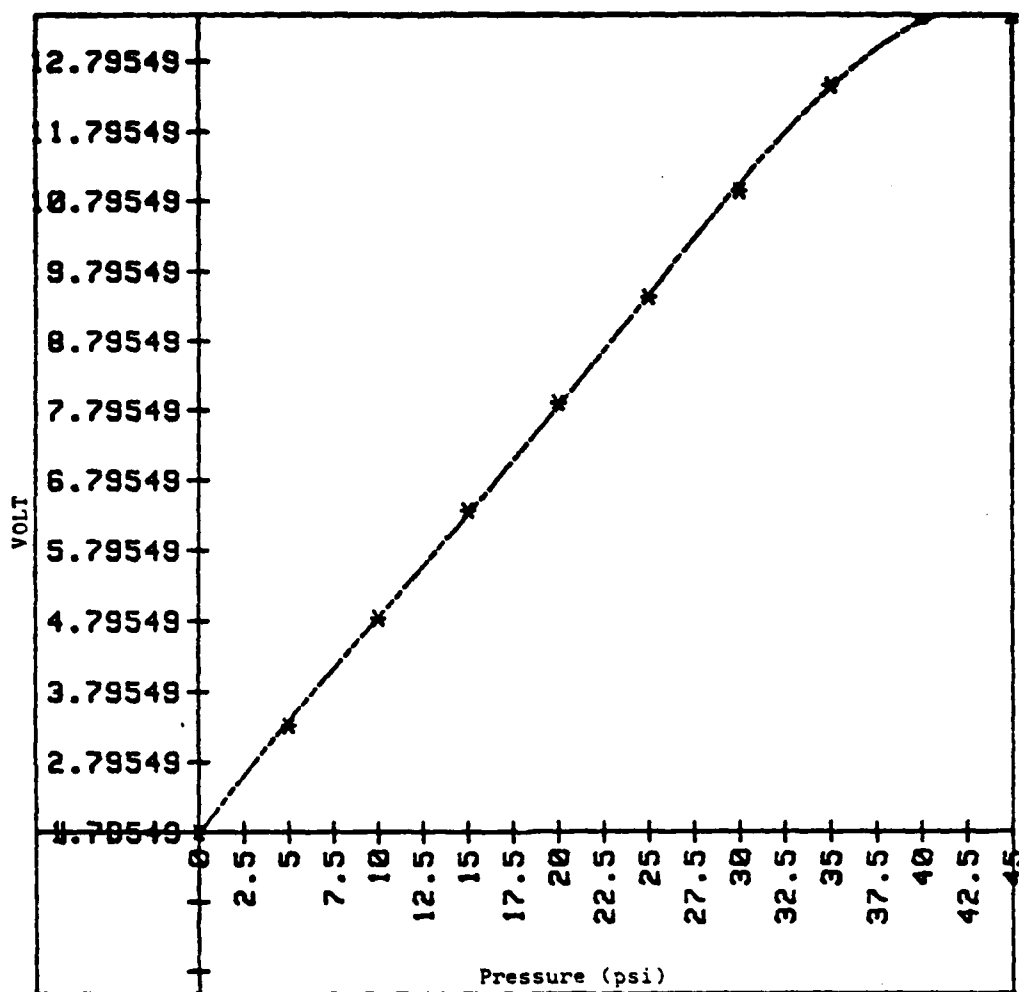




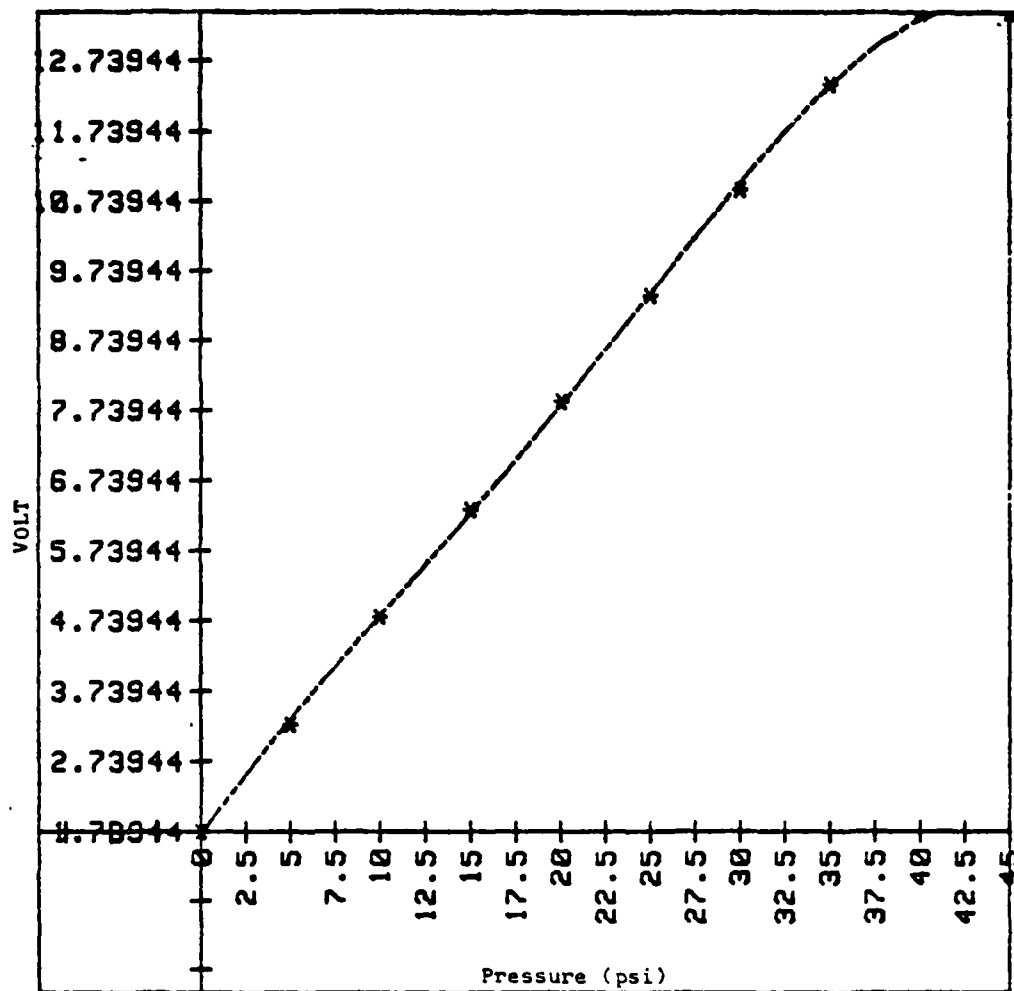
Calibration Plot of Pressure Transducer G



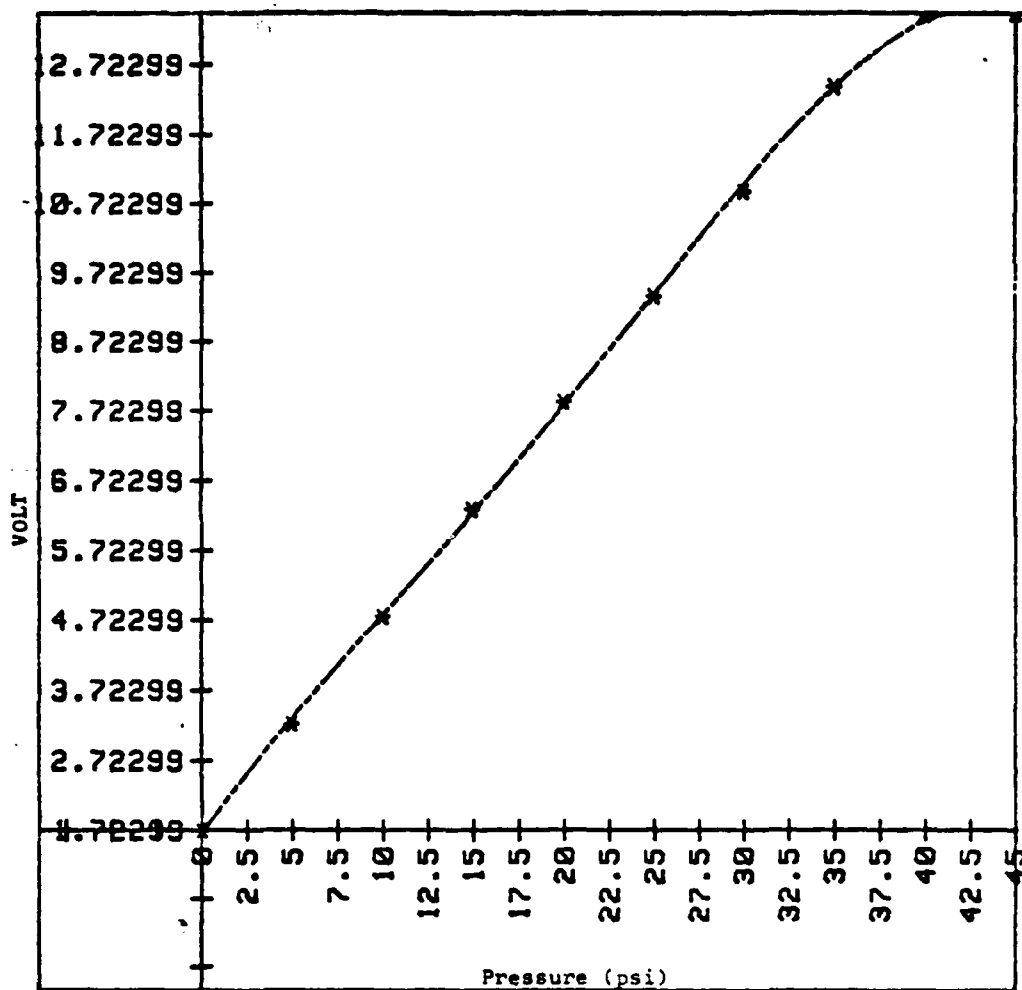
Calibration Plot of Pressure Transducer H



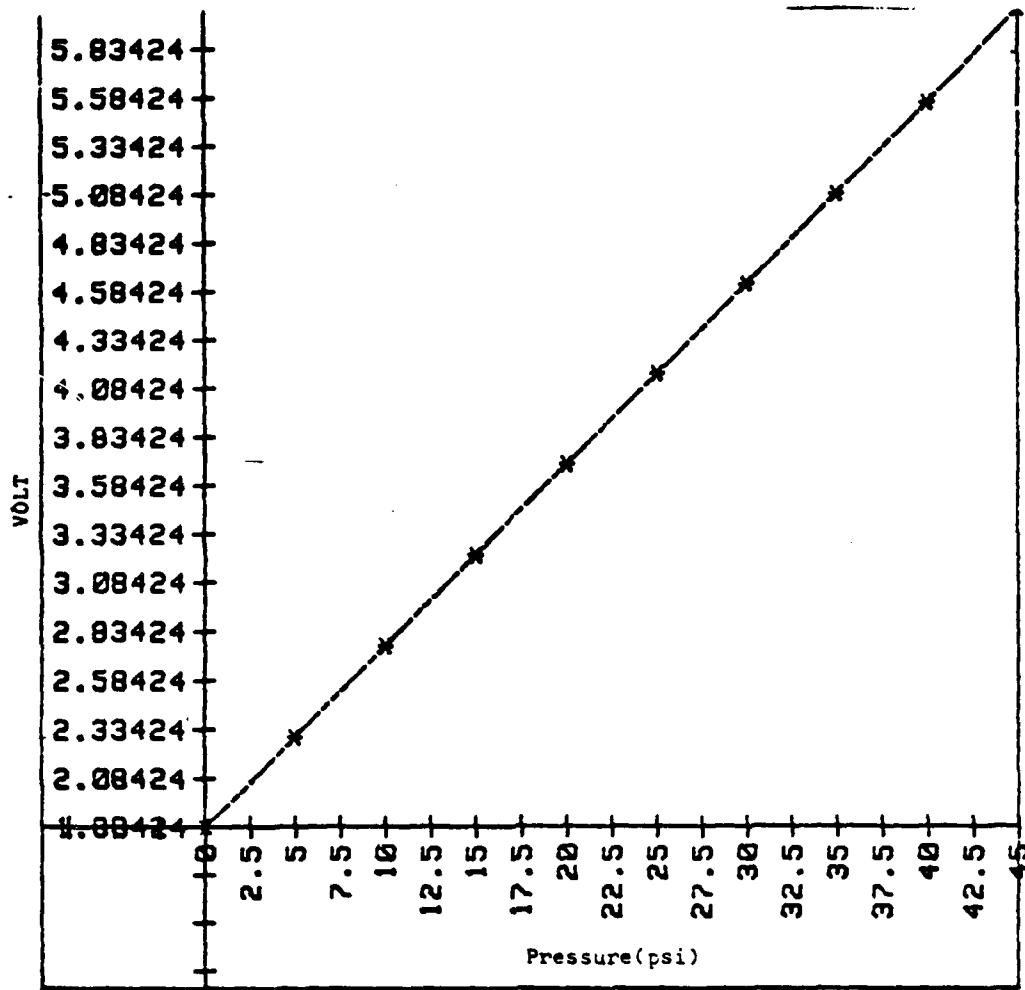
Calibration Plot of Pressure Transducer I



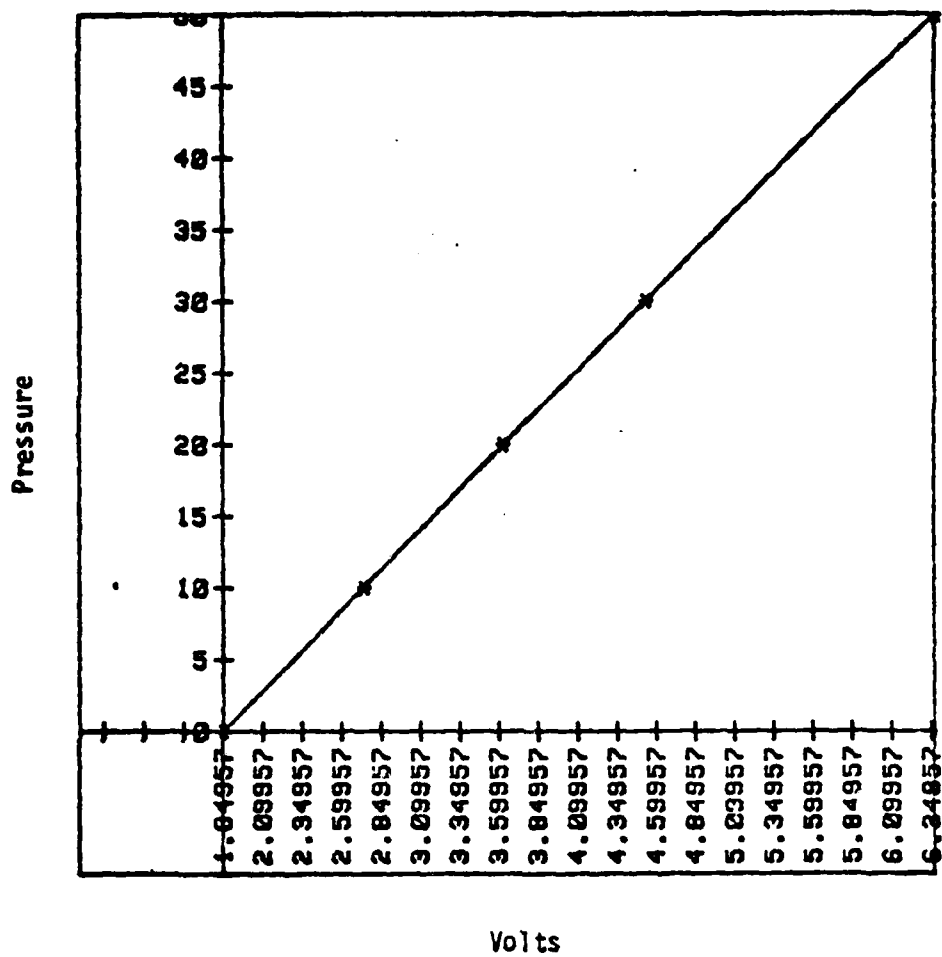
Calibration Plot of Pressure Transducer J



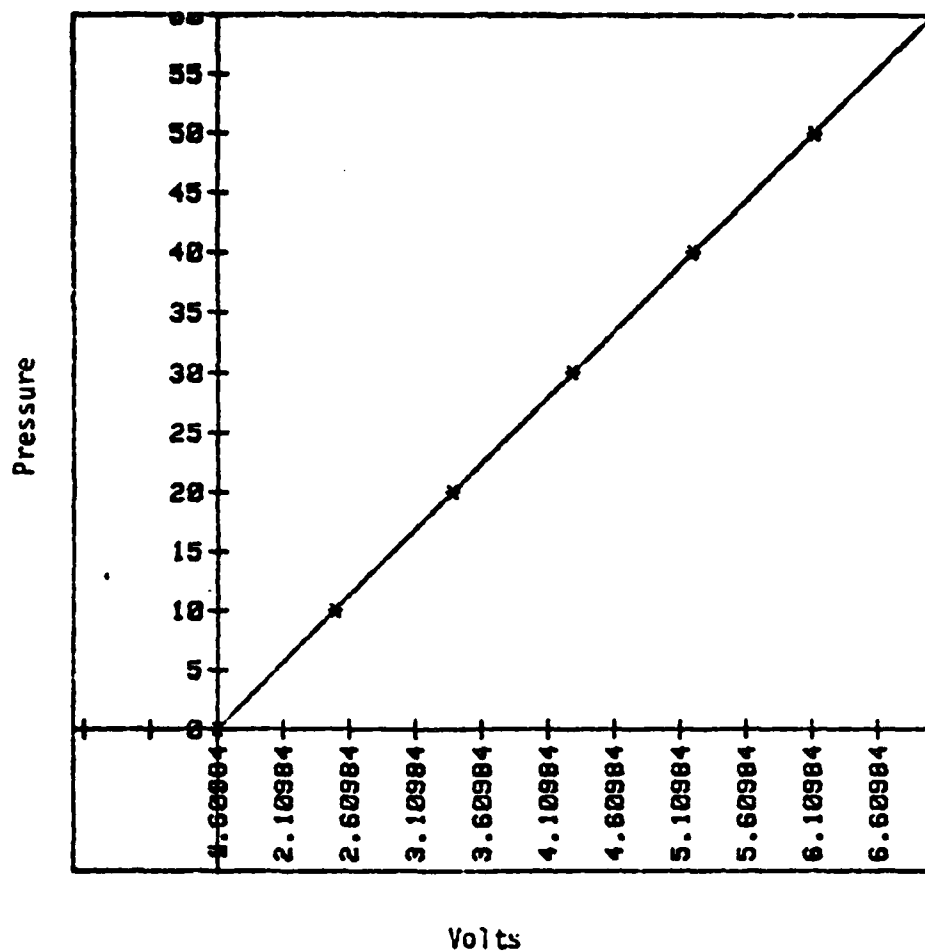
Calibration Plot of Pressure Transducer K



Calibration Plot of Pressure Transducer L

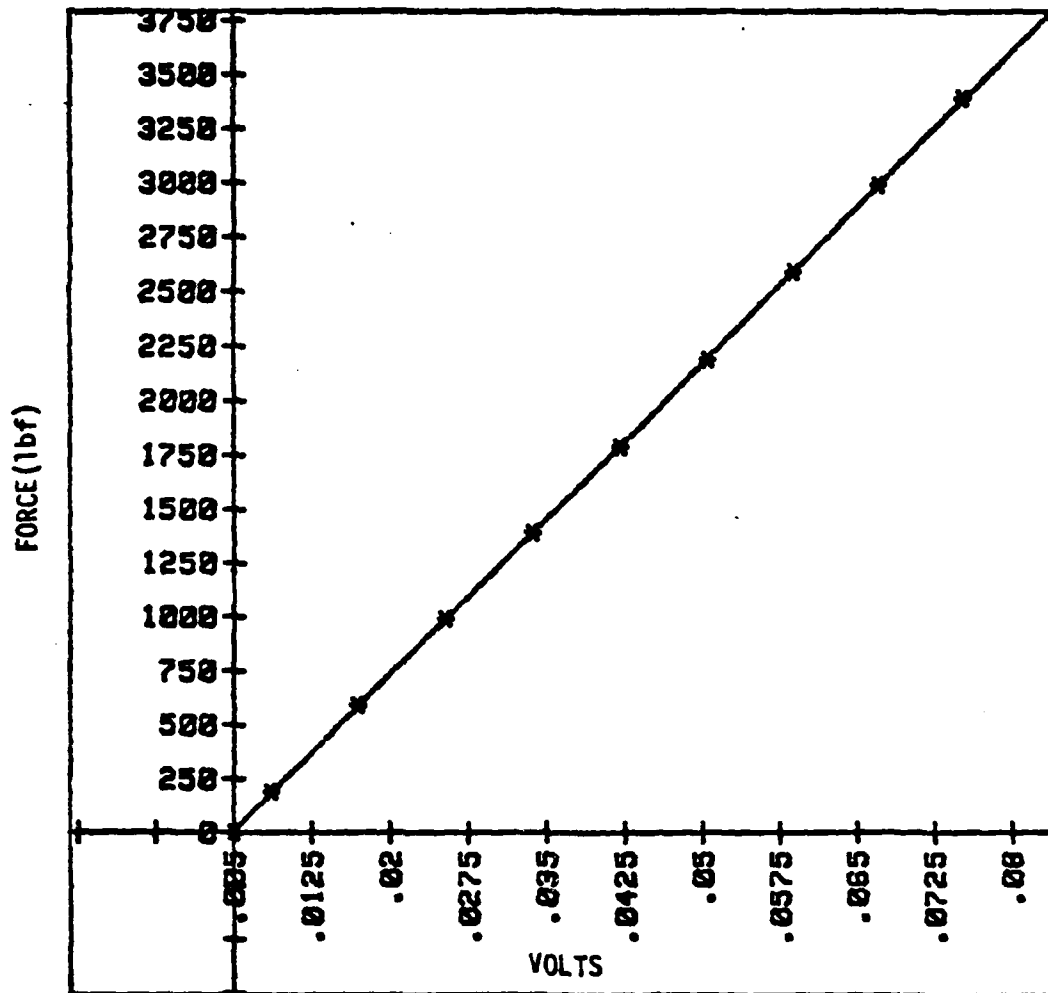


Calibration plot of pressure Transducer 0



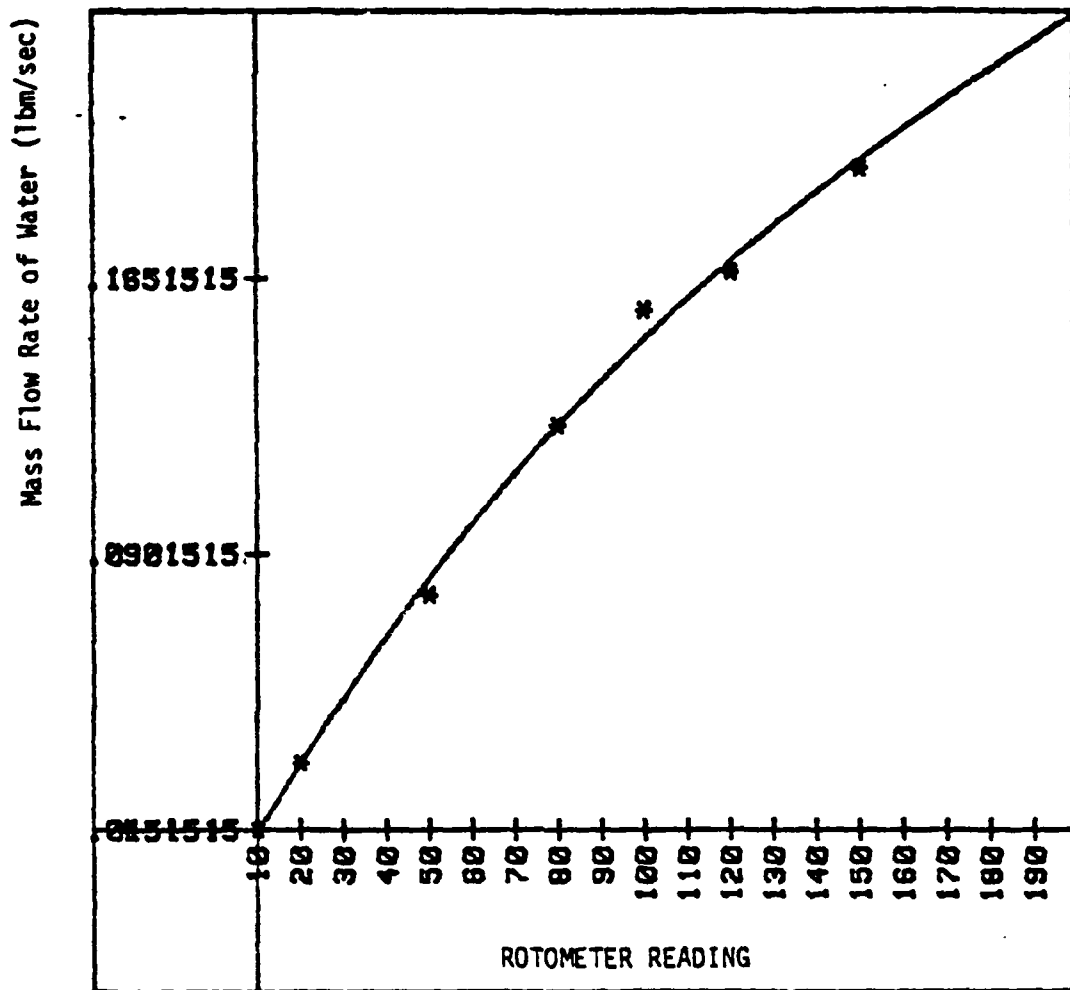
Calibration Plot of Pressure Transducer P

APPENDIX J: CALIBRATION PLOT FOR FORCE-BLOCK



Calibration Plot For Force-Block

APPENDIX K: CALIBRATION PLOT FOR ROTOMETER



Calibration Plot for Rotometer

APPENDIX L: DATA ACQUISITION AND ANALYSIS PROGRAM

```

3  PRINT "INPUT ROTOMETER READING"
4  INPUT Rr
5  PRINT Rr
7  DIM A(14),B(14),C(14),Dist(14),D(14)
8  DIM Press(14)
10 DIM X(18,100),Sum(18),Mean(18),Sum2(18)
30 N=10
40 FOR I=1 TO N
50     OUTPUT 709;"AI1VT1"
60     ENTER 709:X(1,I)
70     OUTPUT 709;"AI2VT1"
80     ENTER 709:X(2,I)
90     OUTPUT 709;"AI3VT1"
100    ENTER 709:X(3,I)
110    OUTPUT 709;"AI4VT1"
120    ENTER 709:X(4,I)
130    OUTPUT 709;"AI5VT1"
140    ENTER 709:X(5,I)
150    OUTPUT 709;"AI6VT1"
160    ENTER 709:X(6,I)
170    OUTPUT 709;"AI7VT1"
180    ENTER 709:X(7,I)
190    OUTPUT 709;"AI8VT1"
200    ENTER 709:X(8,I)
210    OUTPUT 709;"AI9VT1"
220    ENTER 709:X(9,I)
230    OUTPUT 709;"AI10VT1"
240    ENTER 709:X(10,I)
250    OUTPUT 709;"AI11VT1"
260    ENTER 709:X(11,I)
261    OUTPUT 709;"AI12VT1"
262    ENTER 709:X(12,I)
263    OUTPUT 709;"AI13VT1"
264    ENTER 709:X(13,I)
265    OUTPUT 709;"AI14VT1"
266    ENTER 709:X(14,I)
267    OUTPUT 709;"AI15VT1"
268    ENTER 709:X(15,I)
270 NEXT I
440 FOR J=1 TO 16
450     Sum(J)=0
460     FOR I=1 TO N
470         Sum(J)=Sum(J)+X(J,I)
480     NEXT I
490     Mean(J)=Sum(J)/N
510 NEXT J

```



```

520 FOR I=1 TO 14
530 READ A(I),B(I),C(I),D(I),Dist(I)
540 DATA -6.049177,3.92598,-.1736131,.0112105,.5
550 DATA -7.059265,4.233834,-.2292065,.01415403,1.5
560 DATA -6.4905288,3.93412037,-.160249802,.009448598,2.5
570 DATA -7.05653853,4.69207328,-.341175732,.0189756532,3.5
580 DATA -7.00950749,4.15163753,-.207173977,.0123697788,4.5
590 DATA -6.38855694,3.93348997,-.160546739,.0095126941,5.5
600 DATA -6.58938462,3.95696947,-.164270383,.0096133938,6.5
610 DATA -6.587731,3.90750953,-.158062598,.0094269372,7.5
620 DATA -6.59491204,3.94889685,-.162699726,.009565139,8.5
630 DATA -6.35328646,3.92659595,-.165085722,.0099773509,9.5
640 DATA -6.20324295,3.84628939,-.146968664,.0088046866,10.5
650 DATA -20.2065629,11.1571691,-.0859126,.004917379,0
660 DATA -20.6682414,11.0677155,-.07562378,.011185984,0
670 DATA -18.8267355,11.9281137,-.17715903,.011005888,0
680 Press(I)=A(I)+B(I)*Mean(I)+C(I)*Mean(I).2+D(I)*Mean(I).3
690 Press(I)=Press(I)+14.696
700 NEXT I

```

```

700 Fb=1/453.6*(-234.396482+48715.7753*Mean(15)-11261.538*Mean(15)^2+33899.3*M
ean(15)^3)
710 Mwater=-.0063268+.002097278*Rr-.00000658*Rr^2+.000000011*Rr^3
720 Fn=Fb*22.5/22.8
730 Dw=.920/12
740 Dd=3/12
750 E=1/(1-Dw/Dd)
760 Dp=Mean(14)-Mean(13)
770 Cc=.62
771 K=Cc*E
772 Aa=3.1416*Dw^2/4
774 Y=1-(-.41+.35*(Dw/Dd)^4*(Dp/(1.4*Mean(14))))
780 Gc=32.2
790 Den=.076297*Mean(12)/14.696
800 Mair=K*Aa*Y*SQR(2*Gc*Den*Dp*144)
810 Mtotal=Mair+Mwater
820 Vexit=Fn/(Mtotal)*32.2
830 PRINT
840 PRINT "PRESSURE AS A FUNCTION OF DISTANCE"
850 PRINT "
860 PRINT "DISTANCE(X)", "PRESSURE"
870 PRINT "
FOR I=1 TO 14
880 PRINT Dist(I), Press(I)
890 NEXT I
900 PRINT "MASS FLOW RATE OF WATER"
910 PRINT "MASS FLOW RATE OF AIR"
920 PRINT "TOTAL MASS FLOW RATE"
930 PRINT "THRUST"
940 PRINT "MIXTURE RATIO"
950 PRINT "EXIT VELOCITY"
960 PRINT "INLET PRESSURE"
970 PRINT "
980 END
";Mwater;"LBM/SEC"
";Mair;"LBM/SEC"
";Mtotal;"LBM/SEC"
";Fn;"LBF"
";Mwater/Mair
";Vexit;"FT/SEC"
";Press(12);"PSI"

```

INPUT ROTOMETER READING
0

PRESSURE AS A FUNCTION OF DISTANCE

DISTANCE(X)	PRESSURE
.5	14.5598137417
1.5	14.4607948757
2.5	14.4788575371
3.5	14.4188520649
4.5	14.4711792506
5.5	14.5553698274
6.5	14.4532490549
7.5	14.4461409662
8.5	14.5336111051
9.5	14.5262956477
10.5	14.5245848183
0	14.1098474897
0	11.8981971096
0	17.6235437008
MASS FLOW RATE OF WATER	= .0063268 LBM/SEC
MASS FLOW RATE OF AIR	= .0110711550501 LBM/SEC
TOTAL MASS FLOW RATE	= .00474435505015 LBM/SEC
THRUST	= 1.76920486692 LBF
MIXTURE RATIO	= .571467021403
EXIT VELOCITY	= 12807.6166544 FT/SEC
INLET PRESSURE	= 14.11078474897 PSI

APPENDIX M: DUAL-PHASE TWO-COMPONENT COMPUTER PROGRAM

3000	REAL TWO-PHASE TWO-COMPONENT MODEL PROGRAM MODIFIED FOR USE AT NAVAL POSTGRADUATE SCHOOL PROF. J. SLADKY AND LT. T.C. MOLLIS	DATE 1010 TIME 1150 PAGE 2190
COMMON ZZCZAT	DIRECTION ZZCZAT (11200)	
THE ABOVE COMMON BLOCK FOR SUBROUTINES TABLE AND INTCP	IT MUST BE THE FIRST COMMON BLOCK	
COMMON	ALPHAS	ALPHA
1	ALPHAS	ALPHA
2	ALPHAS	ALPHA
3	ALPHAS	ALPHA
4	ALPHAS	ALPHA
5	ALPHAS	ALPHA
6	ALPHAS	ALPHA
7	ALPHAS	ALPHA
8	ALPHAS	ALPHA
9	ALPHAS	ALPHA
10	ALPHAS	ALPHA
11	ALPHAS	ALPHA
12	ALPHAS	ALPHA
13	ALPHAS	ALPHA
14	ALPHAS	ALPHA
15	ALPHAS	ALPHA
16	ALPHAS	ALPHA
17	ALPHAS	ALPHA
18	ALPHAS	ALPHA
19	ALPHAS	ALPHA
20	ALPHAS	ALPHA
21	ALPHAS	ALPHA
22	ALPHAS	ALPHA
23	ALPHAS	ALPHA
24	ALPHAS	ALPHA
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31	ALPHAS	ALPHA
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35	ALPHAS	ALPHA
36	ALPHAS	ALPHA
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88	ALPHAS	ALPHA
89	ALPHAS	ALPHA
90	ALPHAS	ALPHA
91	ALPHAS	ALPHA
92	ALPHAS	ALPHA
93	ALPHAS	ALPHA
94	ALPHAS	ALPHA
95	ALPHAS	ALPHA
96	ALPHAS	ALPHA
97	ALPHAS	ALPHA
98	ALPHAS	ALPHA
99	ALPHAS	ALPHA
100	ALPHAS	ALPHA

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2 DIMENSION PL(10) , Y0 , YOS , ZZZ , NSTMT
DIMENSION MURAY(75)
DIMENSION DATE(5), CACT(3), OLD(8), KP(2,75), PL1(13)
1 C(16), AD(2,75), KXA(4), OSRAY(103), OSRAY1(103), TDRAY(103), TDRAY1(103)
21(100) DRAY(75), HT(14), DINT(8)
EQUIVALENCE (MURAY(1), DRAY(1))
XX=-1.0
UNIN = 1.0E-6
IZEN = 7
YOS = 7
DINT(1) = 5
DINT(2) = 1
DINT(3) = .01
DINT(4) = .001
DINT(5) = .0001
DINT(6) = .00001
DINT(7) = .000001
DINT(8) = .0000001
CALL ADATA

C INITIALIZE TABLE TAPP, BRING IN PERMANENT TABLES (ONE-DIMENSIONAL),
C FILL ARRAY INDICATING TEMPERATURES INCLUDED IN TWO
C DIMENSIONAL TABLES, INITIALIZE PAGE COUNT.
5 LOGCT=1
CALL TABLE
CALL INTERP (VAR,0,2,P)
C ACTIVATE THE FOLLOWING CARD TO GET A PRINTOUT OF THE PROPERTIES
CALL INTERP (VAR,20,7,C)
NSTMT=1999
100 CONTINUE
CALL SECT1
IF (NSTMT-5) 999,5,101
101 CONTINUE
IF (NSTMT-99) 999,99,200
999 CALL SUMF
200 CONTINUE
CALL SECT2
IF (NSTMT-63) 999,300,201
201 IF (NSTMT-52) 999,300,202
202 IF (NSTMT-31) 999,400,203
203 IF (NSTMT-15) 999,700,999
300 CONTINUE
CALL SECT3
IF (NSTMT-32) 999,701,101
101 IF (NSTMT-11) 999,700,102
102 IF (NSTMT-145) 999,700,104
104 IF (NSTMT-161) 999,800,105
305 IF (NSTMT-1142) 999,200,999
400 CONTINUE
CALL SECT4
IF (NSTMT-10) 999,200,401
401 IF (NSTMT-11) 999,700,402
402 IF (NSTMT-123) 999,600,403
403 IF (NSTMT-161) 999,200,999
500 CONTINUE
CALL SECT5
IF (NSTMT-10) 999,200,501
501 IF (NSTMT-11) 999,700,502
502 IF (NSTMT-239) 999,600,999
600 CONTINUE
CALL SECT6
IF (NSTMT-103) 999,500,601
601 IF (NSTMT-113) 999,700,602
602 IF (NSTMT-62) 999,800,602
602 IF (NSTMT-1490) 999,500,999
700 CONTINUE
CALL ENECAT
IF (NSTMT-497) 999,800,999
800 CONTINUE

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FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

Year	1970	1971	1972	1973	1974	1975	1976	1977	1978	1979	1980	1981	1982	1983	1984	1985	1986	1987	1988	1989	1990	1991	1992	1993	1994	1995	1996	1997	1998	1999	2000	2001	2002	2003	2004	2005	2006	2007	2008	2009	2010	2011	2012	2013	2014	2015	2016	2017	2018	2019	2020	2021	2022	2023	2024	2025	2026	2027	2028	2029	2030	2031	2032	2033	2034	2035	2036	2037	2038	2039	2040	2041	2042	2043	2044	2045	2046	2047	2048	2049	2050	2051	2052	2053	2054	2055	2056	2057	2058	2059	2060	2061	2062	2063	2064	2065	2066	2067	2068	2069	2070	2071	2072	2073	2074	2075	2076	2077	2078	2079	2080	2081	2082	2083	2084	2085	2086	2087	2088	2089	2090	2091	2092	2093	2094	2095	2096	2097	2098	2099	2100
1970	1971	1972	1973	1974	1975	1976	1977	1978	1979	1980	1981	1982	1983	1984	1985	1986	1987	1988	1989	1990	1991	1992	1993	1994	1995	1996	1997	1998	1999	2000	2001	2002	2003	2004	2005	2006	2007	2008	2009	2010	2011	2012	2013	2014	2015	2016	2017	2018	2019	2020	2021	2022	2023	2024	2025	2026	2027	2028	2029	2030	2031	2032	2033	2034	2035	2036	2037	2038	2039	2040	2041	2042	2043	2044	2045	2046	2047	2048	2049	2050	2051	2052	2053	2054	2055	2056	2057	2058	2059	2060	2061	2062	2063	2064	2065	2066	2067	2068	2069	2070	2071	2072	2073	2074	2075	2076	2077	2078	2079	2080	2081	2082	2083	2084	2085	2086	2087	2088	2089	2090	2091	2092	2093	2094	2095	2096	2097	2098	2099	2100	

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15 IF (C(L) - 1000000.) 18, 9916, 18
18 XP(1, K) = C(L)
19 IF (K = 1) K = 1
19 IF (K = 1) 19, 19, 21
19 K = K + 1
1320 CONTINUE
GO TO 7
9916 K = K - 1
C THIS IS A FIX FOR PRESSURE TABLE INTERP AT STEP 51 JHM
C KXPI = K
C PRINTOUT 1(P) TABLE JUST READ IN
L1=1
L2=9
WRITE (6, 915)
16 WRITE (6, 919) (XP(1, LT), LT=L1, L2)
WRITE (6, 917) (XP(2, LT), LT=L1, L2)
LCT=LC+1+3
IF (L2-K) 11, 22, 22
13 L1=L1+4
L2=AMIN(10, L2+8, K)
IF (LCT=1-5) 16, 17, 17
C MAXIMUM LINE COUNT EXCEEDED - START NEW PAGE.
17 WRITE (6, 918) LPGCT
LCT=4
LPGCT=LPGCT+1
GO TO 16
C 1(P) TABLE LIMITS EXCEEDED - PRINT ERROR MESSAGE AND
C EXIT TO NEXT CASE.
21 WRITE (6, 921)
GO TO 10
22 CONTINUE
C APPROPRIATE TABLES HAVE BEEN READ IN AS REQUIRED
1360 CONTINUE
WRITE (6, 916)
RETURN
5 NGMT=5
RETURN
7777 STOP
409 FORMAT (1H3, 1X, 5HPRESS, 1X, 10P15.4)
910 FORMAT (4H9, A2, 3H4, 7H9, A2, 216)
911 FORMAT (1110)
912 FORMAT (5F12.6, //, 6F12.6, //, 6F12.6, //, 6F12.6)
913 FORMAT (1H1, 1X, 4HREAL, 1X, 13HPAGE TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE ELEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TWELVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THIRTEEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOURTEEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIFTEEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIXTEEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVENTEEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHTEEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINETEEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TWENTY-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TWENTY-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TWENTY-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TWENTY-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TWENTY-FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TWENTY-FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TWENTY-SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TWENTY-SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TWENTY-EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE TWENTY-NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THIRTY-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THIRTY-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THIRTY-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THIRTY-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THIRTY-FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THIRTY-FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THIRTY-SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THIRTY-SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THIRTY-EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THIRTY-NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FORTY-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FORTY-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FORTY-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FORTY-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FORTY-FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FORTY-FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FORTY-SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FORTY-SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FORTY-EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FORTY-NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIFTY-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIFTY-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIFTY-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIFTY-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIFTY-FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIFTY-FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIFTY-SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIFTY-SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIFTY-EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIFTY-NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIXTY-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIXTY-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIXTY-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIXTY-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIXTY-FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIXTY-FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIXTY-SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIXTY-SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIXTY-EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SIXTY-NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVENTY-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVENTY-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVENTY-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVENTY-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVENTY-FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVENTY-FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVENTY-SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVENTY-SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVENTY-EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE SEVENTY-NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHTY-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHTY-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHTY-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHTY-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHTY-FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHTY-FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHTY-SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHTY-SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHTY-EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE EIGHTY-NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINETY-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINETY-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINETY-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINETY-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINETY-FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINETY-FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINETY-SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINETY-SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINETY-EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE NINETY-NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE HUNDRED-COMPONENT NOZZLE FLOW, 1X, 13HPAGE HUNDRED-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE HUNDRED-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE HUNDRED-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE HUNDRED-FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE HUNDRED-FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE HUNDRED-SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE HUNDRED-SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE HUNDRED-EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE HUNDRED-NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE ONE 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NOZZLE FLOW, 1X, 13HPAGE THREE HUNDRED-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THREE HUNDRED-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THREE HUNDRED-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THREE HUNDRED-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THREE HUNDRED-FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THREE HUNDRED-FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THREE HUNDRED-SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THREE HUNDRED-SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THREE HUNDRED-EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE THREE HUNDRED-NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOUR HUNDRED-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOUR HUNDRED-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOUR HUNDRED-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOUR HUNDRED-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOUR HUNDRED-FOUR-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOUR HUNDRED-FIVE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOUR HUNDRED-SIX-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOUR HUNDRED-SEVEN-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOUR HUNDRED-EIGHT-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FOUR HUNDRED-NINE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIVE HUNDRED-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIVE HUNDRED-ONE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIVE HUNDRED-TWO-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIVE HUNDRED-THREE-COMPONENT NOZZLE FLOW, 1X, 13HPAGE FIVE HUNDRED-FOUR-COMPONENT NOZZLE
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CC 1 2110 1
1 2090 1
1 2070 1
1 2050 1
1 2030 1
1 2010 1
1 1990 1
1 1970 1
1 1950 1
1 1930 1
1 1910 1
1 1890 1
1 1870 1
1 1850 1
1 1830 1
1 1810 1
1 1790 1
1 1770 1
1 1750 1
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1 150 1
1 130 1
1 110 1
1 90 1
1 70 1
1 50 1
1 30 1
1 10 1
1 0 1

744, A2)
8077 FORMAT(1H0,10X,19HISOTHERMAL FLOW TO ,1PE15.4 , 6H PSTA.)
END

SUBROUTINE SECT2

COMMON TZZZAY
DIMENSION TZZZAY(11210)
THE ABOVE COMMON BLOCK FOR SUBROUTINES TABLE AND INTRP
IT MUST BE THE FIRST COMMON BLOCK

C	COMMON	A	ADAB	ALAN	ALPHAB	ALPHA	ALPHMB
1		ALPHS	AM	AS	AT	BA	
2		BB	BETAB	BETAMB	BETAS	BB	
3		C	CA	CANB	CANB	CANMB	
4	CBG1	CAL	CASE	CMBMB	CMB	CMBL	
5	COMMON	CMS	CB	CD	CPIN	CPIN	
6		CPMS	CB	CD	CPIN	CPIN	
7		CLMS	CB	CD	CPIN	CPIN	
8		DD	DELSI	DELSO	DELSO	DELSO	
9	COMMON	DELAGB	DELAGB	DELAGB	DELAGB	DELAGB	
1		DELSIS	DELSO	DELSO	DELSO	DELSO	
2		DRO	DR	DS1	DSAVE	DSAVE	
3		DSMAX	DSMIN	DSRAY1	DSRAY	DSRAY	
4	COMMON	DSIES	DS	DSX1	DSX2	DSX2	
5		DTG	DTG	DTL1	DTL1	DTL1	
6		DVBSQ	DVBSQ	DVBSQ	DVBSQ	DVBSQ	
7		DT	DT	DT	DT	DT	
8		EMBG	EMGB	EMGB	EMGB	EMGB	
9	COMMON	EMLS	EMLS	EMLS	EMLS	EMLS	
1		EM	EM	EM	EM	EM	
2		EMK1	EMK1	EMK1	EMK1	EMK1	
3		EMK2	EMK2	EMK2	EMK2	EMK2	
4	COMMON	EMK3	EMK3	EMK3	EMK3	EMK3	
5		EMK4	EMK4	EMK4	EMK4	EMK4	
6		EMK5	EMK5	EMK5	EMK5	EMK5	
7		EMK6	EMK6	EMK6	EMK6	EMK6	
8		EMK7	EMK7	EMK7	EMK7	EMK7	
9		EMK8	EMK8	EMK8	EMK8	EMK8	
1	COMMON	EMK9	EMK9	EMK9	EMK9	EMK9	
2		EMK10	EMK10	EMK10	EMK10	EMK10	
3		EMK11	EMK11	EMK11	EMK11	EMK11	
4		EMK12	EMK12	EMK12	EMK12	EMK12	
5		EMK13	EMK13	EMK13	EMK13	EMK13	
6		EMK14	EMK14	EMK14	EMK14	EMK14	
7		EMK15	EMK15	EMK15	EMK15	EMK15	
8		EMK16	EMK16	EMK16	EMK16	EMK16	
9		EMK17	EMK17	EMK17	EMK17	EMK17	
1	COMMON	EMK18	EMK18	EMK18	EMK18	EMK18	
2		EMK19	EMK19	EMK19	EMK19	EMK19	
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9		EMK26	EMK26	EMK26	EMK26	EMK26	
1	COMMON	EMK27	EMK27	EMK27	EMK27	EMK27	
2		EMK28	EMK28	EMK28	EMK28	EMK28	
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5		EMK31	EMK31	EMK31	EMK31	EMK31	
6		EMK32	EMK32	EMK32	EMK32	EMK32	
7		EMK33	EMK33	EMK33	EMK33	EMK33	
8		EMK34	EMK34	EMK34	EMK34	EMK34	
9		EMK35	EMK35	EMK35	EMK35	EMK35	
1	COMMON	EMK36	EMK36	EMK36	EMK36	EMK36	
2		EMK37	EMK37	EMK37	EMK37	EMK37	
3		EMK38	EMK38	EMK38	EMK38	EMK38	
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6		EMK41	EMK41	EMK41	EMK41	EMK41	
7		EMK42	EMK42	EMK42	EMK42	EMK42	
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9		EMK44	EMK44	EMK44	EMK44	EMK44	
1	COMMON	EMK45	EMK45	EMK45	EMK45	EMK45	
2		EMK46	EMK46	EMK46	EMK46	EMK46	
3		EMK47	EMK47	EMK47	EMK47	EMK47	
4		EMK48	EMK48	EMK48	EMK48	EMK48	
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6		EMK50	EMK50	EMK50	EMK50	EMK50	
7		EMK51	EMK51	EMK51	EMK51	EMK51	
8		EMK52	EMK52	EMK52	EMK52	EMK52	
9		EMK53	EMK53	EMK53	EMK53	EMK53	
1	COMMON	EMK54	EMK54	EMK54	EMK54	EMK54	
2		EMK55	EMK55	EMK55	EMK55	EMK55	
3		EMK56	EMK56	EMK56	EMK56	EMK56	
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7		EMK60	EMK60	EMK60	EMK60	EMK60	
8		EMK61	EMK61	EMK61	EMK61	EMK61	
9		EMK62	EMK62	EMK62	EMK62	EMK62	
1	COMMON	EMK63	EMK63	EMK63	EMK63	EMK63	
2		EMK64	EMK64	EMK64	EMK64	EMK64	
3		EMK65	EMK65	EMK65	EMK65	EMK65	
4		EMK66	EMK66	EMK66	EMK66	EMK66	
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9		EMK71	EMK71	EMK71	EMK71	EMK71	
1	COMMON	EMK72	EMK72	EMK72	EMK72	EMK72	
2		EMK73	EMK73	EMK73	EMK73	EMK73	
3		EMK74	EMK74	EMK74	EMK74	EMK74	
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9		EMK80	EMK80	EMK80	EMK80	EMK80	
1	COMMON	EMK81	EMK81	EMK81	EMK81	EMK81	
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1	COMMON	EMK90	EMK90	EMK90	EMK90	EMK90	
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9		EMK98	EMK98	EMK98	EMK98	EMK98	
1	COMMON	EMK99	EMK99	EMK99	EMK99	EMK99	
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3		EMK101	EMK101	EMK101	EMK101	EMK101	
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9		EMK107	EMK107	EMK107	EMK107	EMK107	
1	COMMON	EMK108	EMK108	EMK108	EMK108	EMK108	
2		EMK109	EMK109	EMK109	EMK109	EMK109	
3		EMK110	EMK110	EMK110	EMK110	EMK110	
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8		EMK115	EMK115	EMK115	EMK115	EMK115	
9		EMK116	EMK116	EMK116	EMK116	EMK116	
1	COMMON	EMK117	EMK117	EMK117	EMK117	EMK117	
2		EMK118	EMK118	EMK118	EMK118	EMK118	
3		EMK119	EMK119	EMK119	EMK119	EMK119	
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6		EMK122	EMK122	EMK122	EMK122	EMK122	
7		EMK123	EMK123	EMK123	EMK123	EMK123	
8		EMK124	EMK124	EMK124	EMK124	EMK124	
9		EMK125	EMK125	EMK125	EMK125	EMK125	
1	COMMON	EMK126	EMK126	EMK126	EMK126	EMK126	
2		EMK127	EMK127	EMK127	EMK127	EMK127	
3		EMK128	EMK128	EMK128	EMK128	EMK128	
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9		EMK134	EMK134	EMK134	EMK134	EMK134	
1	COMMON	EMK135	EMK135	EMK135	EMK135	EMK135	
2		EMK136	EMK136	EMK136	EMK136	EMK136	
3		EMK137	EMK137	EMK137	EMK137	EMK137	
4		EMK138	EMK138	EMK138	EMK138	EMK138	
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6		EMK140	EMK140	EMK140	EMK140	EMK140	
7		EMK141	EMK141	EMK141	EMK141	EMK141	
8		EMK142	EMK142	EMK142	EMK142	EMK142	
9		EMK143	EMK143	EMK143	EMK143	EMK143	
1	COMMON	EMK144	EMK144	EMK144	EMK144</		

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DIMENSION NDRAY(75)
1 DIMENSION DATP(5), CASE(3), QID(9), TP(2,75), PL1(10)
2 C(6), A2(2,75), KXA(8), DSRAY(100), DSRAY1(100), TDRAY(100), TDRAY1(100)
EQUIVALENT(75), HZ(14), DINT(9)
IF(NS=1-10) 1,30,2
1 CALL DUMF
2 IF(NS=1-145) 1,145,3
3 IF(NS=1-161) 1,161,4
4 IF(NS=1-1102) 1,1142,5
5 CONTINUE

SET ITERATION COUNT ZERO
NNR=0

SET INITIAL CONDITIONS INDICATION
***** STATEMENT NO. 20 *****
20 IFP=1
  RPD=1
  ISLA=0
  KAI=1
  NNR=0
  SO=0.7
  PHI=PHI0.017 3329
  AT=103300.3

INITIALIZE PRINTOUT COUNT
NRP=1

SETUP MIDPOINT AND ENDPOINT INDICATION.
INC = 1, NEXT ENTRY TO STATEMENT 32 COMPUTES MIDPOINT
QUANTIFIED.
INC = 1, NEXT ENTRY TO STATEMENT 32 COMPUTES ENDPOINT
QUANTIFIED.
IN=1

RESET THROAT INDICATOR
THRT = 0 - THROAT NOT REACHED
THRT = 1 - THROAT REACHED
THRT=0

RESET DS ITERATION COUNTER.
NDS=0
RESET OPTIMUM S ITERATION COUNTER.
NSO=0

BACKSTEP P FOR FIRST ENTRY TO 30
P = PD-DP1/2.3

SETUP INITIAL CONDITIONS
28 IS=TQ
  IL=TL0
  VG=VLO
  VL=VLO
  D=0
27 I=0.
  DTL=0.0
  DS=0.
  R=0.
  DTG=0.0
  AT=0.

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VGSQ=VG*VG
VLSQ=VL*VL
THQ=THQ
THI=THQ
DELTV=VG-VLSQ
DSI=0.
R=0.
DTM=0.0
DTNI=0.0
***** STATEMENT NO. 10 *****
10 NSTAT=9999
IF (MP) 611,610,630
THIS IS FIRST POINT - HALF STEP P BY INITIAL DP STEP-SIZE.
611 P=P+DP/2.0
GO TO 632
THIS IS NOT FIRST POINT - HALF STEP PRESSURE BY STEP
SIZE DP.
630 P=P+DP/2.0
632 IF (P) 1130,1130,131
STEPPING PRESSURE MADE IT NEGATIVE - PRINT DIAGNOSTIC, AND
DATA COMPUTED TO THIS POINT AND GO TO NEXT CASE.
1130 WRITE (6,930)
LL=1
GO TO 500
EXTRAPOLATE TG AND TL, PRINT DIAGNOSTICS IF THEY GO
NEGATIVE, AND PRINT DATA COMPUTED TO THIS POINT.
131 CONTINUE
TG=TO+DTG/2.0
IF (TG) 132,132,133
132 WRITE (6,931)
LL=2
GO TO 500
133 TL=TL+DTL/2.0
IF (TL) 134,134,135
134 WRITE (6,932)
LL=3
GO TO 500
INTERPOLATE FOR PBO FOR THIS TL
135 N=9
NN=1
CALL INTER(PBO,N,TL,0)
IF (N) 136,136,137
136 WRITE (6,913) NT(NN),TL,P
LL=1
GO TO 500
CHECK DENOMINATOR OF EQUATION FOR PARTIAL PRESSURE
OF COMPONENT A, PRINT DIAGNOSTIC IF SUFFICIENTLY
CLOSE TO ZERO RENDERING PA INFINITE.
137 IF (ABS(1.0-H*PBO)-.0001) 138,138,139
138 WRITE (6,938)
LL=1
GO TO 500
139 PA=(P-PBO)/(1.0-H*PBO)
COMPUTE PA AND PB. IF EITHER IS NEGATIVE, PRINT
DIAGNOSTIC AND GO TO NEXT CASE.

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[illegible]

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CC TABLE LIMITS EXCEEDED - WRITE DIAGNOSTIC.
160 LL=8
161 WRITE (6,2136) HT(NN),TL,P
162 NGENT=1944
GO TO 500
CC INTERPOLATE IN TABLE FOR DENSITY OF LIQUID B AT THIS
TEMPERATURE.
162 NN=11
CALL INTER(ROBL,N,TL,P)
IF(N) 161,161,163
CC COMPUTE DENSITY OF LIQUID MIXTURE
163 RHOL=1.0/((ALPHA/POAL)+(1.0-ALPHA)/ROSL)
CC COMPUTE FLOW RATES OF LIQUID MIXTURE AND OF GAS MIXTURE.
ENG=PNT/(1.1+R)
ENC=P*ENG
CC COMPUTE RATIO OF GAS VOLUME FLOW TO LIQUID VOLUME FLOW.
RV=RHOL/(R*RHOG)
CC INTERPOLATE IN SPECIFIC HEAT TABLES FOR A AND B AT
THIS TEMPERATURE AND PRESSURE.
NN=1
NN=1
CALL INTER(CAG,N,TG,PA)
IF(N) 165,165,170
165 LL=3
GO TO 1142
170 NN=2
NN=2
CALL INTER(CBG,N,TG,PB)
IF(N) 165,165,171
CC INTERPOLATE INTO LATENT HEAT OF VAPORIZATION TABLES
FOR A AND B AT THIS TEMPERATURE.
171 NN=7
NN=7
CALL INTER(HLA,N,TL,P)
IF(N) 161,161,172
172 NN=8
NN=8
CALL INTER(HLB,N,TL,P)
IF(N) 161,161,31
31 IF(NGENT-1) 12,40,32
32 IF(NGENT-1) 60,50,60
40 NGENT=40
50 NGENT=50
60 NGENT=60
500 *DIAG=1
CALL JUEUT(NNN,LIGCT,NGENT)
NGENT=110
RETURN
910 FORMAT(1H0,7X,11HNONPOSITIVE P)
911 FORMAT(1H0,7X,14HNONPOSITIVE TG)
912 FORMAT(1H0,7X,14HNONPOSITIVE TL)
913 FORMAT(1H0,7X,11HINFINITE PA)
9136 FORMAT(1H0,7X,17HOUTSIDE RANGE OF ,A6,7H TABLE,5X,

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      YI=(YAXO-HI)/COS(PHI)
      YO=(YO-PAXO)/COS(PHI)
C   WRITE LINES 1,2,3 OF OUTPUT AND GO TO 33.
C   43 IF (LCT1-42) 644,643,641
C   643 LCT1=0
C       WRITE (6,9543) LRGCT
C       LRGCT=LRGCT+1
C   644 WRITE (7,1331) X,P,R,V3,A,TG,TL,VG,V1
C       WRITE (7,1332) VS,S,V,AV,AA,ALPHA,BETA,ENG,ENL
C       WRITE (7,1333) RHCG,RHOL,WAG,WBG,W,PA,PB,HLA,HLB
C       LCT1=LCT1+9
C       GO TO 30
C   MIDPOINT OF INTERVAL - COMPUTE MID-POINT PARAMETERS.
C   RESET MIDEQINT/ENDPOINT INDICATOR SO NEXT ENTRY TO 32
C   WILL BE AN ENDPOINT.
C   ***** STATEMENT NO. 50 *****
C   50 INT=0
C       MST=9999
C   INTERPOLATE IN VISCOSITY (GAS) TABLES FOR A AND B AT
C   THIS GAS TEMPERATURE.
C       N=15
C       NN=16
C       CALL INTRP(VIAG,N,TG,PA)
C       IF (N) 205,205,206
C   205 LL=12
C       GO TO 1142
C   206 N=17
C       NN=17
C       CALL INTRP(VIBG,N,TG,PB)
C       IF (N) 207,207,208
C   207 LL=12
C       GO TO 145
C   INTERPOLATE IN SPECIFIC HEAT TABLES FOR A AND B AT
C   THIS LIQUID TEMPERATURE.
C   208 N=5
C       NN=5
C       CALL INTRP(CAL,N,TL,P)
C       LL=12
C       IF (N) 161,161,209
C   209 NN=6
C       N=6
C       CALL INTRP(CBL,N,TL,P)
C       LL=12
C       IF (N) 161,161,210
C   COMPUTE MEAN SPECIFIC HEATS OF LIQUID, CLN, AND OF
C   GAS, CGN.
C   210 CLN=ALPHA*CAL+(1.0-ALPHA)*CBL
C       CGN=(1.0-BETA)*CGA+(BETA)*CGG
C       PHIA=1/(1.0+((VIAG/VIBG)**.5)*((WBG/WAG)**.25)**2)/
C       PHIB=1/(1.0+((VIBG/VIAG)**.5)*((WAG/WBG)**.25)**2)/
C       PHISA=1/(1.0+((VIBG/VIAG)**.5)*((WAG/WBG)**.25)**2)/
C       PHISB=1/(1.0+((VIAG/VIBG)**.5)*((WBG/WAG)**.25)**2)/
C   COMPUTE MEAN VISCOSITY OF A AND B GAS.
C       VIGH=(VIAG/(1.0+(BETA/(1.0-BETA))*((WAG/WBG)*PHIA)))+
C       VIBG/(1.0+(((1.0-BETA)/BETA)*((WBG/WAG)*PHIB)))
C   INTERPOLATE IN VISCOSITY (LIQUID) TABLES FOR A AND B AT
C   THIS TEMPERATURE.

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DSMAX = 1.0 * 2 - 1.0E-06
GO TO 70
C
SPECIFIED X(P) IS TO BE USED.
52 P10=P*DP/2.0
P11=P*DP/2.0
PINT=P10
KL=1
GO TO 1100
1152 X10=XINT
PINT=P11
KL=2
GO TO 1100
1153 X11=XINT
C
GET DIFFERENCE OF INTERPOLATED X VALUES FROM
X(P) TABLE
DX=X10-X11
S1=(0.75*RHOG*(1.0+R)*DX)/(D*RHOL)
S2=(4*H33.04*(1.0+R)*DP)/RHOL
S3=(1.0+R)/2.0
S4=1.05/(1.0+R)
GO TO 10070
C
INTERPOLATE INTO X(P) TABLE FOR TWO PRESSURE VALUES
AT EITHER SIDE OF CURRENT INTERVAL.
1100 CONTINUE
THIS CHANGE TO ALLOW FOR MONOTONIC INCREASING OR DECREASING TABLE
TERZ=PINT-XP(1,1)*1.0E-9
IF (TERZ*(XP(1,1)-XP(1,2))) 1106,1102,1105
1106 CONTINUE
DO 1103 KR=2,KXP1
TERZ=XP(1,KR)-PINT
IF (TERZ*TERZ) 1101,1104,1104
1101 CONTINUE
1103 CONTINUE
GO TO 1120
1102 CONTINUE
KR=2
R=1+R-1
4INT=XP(2,KR1)+TERZ/(XP(1,KR)-XP(1,KR1))*(XP(2,KR)-XP(2,KR1))
GO TO (1152,1153),KL
1105 CONTINUE
IF (TERZ<-0.001) 1108,1120,1120
IF (TERZ>0.001) 1120,1120,1113
1110 TERZ=-1.00*TERZ
GO TO 1106
1120 *RINT (6,9105) P10,P11,KL,TERZ,TERZ, PINT
LL=14
GO TO 500
30 NSTRT=10
RETURN
1142 NSTRT=1142
RETURN
145 NSTRT=145
RETURN
161 NSTRT=161
RETURN
70 NSTRT=70
RETURN
500 NDIAG=1
CALL OUTPUT(NNN,LPGCT,JGEO)
NSTRT=110
RETURN
C
FORMAT STATEMENTS FOR 3-LINE PERMANENT OUTPUT.
1001 FORMAT (1H0,13X,1HX,13X,1HP,13X,1HR,12X,2HVB,13X,1HA,12X,2HTG,12X,
12HTL,12X,2HVG,12X,2HVL,/,61,1P4E14.3)

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231 GO TO 500
    VLSQ=VLSQ+DVLSQ
    IF (VLSQ) 232,232,233
232 WRITE (6,9232)
    LL=15
    GO TO 500
C
C      COMPUTE NEW VELOCITIES FOR LIQUID AND GAS.
C
233 VG=ABS (SCRT (VGSQ))
    VL=ABS (SCRT (VLSQ))
C
C      COMPUTE RATIO OF GAS FLOW AREA TO LIQUID FLOW AREA.
C
    RA=RV*(VL/VG)
C
C      COMPUTE NEW LIQUID AND GAS FLOW RATES.
C
    RHAG=RHAG+DEHAG
    RHOG=RHOG+DERHOG
    DELV2=VLSQ-VLSQ
C
C      SET UP INITIAL QUANTITIES FOR NEXT END-POINT.
C
    DTLL=DTL
    DTGL=DTG
    HLAG=HLA
    HLB1=HLB
    DELT=TL-TL
    CAGI=CAG
    CGLI=CGL
    VS=S*VR
C
C      INTERPOLATE INTO SURFACE TENSION TABLE AT THIS LIQUID
C      TEMPERATURE.
C
    N=18
    NN=14
    CALL INTERP(SIG,N,TL,2)
    LL=15
    IF (N) 16,16,240
C
C      IF DROPLET BREAKUP OPTION SPECIFIED, COMPUTE DROPLET
C      DIAMETER.
240 IF (NEU-1) 241,238,243
238 DZ=(.3174*SI)/((RHOG*VS**2))
    IF (D-2) 243,243,249
239 D=DZ
C
C      COMPUTE CHANGE IN NOZZLE CROSS-SECTIONAL AREA SINCE
C      LAST END-POINT.
241 DA=144.0*ENG*(1.0/(RHOG*VG)+R/(PHOL*VL))-A
C
C      COMPUTE MEAN AREA BETWEEN THIS AND PREVIOUS ENDPOINT.
C
    AM=A+DA/2.0
C
C      COMPUTE PRESENT AREA
C
    A=A+DA
    IF (NEO-1) 67,68,67
C
C      COMPUTE RADII AND DISTANCE FROM AXIS TO WALL OF
C      CIRCULAR NOZZLE.
C      ***** STATEMENT NO. 67 *****
67 DYO=SQRT (A/.1416)-YO
    RCH=YO+DYO/2.0
    DRO=DYO
    YO=YO+DYO
    RO=YO

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      WIN=7.0
      DELT=0.0
      DELSI=3.0
      DELIN=0.0
      CFIM=0.0
      TWIN=0.0
      GO TO 64

UUU  COMPUTE GEOMETRY OF ANNULAR NOZZLE AND BOUNDARY-LAYER
      GROWTH AND JUMP FOR INNER WALL.
UUU
68  RAX=PAXC-X* SIN (PHI)
   IF (PAX**2-(A*COS (PHI)))/(2.0*3.1416) 241,242,242
241  WRITE (6,2241)
      LL=17
      GO TO 500
242  DRI=SQRT (RAX**2-(A*COS (PHI)))/(2.0*3.1416)-RI
      DRC=SQRT (RAX**2-(A*COS (PHI)))/(2.0*3.1416)-RO
      RT=DRI/2.0
      RI=RI-DRI
      ROR=RO-DRO/2.0
      RO=RO-DRO
      DTI=(RAX-RI)/(COS (PHI))-YI
      DYO=(RO-RAX)/(COS (PHI))-YO
      YI=YI-DYI
      YO=YO-DYO

UUU  COMPUTE ANGLE OF NOZZLE INNER WALL RELATIVE TO AXIS.
UUU
      WIN=ATAN (DYI/DXI)
      WIN=57.29578*WIN
246  DTHI=7.5*CFIM*((A*BN)/(1.0+BN))*DX-THIM*
      ((1.141*DVBSQ)/VBSQ-DA/AN*DRI/PIH)
      THIM=(CFIM*HOL*VLSQ)/(6559.6*(1.0+BN))
      THI=THI-DTHI
      DELI=1.246*THI
      DELSI=1.2467*THI
69  DTHO=7.5*CFIM*((A*BN)/(1.0+BN))*DX-THOM*
      ((1.141*DVBSQ)/VBSQ-DA/AN*DRO/ROH)
      THOM=(CFIM*HOL*VLSQ)/(6559.6*(1.0+BN))
      THO=THO-DTHO
      DELO=1.246*THO
      DELSO=1.2467*THO

UUU  COMPUTE ANGLE OF NOZZLE OUTER WALL RELATIVE TO AXIS.
UUU
      WOH=ATAN (DYO/DX)
      WOH=57.29578*WOH

UUU  COMPUTE MEAN VELOCITY (INCLUDING BOUNDARY LAYER).
UUU
251  VBD=VB*(1.0-((2.0*3.1416)/A)*
      ((DTHO+RI+THI))
      GO TO 41

UUUU  IF AREA LESS ZERO, NOZZLE IS CONVERGING AND THROAT HAS
      NOT JUST BEEN PASSED. IF NOT, NOZZLE THROAT MAY
      HAVE JUST BEEN PASSED.
UUUU
61  IF (A-AT) 66,62,62

UUUU  ADVANCE STEP COUNT AND RETURN TO STEP 30 IF MAXIMUM
      STEP COUNT NOT REACHED.
UUUU
62  NNS=NNS+1
   IF (NNS-NS) 30,260,260

UUUU  INITIALIZE TB AND COUNTERS AND GO TO STEP 140 TO
      COMPUTE BURNED QUANTITIES.
UUUU
260  TB=TL
      NBS=0

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      NND=1
      GO TO 180
C      OUTPUT HAS JUST BEEN PRINTED. RETURN TO STEP 30
C      IF ITERATION IS NOT COMPLETED.
C      63 NND=NND+1
      NSTMT=1999
      IF (NND-NE) 30,64,64
C
C      64 PL1(1)=SS
      PL1(2)=SS
      PL1(3)=SS
      PL1(4)=YES
      PL1(5)=ASS
      PL1(6)=TGS
      PL1(7)=TUS
      PL1(8)=VGS
      PL1(9)=VLS
      PL1(10)=1.0
C      WRITE (18) PL1
      NND=NND+1
C      264 CALL OUTPUT (NND,LPGCT,BGEO)
      GO TO 110
C      66 AT=A
      DT=D
      ES=E
      PS=P
      TUS=TU
      VGS=V
      RS=R
      VES=VB
      AS=A
      TGS=TG
      VIC=VI
      VSS=VS
      SS=S
      DD=D
      PVS=V
      RAD=RA
      ALPH=ALPHA
      BETA=BETA
      PRG=PG
      ENL=ENL
      RHOL=RHOL
      WAG=WA
      WGS=WG
      CAS=CA
      PRG=PG
      HLAG=HLA
      HLAG=HLA
      CLC=CLC
      CLC=CLC
      CLC=CLC
      VIG=VIG
      VILM=VILM
      RHGS=RHGS
      RHGS=RHGS
      CD=CD
      HNS=HN
      YOS=YO
      WONS=WONS
      THUS=THO
      DZOS=DZLO
      DZOS=DZLO
      RDOS=RDZLO
      CFCH=CFCH
      THOS=THCH
      VDS=VDS

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      1999 2690
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      1999 1200
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      1999 1180
      1999 1170
      1999 1160
      1999 1150
      1999 1140
      1999 1130
      1999 1120
      1999 1110
      1999 1100
      1999 1090
      1999 1080
      1999 1070
      1999 1060
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      1999 1040
      1999 1030
      1999 1020
      1999 1010
      1999 1000
      1999 990
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      1999 810
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      1999 770
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      1999 740
      1999 730
      1999 720
      1999 710
      1999 700
      1999 690
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19690	19690
19700	19700
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19960	19960
19970	19970
19980	19980
19990	19990
20000	20000

[illegible]

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      C2 = C2*.33333
      SO = SO*C2
      SSM = SC-SM
      81 IF (ABS (SC-SM) - ESO) 1481, 1481, 1482
      1481 NIS=NMSO
      NMSO=0
      REMIND 16
      GO TO 42
C*****THIS IS TO BYPASS ITERATION COUNTER DURING PLOTTING IJH = 4
      1482 CONTINUE
      1482 NMSO = NMSO + 1
      16060 IF (NMSO-NSO) 1483, 1483, 1490
C
      STORE THIS VALUE OF DS1 AND WRITE DS1 ARRAY IF FULL.
      1483 CONTINUE
C*****DETERMINING AID FOR BINARY CUT CONVERGENCE
      DSPAT(KDS) = 22222222.
      DSPAT(KDS+1) = S*J1
      DSPAT(KDS+2) = SCJ1
      DSPAT(KDS+3) = SCJ2
      DSPAT(KDS+4) = SM
      DSPAT(KDS+5) = SO
      DSPAT(KDS+6) = SSM
      DSPAT(KDS+7) = DDLIN
      DSPAT(KDS+8) = SCJ2
      DSPAT(KDS+9) = RDLIN
      KDS = KDS + 10
      IF (KDS - 100) 16003, 16003, 1484
      1484 WRITE (16) DSPAT
      KDSCT = KDSCT + 1
      DO 1486 I=1, 100
      DSPAT(I) = 0.0
      1486 CONTINUE
      KDS=1
      16003 CONTINUE
      GO TO (3030, 1031, 3071) IJH
C*****END OF BINARY CUT CONVERGENCE ROUTINE
C
      NO CONVERGENCE ON DS - PRINT DIAGNOSTIC AND GET ALL
      PREVIOUS VALUES OF DS FROM TABS 15 AND TEMPORARY STORAGE IN
      DSRA1. PRINT THEM AND EXIT TO DIAGNOSTIC ROUTINE.
      1490 REMIND 16
      NST=72299
      WRITE (16, 3470)
      1491 IF (KDS-1) 1493, 1493, 1492
      1492 READ (16) DSRA1
      WRITE (6, 3491) (DSRA1(I), I=1, 100)
      KDSCT = KDSCT - 1
      GO TO 1491
      1493 WRITE (6, 3491) (DSRA1(I), I=1, KDS)
      KDS=1
      KDSCT=0
      DO 1495 I=1, 100
      DSPAT(I) = 0.0
      1495 CONTINUE
      LL=18
      REMIND 16
      GO TO 500
C
      OPTIMIZE DS AND DX.
      82 IF (NFO-1) 2382, 2381, 2382
      2381 DS=0.0
      S=SO
      NFO=0
      GO TO 93
      2382 DS=SO-SOF
      83 SOP=SO
      DX = (D / (0.75 * RHOG * ABS (SM) * SM * CD * V * NSQ))
      * (4633.04 * DP * ((BH * 2) * RHOL * DVB SQ) / 2.3)

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SEC5 1486
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SEC5 2600

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2      * ((RN*BHOL*VBMSQ)/(1.0+R))*((2.0*SM*DT)/(1.0+R))-DS))
100  DS1=75
    NST=9999
    VB=VBSC+VBSSQ
    IF (VBSC) 100, 300, 301
300  WRITE (6,9100)
    LL=19
    GO TO 500
C
C      STORE **DPINT QUALITIES FOR NEXT END-POINT.
C
301  ENL=ENL
    RHCL=RHCL
    DELT=TG-TL
    VLN=VM*VBS
    HLEN=HLA
    HLEN=HLCB
    RH=R
C
C      DETERMINE HEAT TRANSFER COEFFICIENT.  SEGMENTS ARE NOT
C      CONTINUOUS AT JUNCTIONS.
C
    IF (REX-1.0) 304, 304, 105
304  HM=(24.4*HKGM)/D
    GO TO 304
305  IF (REX-25.0) 306, 306, 107
306  HM=(360.0)*CM*PHO*((ABS(SN))*VBS*
    * ((2.2/RE)*((.48/(SQRT(REM))))))
    GO TO 105
307  HM=(0.44*HKGM*(REX*.6))/D
C
C      COMPUTE FLAT BOUNDARY-LAYER PARAMETERS FOR CIRCULAR NOZZLE
C      OR OUTER WALL OF ANNULAR NOZZLE.
C
308  DELIN=1.10*CTH/2.0
    DELIN=10.246*THM
    RDELIN=(100.0*PHO*VL*DELIN)/VLN
    IF (RDELIN<.0.1) RDELIN=ABS(RDELIN)
    CF1=.238/((ALOG10(RDELIN)*.425)**2.594)
    IF (CF1) 309, 30, 309
309  THM=1.10*CTH/2.0
    DELIN=10.246*THM
    RDELIN=(100.0*PHO*VL*DELIN)/VLN
    CF1=.238/((ALOG10(RDELIN)*.425)**2.594)
    NS1=10
    RETURN
2399  NS1=2399
500  NDIAG=1
    CALL OUTUT(NNN,LPRCT,NSCO)
    NS1=110
    RETURN
9250  FORMAT(1H0.7X,15XNONPOSITIVE TBM/(1H ,3X,1P6R15.6))
9300  FORMAT(1H0.7X,14XNONPOSITIVE VB)
9490  FORMAT(1H0.7X,24XNO CONVERGE ON DS,/)
9491  FORMAT(1H ,1P10E13.4)
3061  FORMAT(1H0.3XHLK* SYN RESIDUES IN INTERVAL 3 TO ,1P12.4,
    14XFOR DP = ,E15.4)
    END
      SUBROUTINE SECT6
COMMON TZZZAY
DIMENSION TZZZAY(11200)
THE ABOVE COMMON BLOCK FOR SUBROUTINES TABLE AND INTRF
IT MUST BE THE FIRST COMMON BLOCK
C
COMMON A , ABAR , ALAM , ALPHAB , ALPHA , ALPHAB ,
1  ALPHS , AM , AP , AS , AT , SA ,
2  BB , BETAB , BETAMB , BETA , BETAS , BM ,
3  CT , CAG1 , CAGMB , CAG , CAL13 ,
4  CBG1 , CAL , CASE , CGBMB , CGB , CGBND , COL
COMMON CDS , CDP , CD , CPIN , CCFINS , CCFIN ,
5  CDFS , CGBB , CGN , CGMS , CLMB , CLN ,
6  CLBS , CSAVE , D , D2 , DA , DATE ,
7

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```

318 IF (N) 318,318,320
319 LL=20
WRITE (6,931H) HT (NN),TB,P
GO TO 500
C
C
C COMPONENT A. PRINT DIAGNOSTIC IF SUFFICIENTLY CLOSE TO ZERO.
C
320 IF (ABS (1.0-H*PBOR)-.0001) 321,321,323
321 WRITE (6,932H)
LL=20
GO TO 500
C
C
C COMPUTE PARTIAL PRESSURE OF COMPONENT A AND B AND
C PRINT DIAGNOSTIC IF EITHER NONPOSITIVE.
C
323 PAB=(P-PEOB)/(1.-(H*PBOD))
IF (PAB) 317,322,322
322 PBB = P-PAB
IF (PBB) 312,9323,9323
C
C
C INTERPOLATE INTO MOLECULAR WEIGHT TABLES FOR A AND B
C AT THIS TEMPERATURE AND PRESSURE.
C
9323 N=3
NN=3
CALL INTERP(WAGB,N,TB,PAB)
IF (N) 325,325,327
325 LL=20
WRITE (6,931H) HT (NN),TB,PAB
GO TO 500
327 N=4
NN=4
CALL INTERP(WUGB,N,TB,PBB)
IF (N) 329,329,330
329 LL=20
WRITE (6,931H) HT (NN),TB,PBB
GO TO 500
330 LL=20
WRITE (6,931H) HT (NN),TB,PBB
GO TO 500
331 LL=20
WRITE (6,931H) HT (NN),TB,PBB
GO TO 500
332 LL=20
WRITE (6,931H) HT (NN),TB,PBB
GO TO 500
C
C
C COMPUTE MEAN MOLECULAR WEIGHT.
C
330 WGB=(WAGB*PAB)+(WUGB*PBB)/P
C
C
C COMPUTE ALPHA AND BETA.
C
ALPHAB=(WAL/WBL)*H*PAB/
(1.0+((WAL/WBL)-1.0)*H*PAB)
BETAB=(WGB*PBB)/(WGB*P)
C
C
C CHECK DENOMINATOR OF EQUATION FOR R AND PRINT
C DIAGNOSTIC IF SUFFICIENTLY CLOSE TO ZERO.
C
TR=(1.0-(1.0*RC)*ALPHAB)
INT=ABS (TR)
IF (INT-.0001) 333,335,335
333 WRITE (6,933H)
LL=20
GO TO 500
C
C
C COMPUTE BARBED FLOW PARAMETERS.
C
335 ZB=(PC-(1.0*RC)*BETAB)/TR
DENGB=(ZB*(1.0*PB)-ENG)
ZHB=RO*ENG
ZHBGB=((ZHB*WAGB*PAB)/(WGB*P))-ZBAG
DENBGB=((ZHB*WGB*PBB)/(WGB*P))-ZHBG

```

```

      ENHAGP=ENHAG+DENHAGB
      ENHGB=ENHGB+DENHGB
      DTGB=(3-TG)/2.0
      TLGB=(TL+TG)/2.0
      TGGB=(TG+TB)/2.0
      PARG=(PA+PAB)/2.0
      PSGB=(PB+PBB)/2.0

      INTERPOLATE INTO SPECIFIC HEAT TABLES FOR
      CALGB - SPECIFIC HEAT - LIQUID A
      CBLGB - SPECIFIC HEAT - LIQUID B
      CAGGB - SPECIFIC HEAT - GAS A
      CBGB - SPECIFIC HEAT - GAS B

      N=5
      NN=5
      CALL INTER(CALGB,N,TLGB,P)
      IF(N) 340,340,341
340 WRITE (6,1136) HT(NN),TLGB,P
      LL=25
      GO TO 500
341 N=6
      NN=6
      CALL INTER(CBLGB,N,TLGB,P)
      IF(N) 340,340,343
343 N=1
      NN=1
      CALL INTER(CAGGB,N,TGGB,PAB)
      IF(N) 344,344,345
344 LL=25
      WRITE (6,1136) HT(NN),TGGB,PAB
      GO TO 500
345 N=2
      NN=2
      CALL INTER(CBGB,N,TGGB,PBB)
      IF(N) 346,346,347
346 LL=25
      WRITE (6,1136) HT(NN),TGGB,PBB
      GO TO 500
347 ALPHA=(ALPHA+ALPHA0)/2.0
      BETA=(BETA+PETA0)/2.0
      CGB=(ALPHA+CALGB)*(1.0-ALPHA)*CBLGB
      CGGB=(1.0-BETA)*CAGGB+BETA*CBGB
      DTGB=(-1.0/(CGB+CGGB))*((CGB*DTGB)+
1      ((DEHAG0/ENHGB)*(HLA+CAG*(TG-TL)))+
2      ((DEHAG0/ENHGB)*(HLB+CGB*(TG-TL)))+
3      ((DEHAG0/ENHGB)*(((VG**2)-(VL**2))/50072.9)))
4      DTGB=(-1.0/(CGB+CGGB))*((CGB*DTGB)+
      ((DEHAG0/ENHGB)*(HLA+CAG*(TG-TL)))+
      ((DEHAG0/ENHGB)*(HLB+CGB*(TG-TL)))+
      ((DEHAG0/ENHGB)*(((VG**2)-(VL**2))/50072.9)))

      TEST FOR CONVERGENCE ON TB.
      IF (ABS((TB-TL)-DTGB))-ES) 141,141,350
350 NNE=NN+1
      IF (NNE-NB) 351,351,354
      SAVE THIS VALUE OF TB FOR POSSIBLE NON-CONVERGENCE.
351 TB=TL+DTGB
      TBAI(KTB)=TB
      KTB=KTB+1
      IF (KTB-100) 140,140,352
352 WRITE (14) TBAI
      DO 353 I=1,100
      TBAI(I)=0.0
353 CONTINUE
      KTB=1
      KTBCT=KTBCT+1
      GO TO 140

      NO CONVERGENCE ON TB AND ITERATION MAXIMUM REACHED.
      PRINT DIAGNOSTIC AND ALL VALUES OF TB FOUND.

```

```

SEC6 1590
SEC6 1600
SEC6 1610
SEC6 1620
SEC6 1630
SEC6 1640
SEC6 1650
SEC6 1660
SEC6 1670
SEC6 1680
SEC6 1690
SEC6 1700
SEC6 1710
SEC6 1720
SEC6 1730
SEC6 1740
SEC6 1750
SEC6 1760
SEC6 1770
SEC6 1780
SEC6 1790
SEC6 1800
SEC6 1810
SEC6 1820
SEC6 1830
SEC6 1840
SEC6 1850
SEC6 1860
SEC6 1870
SEC6 1880
SEC6 1890
SEC6 1900
SEC6 1910
SEC6 1920
SEC6 1930
SEC6 1940
SEC6 1950
SEC6 1960
SEC6 1970
SEC6 1980
SEC6 1990
SEC6 2000
SEC6 2010
SEC6 2020
SEC6 2030
SEC6 2040
SEC6 2050
SEC6 2060
SEC6 2070
SEC6 2080
SEC6 2090
SEC6 2100
SEC6 2110
SEC6 2120
SEC6 2130
SEC6 2140
SEC6 2150
SEC6 2160
SEC6 2170
SEC6 2180
SEC6 2190
SEC6 2200
SEC6 2210
SEC6 2220
SEC6 2230
SEC6 2240
SEC6 2250
SEC6 2260
SEC6 2270
SEC6 2280
SEC6 2290
SEC6 2300

```

```

354 WRITE (6,9352)
359 IF (KTBC) 403,403,400
400 READ (14) TBAY1
WRITE (6,9491) (TBAY1(I), I=1,100)
KTBC=KTBC-1
GO TO 359
403 WRITE (6,9491) (TBAY1(I), I=1,KTBC)
C
C CLEAR TB SAVE ARRAY.
C
KTBC=1
KTBC=0
DO 405 I=1,100
TBAY1(I) = 0.0
405 CONTINUE
LL=25
GO TO 500
C STORE NUMBER OF ITERATIONS REQUIRED TO CONVERGE ON TB.
C
C ***** STATEMENT NO. 140 *****
C
141 NIP=NNB
NNB=1
C
C COMPUTE DENSITY OF GAS MIXTURE.
C
RHOGB=(WGB*P)/(10.732*TD)
C
C INTERPOLATE FOR DENSITIES OF LIQUIDS A AND B.
C
N=10
NN=10
CALL INTRP(ROALB,N,TB,P)
LL=27
IF (N) 319,319,360
360 N=11
NN=11
CALL INTRP(ROBLB,N,TB,P)
LL=27
IF (N) 319,319,363
C
C COMPUTE DENSITY OF LIQUID MIXTURE.
C
363 RHOLB=1.0/((ALPHA*ROALD)+(1.0-ALPHA)/(ROBLD))
ALAB=((144.0*ENCL)*((1.0/RHOCB+3B/RHOLB))/VR
RVE=1.0/((RHO*RHOCB)
Q=(RHOLB*ALAB+RVE)/(144.0*ENLB)
C
C INTERPOLATE INTO VISCOSITY TABLES FOR LIQUID A AND B.
C
N=14
NN=14
CALL INTBP(VIALD,N,TB,P)
LL=24
IF (N) 319,319,366
366 N=15
NN=15
CALL INTBP(VIBLB,N,TB,P)
IF (N) 369,369,370
369 LL=24
GO TO 319
370 VILB=ALPHA*VIALD+(1.0-ALPHA)*VIBLB
REF=(8617.2*ENLB)/(VILB*(30TT(1847)))
C ENTER CURRENT VALUES OF A AND P INTO A VS P TABLE
C
529 NSTMT=529
RETURN
C
C *****
C
C FIND GOOD APPROXIMATION FOR DS1 TO USE IN SUCCESSIVE

```


1950	100
1951	100
1952	100
1953	100
1954	100
1955	100
1956	100
1957	100
1958	100
1959	100
1960	100
1961	100
1962	100
1963	100
1964	100
1965	100
1966	100
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2029	100
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2031	100
2032	100
2033	100
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2036	100
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2086	100
2087	100
2088	100
2089	100
2090	100
2091	100
2092	100
2093	100
2094	100
2095	100
2096	100
2097	100
2098	100
2099	100
2100	100

1950	1951	1952	1953	1954	1955	1956	1957	1958	1959	1960	1961	1962	1963	1964	1965	1966	1967	1968	1969	1970	1971	1972	1973	1974	1975	1976	1977	1978	1979	1980	1981	1982	1983	1984	1985	1986	1987	1988	1989	1990	1991	1992	1993	1994	1995	1996	1997	1998	1999	2000	2001	2002	2003	2004	2005	2006	2007	2008	2009	2010	2011	2012	2013	2014	2015	2016	2017	2018	2019	2020	2021	2022	2023	2024	2025	2026	2027	2028	2029	2030	2031	2032	2033	2034	2035	2036	2037	2038	2039	2040	2041	2042	2043	2044	2045	2046	2047	2048	2049	2050	2051	2052	2053	2054	2055	2056	2057	2058	2059	2060	2061	2062	2063	2064	2065	2066	2067	2068	2069	2070	2071	2072	2073	2074	2075	2076	2077	2078	2079	2080	2081	2082	2083	2084	2085	2086	2087	2088	2089	2090	2091	2092	2093	2094	2095	2096	2097	2098	2099	2100	2101	2102	2103	2104	2105	2106	2107	2108	2109	2110	2111	2112	2113	2114	2115	2116	2117	2118	2119	2120	2121	2122	2123	2124	2125	2126	2127	2128	2129	2130	2131	2132	2133	2134	2135	2136	2137	2138	2139	2140	2141	2142	2143	2144	2145	2146	2147	2148	2149	2150	2151	2152	2153	2154	2155	2156	2157	2158	2159	2160	2161	2162	2163	2164	2165	2166	2167	2168	2169	2170	2171	2172	2173	2174	2175	2176	2177	2178	2179	2180	2181	2182	2183	2184	2185	2186	2187	2188	2189	2190	2191	2192	2193	2194	2195	2196	2197	2198	2199	2200	2201	2202	2203	2204	2205	2206	2207	2208	2209	2210	2211	2212	2213	2214	2215	2216	2217	2218	2219	2220	2221	2222	2223	2224	2225	2226	2227	2228	2229	2230	2231	2232	2233	2234	2235	2236	2237	2238	2239	2240	2241	2242	2243	2244	2245	2246	2247	2248	2249	2250	2251	2252	2253	2254	2255	2256	2257	2258	2259	2260	2261	2262	2263	2264	2265	2266	2267	2268	2269	2270	2271	2272	2273	2274	2275	2276	2277	2278	2279	2280	2281	2282	2283	2284	2285	2286	2287	2288	2289	2290	2291	2292	2293	2294	2295	2296	2297	2298	2299	2300	2301	2302	2303	2304	2305	2306	2307	2308	2309	2310	2311	2312	2313	2314	2315	2316	2317	2318	2319	2320	2321	2322	2323	2324	2325	2326	2327	2328	2329	2330	2331	2332	2333	2334	2335	2336	2337	2338	2339	2340	2341	2342	2343	2344	2345	2346	2347	2348	2349	2350	2351	2352	2353	2354	2355	2356	2357	2358</
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      SGN=SGN/10.0
      ISL=ISL+1
      IF (ISL-5) 2403,2403,5060
12017 CONTINUE
      DSAVE=DSRES
      DSX2=DS1
      GO TO 2409

      EVALUATE DS FOR GIVEN DS1.  TEST FOR CONVERGENCE
      #####

5000 CONTINUE
      SM=S*DS1/2.0
      AM=1.0+(R*SM)/(1.0+R)
      BM=1.0-SM/(1.0+R)
      DVBSQ=(-V1/V1)*(-V2/V2)
      VBNSQ=VBSSQ+DVBSQ/2.0
      IF (V1=0) GO TO 5001,5001,5001

6000 CONTINUE
      WRITE (6,7001) SM,S,DS1,AM,B,DM,DVBSQ,V1,V2,VBNSQ,VBN
5001 CONTINUE
      VBN=SQRT(VBNSQ)
      REM=1+VBN*ARS(S-1)
      IF (REM=-0.000001) 5015,5015,5005
5005 IF (REM=-0.1) 5020,5020,5010
5010 IF (REM=-0.0001) 5030,5025,5025
5015 CD=2.4E-7
      GO TO 5015

5020 CD=24.3E-7
      GO TO 5015

5025 CD=0.4E-7
      GO TO 5015

5030 CD=EXP(3.271-0.8843*(ALOG(ABS(1+0.03417*
      11*(ALOG(ABS(1+0.01344*(ALOG(ABS(1+0.01344*
5035 CONTINUE
      DSXP=-(0.51408*(SM)*SM*CD)/DM)*(S2/(DM*VUNSQ))
      I=(S1+1)*DVBSQ/(VUNSQ)+SM*SH
      DSRP=57E-27
      IF (ABS(DSRP)-DS1) 91,91,5040
5040 DSAVE(DS)=DS1
      DSAVE(KDS+1)=DSRES
      KDS=KDS+1
      IF (KDS-100) 5055,5055,5045
5045 WRITE (16) DSRP
      KDSCT=KDSCT+1
      DO 5060 L=1,100
5050 UCTAY(L)=0.3
      KDS=1
5055 VUNSQ=VUNSQ+1
      IF (VUNSQ-NDS) 5070,5070,5060
5060 WRITE (6,5060) DSX1,DSX2,DS1,DSRES1,DSRES2,DSRES
      LL=18
      GO TO 1490
5070 CONTINUE
      GO TO (2401,2402,2404,12411,2405,2407,2409),ISL
      #####

      COMPUTE NEXT APPROXIMATION TO DS.

91 NIS=KNNS
      IRLX=1
      DS=DS1
      DSAVE=DS1
      KNS=0
      GO TO 100
1490 NSTXT=1490
      RETURN
100 NSTXT=100
      RETURN
500 NDIAG=1
      CALL OUTPUT(NNN,LPGCT,NGB0)

```



```

GO TO(501,502,503,504,505,506,507,508,509,510,511,512,513,514,
1 515,516,517,518,519,520,521,522,523,524,525,526,527,528,
2 529),LL
cccc
DIAGNOSTIC OUTPUT. ENTRY IS MADE TO SET CERTAIN
SET OF OUTPUT VALUES INTO THE ARRAY DRAY DEPENDNG
ON THE VALUE OF LL, WHICH IS SET IN VARIOUS POINTS THROUGHOUT
THE MAIN PROGRAM.
529 DRAY(54)=RPF
NSTA=8000
DRAY(45)=VILB
528 DRAY(53)=Q
DRAY(34)=AAR
DRAY(47)=HHOLB
527 DRAY(46)=THOGB
526 DRAY(39)=TU
525 DRAY(44)=CNLB
DRAY(43)=ENGU
DRAY(37)=RU
524 DRAY(42)=BETAB
DRAY(41)=ALPHAB
DRAY(50)=WGB
DRAY(49)=WUTB
523 DRAY(48)=WAB
522 DRAY(52)=TAB
521 DRAY(51)=PAB
520 DRAY(73)=CEI
DRAY(69)=RDELIN
DRAY(71)=TILN
DRAY(61)=CFLN
DRAY(53)=DELON
DRAY(36)=HY
DRAY(35)=CU
519 DRAY(34)=RBN
518 DRAY(63)=VBJ
DRAY(64)=FOM
DRAY(59)=DELLO
DRAY(58)=DELLO
DRAY(57)=THO
DRAY(62)=TJON
DRAY(63)=DELLO
DRAY(66)=THI
DRAY(56)=WIT
517 DRAY(51)=A
DRAY(12)=D
DRAY(24)=SIT
516 DRAY(34)=RA
DRAY(9)=VL
DRAY(4)=VG
515 DRAY(1)=X
DRAY(10)=CLN
DRAY(21)=CLN
DRAY(31)=VTLN
DRAY(32)=VTLN
DRAY(33)=HKTN
GO TO 1511
514 DRAY(13)=HKTN
DRAY(32)=VILN
513 DRAY(31)=VTLN
DRAY(20)=CLN
DRAY(30)=CLN
512 DRAY(64)=VBO
DRAY(54)=RBN
DRAY(54)=DELLO
DRAY(54)=DELLO
DRAY(57)=THO
DRAY(62)=TJON
DRAY(68)=DELLO
DRAY(57)=THI
DRAY(66)=THI

```

```

DIAG 0850
DIAG 0860
DIAG 0870
DIAG 0880
DIAG 0890
DIAG 0900
DIAG 0910
DIAG 0920
DIAG 0930
DIAG 0940
DIAG 0950
DIAG 0960
DIAG 0970
DIAG 0980
DIAG 0990
DIAG 1000
DIAG 1010
DIAG 1020
DIAG 1030
DIAG 1040
DIAG 1050
DIAG 1060
DIAG 1070
DIAG 1080
DIAG 1090
DIAG 1100
DIAG 1110
DIAG 1120
DIAG 1130
DIAG 1140
DIAG 1150
DIAG 1160
DIAG 1170
DIAG 1180
DIAG 1190
DIAG 1200
DIAG 1210
DIAG 1220
DIAG 1230
DIAG 1240
DIAG 1250
DIAG 1260
DIAG 1270
DIAG 1280
DIAG 1290
DIAG 1300
DIAG 1310
DIAG 1320
DIAG 1330
DIAG 1340
DIAG 1350
DIAG 1360
DIAG 1370
DIAG 1380
DIAG 1390
DIAG 1400
DIAG 1410
DIAG 1420
DIAG 1430
DIAG 1440
DIAG 1450
DIAG 1460
DIAG 1470
DIAG 1480
DIAG 1490
DIAG 1500
DIAG 1510
DIAG 1520
DIAG 1530
DIAG 1540
DIAG 1550
DIAG 1560

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```

DPAT(65)=WIN
DRAT(5)=A
DRAT(12)=D
DRAT(28)=SIG
DRAT(14)=PA
DRAT(9)=VL
DRAT(8)=VG
DRAT(1)=Y
1511 DRAT(55)=TO
511   DRAT(54)=TI
      DRAT(11)=S
      DRAT(4)=VB
      DRAT(10)=VS
      DRAT(5)=A
      DRAT(27)=HLB
      DRAT(26)=HLA
510   DRAT(11)=RV
509   DRAT(18)=CML
      DRAT(17)=FMT
      DRAT(2)=RNDL
508   DRAT(17)=RHOG
      DRAT(3)=R
507   DRAT(16)=BETA
      DRAT(15)=ALPHA
      DRAT(23)=WG
      DRAT(22)=WEG
506   DRAT(21)=WAG
505   DRAT(25)=ZU
504   DRAT(24)=PA
503   DRAT(7)=TL
502   DRAT(6)=TG
501   DRAT(2)=P
      NCDPAT(72)=NNA
      NCDPAT(73)=NIS
      NCDPAT(74)=NIB
      IF(NCDPAT-1)540,550,540
540   WRITE(1) DPAT
      NNN=NNN+1
      DRAT(12)=7.0
      WRITE(18)(DRAT(J),J=1,10)
      NNN=N+1
      GO TO 63
DIAGNOSTIC OUTPUT - VARIABLES NOT YET COMPUTED THIS
ITERATION ARE SET TO ZERO. PRINT OUT NINE LINES
OF OUTPUT.
550 WRITE(6,1001) (DRAT(I),I=1,9)
551 WRITE(6,1002) (DRAT(I),I=10,18)
552 WRITE(6,1003) (DRAT(I),I=19,27)
553 WRITE(6,1004) (DRAT(I),I=28,36)
554 WRITE(6,1005) (DRAT(I),I=29,35)
555 WRITE(6,1006) (DRAT(I),I=40,44)
556 WRITE(6,1007) (DRAT(I),I=55,63)
557 WRITE(6,1008) (DRAT(I),I=64,71)
558 CONTINUE
      WRITE(6,1009) NNA,NIS,NIB
10   NSINT=10
      RETURN
63   NSINT=63
      RETURN
FORMAT STATEMENTS FOR 9-LINE PERMANENT OUTPUT.
1001 FORMAT(1H1,1IX,1HX,13X,1H1,13X,1HR,12X,2HV3,1IX,1HA,12X,2HTG,12X,
12HVL,12X,2HVG,12X,2HVL,/,5X,2FO6.4)
1002 FORMAT(15I,12X,2HVS,1IX,1HS,1IX,1H5,12X,2HPV,12X,2HFA,9X,
15ALAN,10X,4HRTM,12X,2HVS,12X,2HVL,12X,2HVL,/,5X,2E14.3)
1003 FORMAT(1IX,1HX,1H60,1IX,1HAC,1IX,1HWAG,1IX,1HBSG,12X,2HWG,
112X,2HFA,12X,2HVS,12X,2HVA,12X,2HLS,/,5X,2E14.3)
1004 FORMAT(1H0,9X,5HS,7MA,1IX,1HCGR,1IX,1HCLA,10X,4HVIGM,1CX,
14HWLL,1IX,1HKGK,1IX,1HREM,1IX,1HCDA,12X,2HHN,/,5X,15E210.4)

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186

	R=1	TABL 3470
	IF (L-35) 4,4,27	TABL 3440
C	THIS CHANGE TO MAKE THE MAX DIMENSION CONSISTENT WITH THE DIMENSTION	TABL 3490
	15 IF (I-8) 16,17,17	TABL 3500
	16 L=L+1	TABL 3510
	I = I + 1	TABL 3520
	K=1	TABL 3530
	GO TO 4	TABL 3540
	27 WRITE (4,927) I	TABL 3550
	GO TO 17	TABL 3560
	17 CONTINUE	TABL 3570
	REWIND 12	TABL 3580
	WRITE (12) XVAR	TABL 3590
	WRITE (12) TSAVE1	TABL 3600
	WRITE (12) TSAVE2	TABL 3610
C	NEXT CARD IS FIRST CARD OF TVAR ARRAY	TABL 3620
	DO 36 J=1,14	TABL 3630
	K=1	TABL 3640
	29 READ (5,971) (C(L),L=1,6)	TABL 3650
	ICARD=ICARD+1	TABL 3660
	DO 34 L=1,5,2	TABL 3670
	IF (C(L)+C(L+1)-199999.) 32,30,30	TABL 3680
	30 TVAR (L,1,K)=100000.	TABL 3690
	TVAR (L+1,K)=100000.	TABL 3700
	GO TO 41	TABL 3710
	32 TVAR (L,1,K)=C(L)	TABL 3720
	TVAR (L+1,K)=C(L+1)	TABL 3730
	IF (K-1) 33,37,37	TABL 3740
	K=K+1	TABL 3750
	34 CONTINUE	TABL 3760
	DO 35 J=1,29	TABL 3770
	38 CONTINUE	TABL 3780
	16 WRITE (12) TVAR	TABL 3790
	CSAVE=1.0	TABL 3800
	READ (5,301) (C(L),L=1,6)	TABL 3810
	ICARD=ICARD+1	TABL 3820
	DO 1036 J=1,6	TABL 3830
	1036 CSAVE = CSAVE * C(J)	TABL 3840
	IF (CSAVE) 37,37,37,37	TABL 3850
	37 CALL INTERP(4,20,1.0,P)	TABL 3860
	CALL ROUTE	TABL 3870
	CALL EXIT	TABL 3880
	1037 CONTINUE	TABL 3890
	EXECUTE 12	TABL 3900
	REWIND 12	TABL 3910
	PETROL	TABL 3920
	901 FFORMAT (1H,7X,19HERROR IN INPUT DATA,126)	TABL 3930
	901 PFORMAT (6E12.6)	TABL 3940
	927 PFORMAT (1H,7X,36HPRECEDING MAXIMUM TEMPERATURE INPUT ,	TABL 3950
	17HFOR 4 = 13)	TABL 3960
	END	TABL 3970
	SUBROUTINE INTERP(VAR,N,T,P)	TABL 3980
C	IF N=3, READ IN ALL TABLES SET UP T ARRAY, AND EXIT.	TABL 3990
C	IF N=20 WRITE OUT ALL TABLES AND EXIT.	TABL 4000
C	IF N=1 THEN 1, GET APPROPRIATE TABLES FROM TAPE.	TABL 4010
C	INTERPOLATE FOR VALUE OF VAR AND EXIT.	TABL 4020
C	IF N=5 THEN 13, INTERPOLATE FOR VALUE OF VAR AND EXIT.	TABL 4030
	DIMENSION TVAR(14,2,50),TVAR1(35,4,35),TVAR2(35,4,35)	TABL 4040
	1 TREAT(4,35),H(14)	TABL 4050
	COMMON TVAR1,TVAR2,TVAR	TABL 4060
	IF N=0	TABL 4070
	IF (N) 1,1,10	TABL 4080
	1 K=	TABL 4090
	REWIND 12	TABL 4100
	READ (12) TRAY	TABL 4110
	READ (12) TVAR1	TABL 4120
	READ (12) TVAR2	TABL 4130
C	T ARRAY HAS BEEN SET UP, NO. OF T RECORDS (K) IS SET, NOW	TABL 4140
C	PROCEED TO BRING IN PERMANENT ARRAYS.	TABL 4150

C	READ (12) TVAR	INTR 1220
	RETURN	INTR 1230
10	IF (N-20) 100, 12, 11	INTR 1240
11	WRITE (6, 111)	INTR 1250
	CALL EXIT	INTR 1260
C	WRITE OUT TABLES FOR N=20, T IS PAGE COUNT.	INTR 1270
12	CALL ADATA	INTR 1280
	LC1=1	INTR 1290
	LPGCT=1	INTR 1300
	WRITE (6, 122) LPGCT	INTR 1310
	LPGCT=LPGCT+1	INTR 1320
	DO 30 L=1, 14	INTR 1330
13	NT=1	INTR 1340
14	TEST1=TVAR(L, 1, NT)*TVAR(L, 2, NT)	INTR 1350
	IF (TEST1-2.345678) 15, 15, 16	INTR 1360
15	NT=NT+1	INTR 1370
	IF (NT-50) 14, 14, 1000	INTR 1380
16	TVAR(L, 1, NT)=0.0	INTR 1390
	TVAR(L, 2, NT)=0.0	INTR 1400
	IF (NT-11515, 520, 515)	INTR 1410
C	40 INQUIRE THIS PAGE	INTR 1420
520	WRITE (6, 1520) H(L)	INTR 1430
	LC1=LC1+4	INTR 1440
	GO TO 510	INTR 1450
515	NT=NT+1	INTR 1460
	WRITE (6, 116) H(L)	INTR 1470
	L1=1	INTR 1480
	L2=AMINO(H, NT)	INTR 1490
	LT=1	INTR 1500
18	WRITE (6, 118) H(L), (TVAR(L, 2, LK), LK=L1, L2)	INTR 1510
	WRITE (6, 119) (TVAR(L, 1, LK), LK=L1, L2)	INTR 1520
	IF (L2-NT) 19, 20, 20	INTR 1530
19	L1=L1+1	INTR 1540
	L2=AMINO(L2+1, NT)	INTR 1550
	LT=LT+1	INTR 1560
	GO TO 14	INTR 1570
20	LC1=LT+1+4*LC1	INTR 1580
530	IF (LC1-2110, 21, 2)	INTR 1590
21	WRITE (6, 123) LPGCT	INTR 1600
	LPGCT=LPGCT+1	INTR 1610
	LC1=1	INTR 1620
30	CONTINUE	INTR 1630
	PERIOD 12	INTR 1640
	DATA H(1)/48 CAG/	INTR 1650
	DATA H(2)/48 CBG/	INTR 1660
	DATA H(3)/48 WAG/	INTR 1670
	DATA H(4)/48 WBG/	INTR 1680
	DO 50 L=1, 4	INTR 1690
	IF (22*AY(L, 1)-99999.) 29, 29, 50	INTR 1700
29	WRITE (6, 124) LPGCT	INTR 1710
31	WRITE (6, 115) H(L)	INTR 1720
	LPGCT=LPGCT+1	INTR 1730
	LC1=1	INTR 1740
	DO 45 KK=1, 35	INTR 1750
	IF (22*AY(L, KK)-99999.) 32, 32, 45	INTR 1760
32	WRITE (6, 116) TRAY(L, KK)	INTR 1770
	LC1=LC1+1	INTR 1780
	NCT=1	INTR 1790
34	TEST1=TVAR1(KK, L, NCT)+TVAR2(KK, L, NCT)	INTR 1800
	IF (TEST1-2.345678) 35, 36, 36	INTR 1810
35	NCT=NCT+1	INTR 1820
	IF (NCT-35) 34, 34, 1000	INTR 1830
36	NCT=NCT-1	INTR 1840
	L1=1	INTR 1850
	L2=AMINO(R, NCT)	INTR 1860
37	CONTINUE	INTR 1870
38	DO 31 LT=L1, L2	INTR 1880
	IF (TVAR2(KK, L, LT)-100000.) 63, 60, 63	INTR 1890
60	TVAR1(KK, L, LT)=0.0	INTR 1900
	TVAR2(KK, L, LT)=0.0	INTR 1910
63	CONTINUE	INTR 1920

```

WRITE (6,938) H(L), (TVAR2(KK,L,LT),LT=L1,L2)
39 WRITE (6,939) (TVAR1(KK,L,LT),LT=L1,L2)
   LT=LCT+1
   IF (LCT-52) 66,65,65
65 WRITE (6,923) LPGCT
   LPGCT=LPGCT+1
   LCT=1
66 IF (L2-NCT) 40,41,41
40 L1=L1+8
   L2=L1+8 (L2+8,NCT)
   GO TO 38
41 IF (LCT-52) 45,42,42
42 WRITE (6,921) LPGCT
   LPGCT=LPGCT+1
   LCT=1
45 CONTINUE
50 CONTINUE
   IF (LBA-1) 1047,1046,1047
1046 N=0
1047 RETURN
C N=1 THRU 19 - INTERPOLATE ROUTINE
100 IF (N-4) 101,101,150
101 DO 104 I=1,35
   IF (TVAR1(N,I)-T) 104,105,105
104 CONTINUE
   GO TO 140
105 IF (I-1) 106,120,106
106 I=I-1
   TK=TVAR1(N,I)
   TK1=TVAR1(N,I+1)
   DO 107 J=1,35
   IF (TVAR1(IM1,N,J1)-P) 107,107,108
107 CONTINUE
   GO TO 120
108 IF (I1-1) 109,120,109
109 IF (TVAR1(I1,N,J1)+TVAR2(IM1,N,J1)-2.0*99999.1) 110,120,120
110 DIFF1=TVAR1(I1,N,J1)-TVAR1(I1,N,J1)
   DIFF2=TVAR1(I1,N,J1)-0
   PC1=(DIFF1/DIFF2)*(TVAR2(IM1,N,J1)-TVAR2(IM1,N,J1))+
   TVAR2(I1,N,J1)
   DO 111 J2=1,35
   IF (TVAR1(I,N,J2)-P) 111,111,112
111 CONTINUE
   GO TO 120
112 IF (J2-1) 113,120,113
113 J2=J2-1
   IF (TVAR1(I,N,J2)+TVAR2(I,N,J2)-2.0*99999.1) 114,120,120
114 DIFF=TVAR1(I,N,J2)-TVAR1(I,N,J2)
   DIFF1=TVAR1(I,N,J2)-P
   PC2=(DIFF/DIFF1)*(TVAR2(I,N,J2)+TVAR2(I,N,J2))+TVAR2(I,N,J2)
   VAR=(TVAR1(TK1)/TK-TK1)*(PC2-PC1)+PC1
110 PE=JN4
120 N=0
   IN=1
   GO TO 12
150 N=N-4
   IF (N-15) 155,1000,1000
155 CONTINUE
   DO 151 I=1,50
   IF (TVAR(N,1,I)-T) 151,151,152
151 CONTINUE
   GO TO 120
152 IF (TVAR(N,1,I)+TVAR(N,2,I)-2.0*99999.1) 153,120,120
153 IF (I-1) 154,120,154
154 I=I-1
   VAR=(T-TVAR(N,1,IP))/(TVAR(N,1,IP)-TVAR(N,1,IP'))))
   TVAR(N,2,IP)-TVAR(N,2,IP1)+TVAR(N,2,IP1)
   GO TO 120
1000 CONTINUE
   CALL OUTPUT

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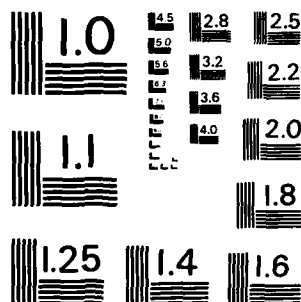
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MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS-1963-A

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CALL DUMP
RETURN
911 FORMAT (1H,7F,13HILLEGAL CALLING SEQUENCE TO INTRP)
916 FORMAT (1H,56X,11HINPUT TABLE,2X,A5,/)
918 FORMAT (1H,A5,1PH15.6)
919 FORMAT (1H,2X,1PH15.6,1PH15.6)
920 FORMAT (1H,2X,120X,44PAG,1X,11)
914 FORMAT (1H,4X,14HINTERP(CUSE = ,F12.4,/)
918 FORMAT (1H,2X,16,1PH15.6)
919 FORMAT (1H,2X,SHRESS,1X,1PH15.6)
9520 FORMAT (1H,7X,14HNO INPUT FOR TABLE, A6)
END
C.....
SUBROUTINE DUMP
CALL EXIT
RETURN
END
C.....
SUBROUTINE ADATA
DIMENSION HT(18),H(14),HT1(19),H1(14)
COMMON HT,H
DATA HT1/4H CAG,4H CDG,4H ZAG,4H ZAG,4H CAL,
1 4H COL,4H LA,4H LB,4H PBO,4HROAL,
2 4HROAL,4H KAG,4H KAG,4HVIAG,4HVIAG,
3 4HVIAG,4HVIAG,4HVIAG,4HVIAG,
DATA H1/4H CAL,4H CBL,4H HLA,4H HLB,4H PBO,
1 4HROAL,4HROBL,4H KAG,4H KAG,4HVIAG,
2 4HVIAG,4HVIAG,4HVIAG,4HVIAG,
DO 10 I=1,18
HT(I)=HT1(I)
10 CONTINUE
DO 20 I=1,14
H(I)=H1(I)
20 CONTINUE
RETURN
END

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